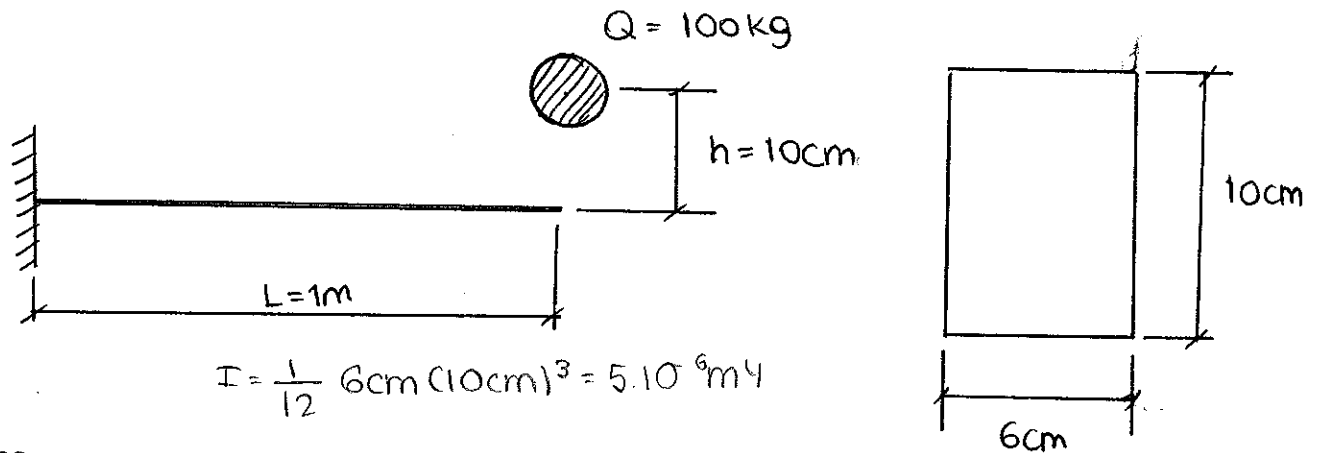
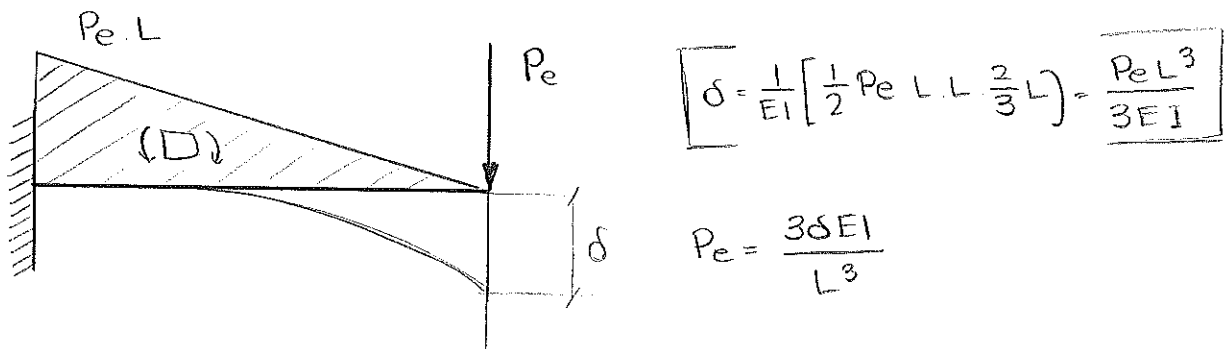


# 6. Teoría elemental del choque.

6.1



$E = 200 \text{ GPa}$ .



$U_s = W_e$ .

$U_s = Q(\delta + h)$

$W_e = \frac{1}{2} P_e \cdot \delta = \frac{3EI \delta^2}{2L^3}$

$Q(\delta + h) = \frac{3EI \delta^2}{2L^3}$

$\frac{Q \cdot L^3}{3EI} (\delta + h) = \frac{\delta^2}{2}$

$2\delta_{ST} (\delta + h) = \delta^2$

$\delta \ll h: \delta = \sqrt{2\delta_{ST} \cdot h}$

$\delta = \sqrt{\frac{2QL^3 \cdot h}{3EI}} \Rightarrow P_e = \frac{3EI}{L^3} \sqrt{\frac{2QL^3 h}{3EI}} = 24494,897 \text{ N}$

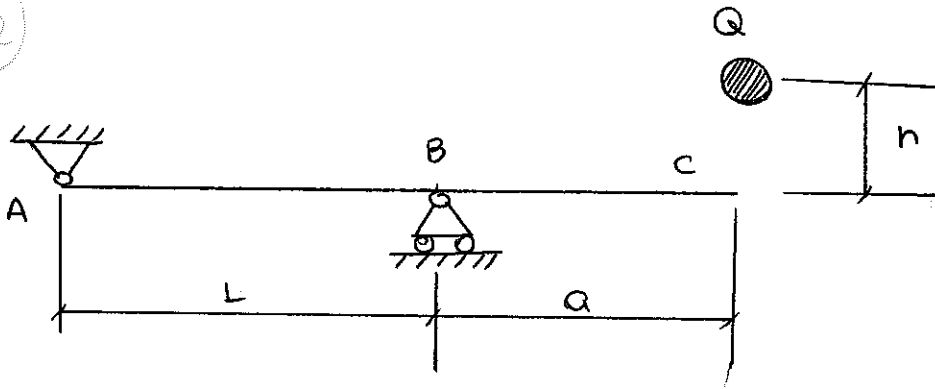
MOMENTO MÁXIMO: Empotramiento ( $M_2 = 24494,897 \text{ Nm}$ )

$\sigma_{xx} = \frac{M_2 \cdot y}{J_2} = 244'95 \text{ MPa}$

$\sigma_{\max} = 244'95 \text{ MPa}$



6.2



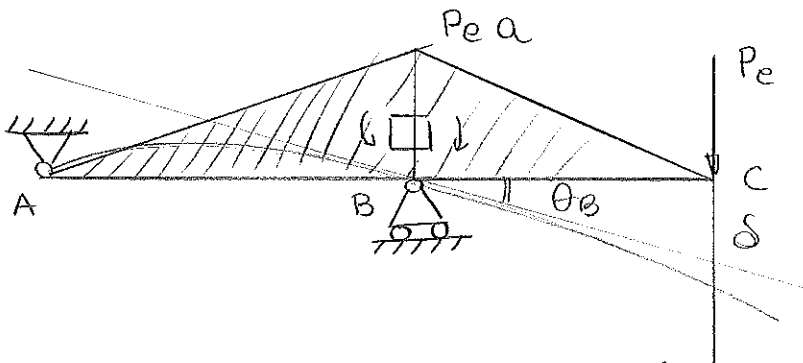
⊗  $EI = 500 \text{ kN.m}^2$

⊗  $Q = 200 \text{ Kg}$

⊗  $h = 10 \text{ cm}$

⊗  $a = 80 \text{ cm}$

⊗  $L = 1 \text{ m}$



$\delta = \theta_B \cdot a + \delta_{CB}$

$\theta_B = \delta_{AB} \cdot \frac{1}{L}$

$\delta_{AB} = \frac{1}{EI} \left[ \frac{1}{2} Pe \cdot a \cdot L \cdot \frac{2}{3} L \right] = \frac{Pe a L^2}{3EI}$

$\theta_B = \frac{Pe \cdot a L}{3EI}$

$\delta_{CB} = \frac{1}{EI} \left[ \frac{1}{2} Pe \cdot a \cdot a \cdot \frac{2}{3} a \right] = \frac{Pe a^3}{3EI}$

$\delta = \frac{Pe a^2 L}{3EI} + \frac{Pe a^3}{3EI}$

$\delta = \frac{Pe a^2}{3EI} (L + a)$

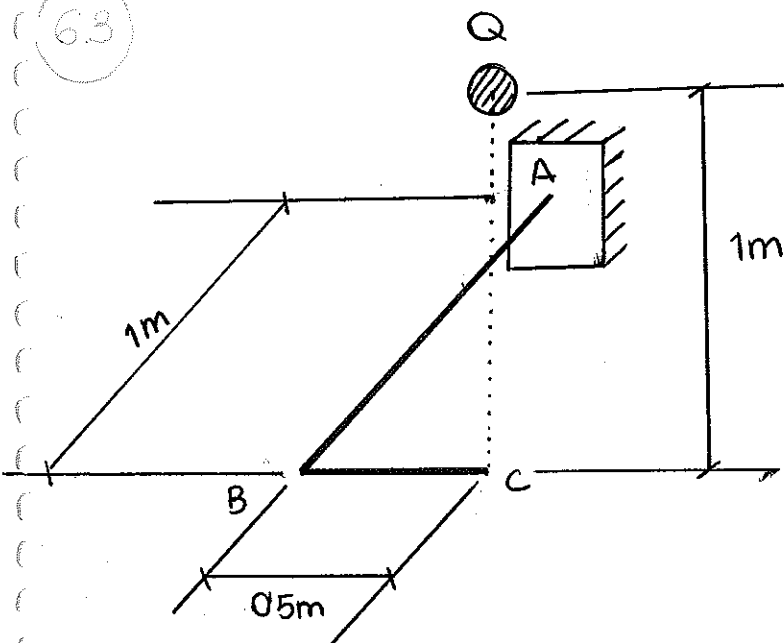
$$\left. \begin{aligned} U_s &= W_e \\ U_s &= Q(h + \delta) \\ W_e &= \frac{1}{2} Pe \cdot \delta \end{aligned} \right\}$$

$$Pe = \frac{3 \delta EI}{a^2 (L + a)}$$

$We = \frac{3EI\delta^2}{2a^2(L+a)} = Q(h+\delta) \Rightarrow \delta = \text{obtuírím}$

$M_{max} = Pe \cdot a = 19927,393 \text{ Nm}$

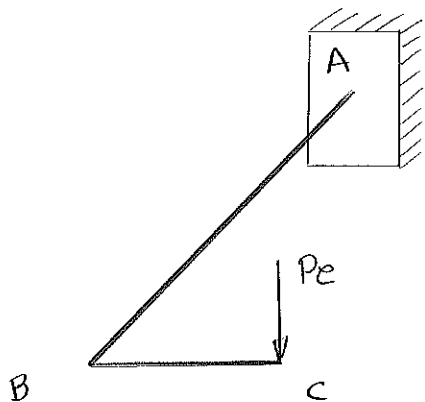




- ⊙  $d = 10\text{cm}$
- ⊙  $Q = 10\text{kg}$
- ⊙  $E = 200\text{GPa}$
- ⊙  $G = 80\text{GPa}$

$$\left. \begin{aligned} U_s &= W_e \\ U_s &= Q(\delta + h) \\ W_e &= \frac{1}{2} P_e \cdot \delta \end{aligned} \right\} Q(\delta + h) = \frac{1}{2} P_e \cdot \delta$$

$$\delta = \delta(P_e)$$



$$\delta = \delta_C = \delta(\text{flexión BC}) + \delta(\text{flexión AB}) + \delta(\text{torsión AB})$$

DIAGRAMA DE ESFUERZOS FLECTORES

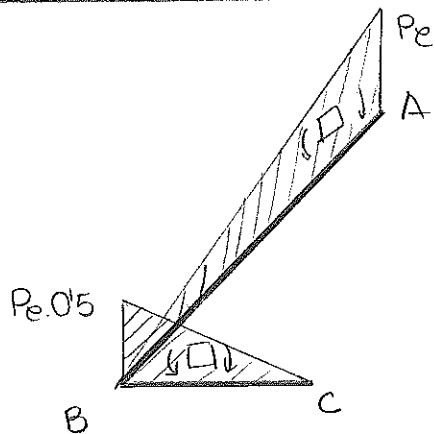
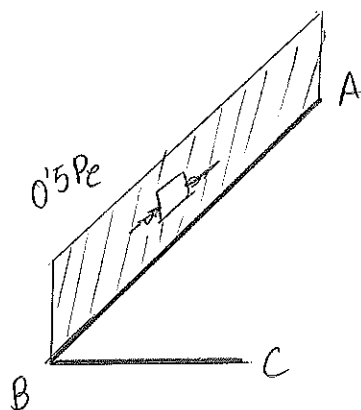


DIAGRAMA DE ESFUERZOS TORSORES



$$\delta_{FI,BC} = \frac{1}{EI} \left[ \frac{1}{2} \cdot 0.5Pe \cdot 0.5 \cdot \frac{2}{3} \cdot 0.5 \right] = \frac{Pe}{24EI}$$

$$\delta_{PI,AB} = \frac{1}{EI} \left[ \frac{1}{2} \cdot Pe \cdot 1 \cdot \frac{2}{3} \cdot 1 \right] = \frac{Pe}{3EI}$$

$$\delta_{T,AB} = \varphi_{BA} \cdot 0.5$$

$$\varphi_{BA} = \frac{M_x \cdot \Delta x_{BC}}{GI_p} = \frac{0.5Pe \cdot 1}{2GI} = \frac{Pe}{4GI}$$

$$\delta_{T,AB} = \frac{Pe}{8GI}$$

$$I = \frac{\pi R^4}{4} = 4,90873521 \cdot 10^{-6} \text{ m}^4$$

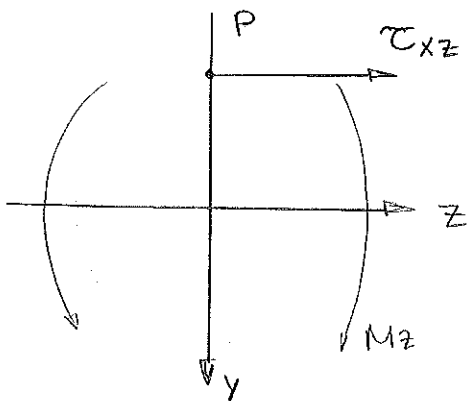
$$\delta = \frac{Pe}{24EI} + \frac{Pe}{EI \cdot 3} + \frac{Pe}{8GI} = 7,002817496 \cdot 10^{-7} Pe$$

$$Pe = Pe(\delta) = 1427996,661 \cdot \delta$$

$$Q(h+\delta) = \frac{1}{2} Pe \cdot \delta = 713998,3304 \delta^2 \Rightarrow \delta = 0.0119 \text{ m}$$

$$Pe = 16999,98 \text{ N} \approx 17 \text{ kN}$$

SECCIÓN MÁS DESFAVORABLE: A (empotramiento)

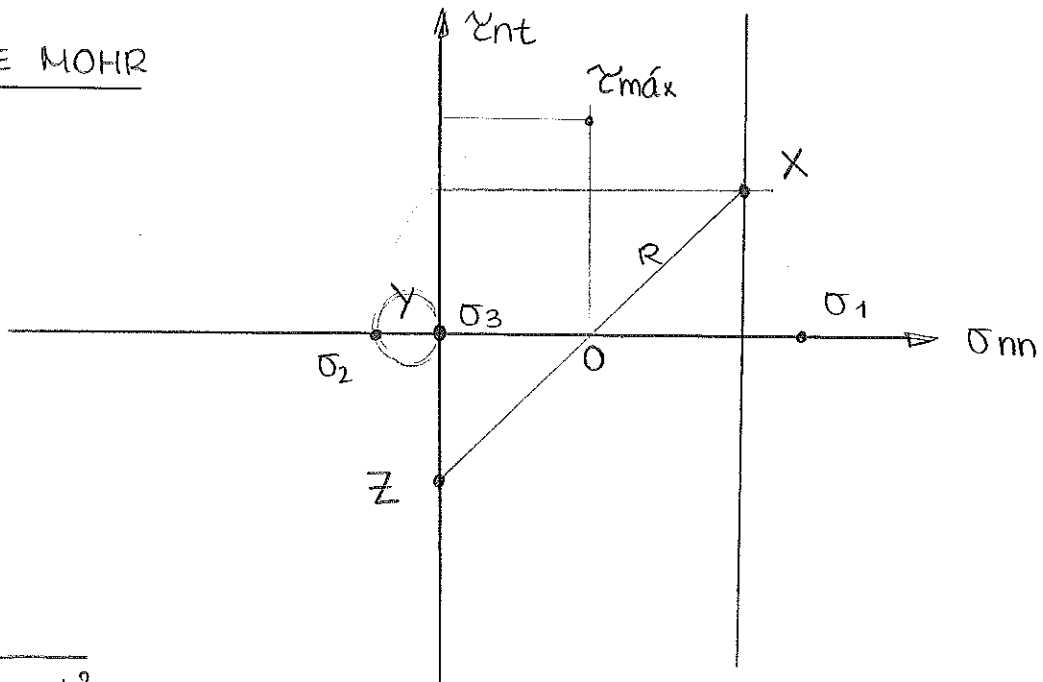


⊗ Punto más desfavorable: P.

$$\sigma_{xx} = \frac{M_z \cdot y}{I_z} = 173,1607 \text{ MPa (+)}$$

$$\tau_{xz} = \frac{M_x \cdot R}{2I_z} = 43,290 \text{ MPa}$$

## CÍRCULO DE MOHR



$$R = \sqrt{\tau_{xz}^2 + \left(\frac{\sigma_{xx}}{2}\right)^2} = 96'800 \text{ MPa}$$

$$\sigma_1 = \frac{\sigma_{xx}}{2} + R = 183'38 \text{ MPa}$$

$$\tau_{m\acute{a}x} = 96'8 \text{ MPa}$$

$$\sigma_2 = \frac{\sigma_{xx}}{2} - R = -10'22 \text{ MPa}$$

$$\sigma_3 = 0$$

Tensiones principales:      tensión tangencial máxima

$$\sigma_1 = 183'38 \text{ MPa}$$

$$\tau_{m\acute{a}x} = 96'8 \text{ MPa}$$

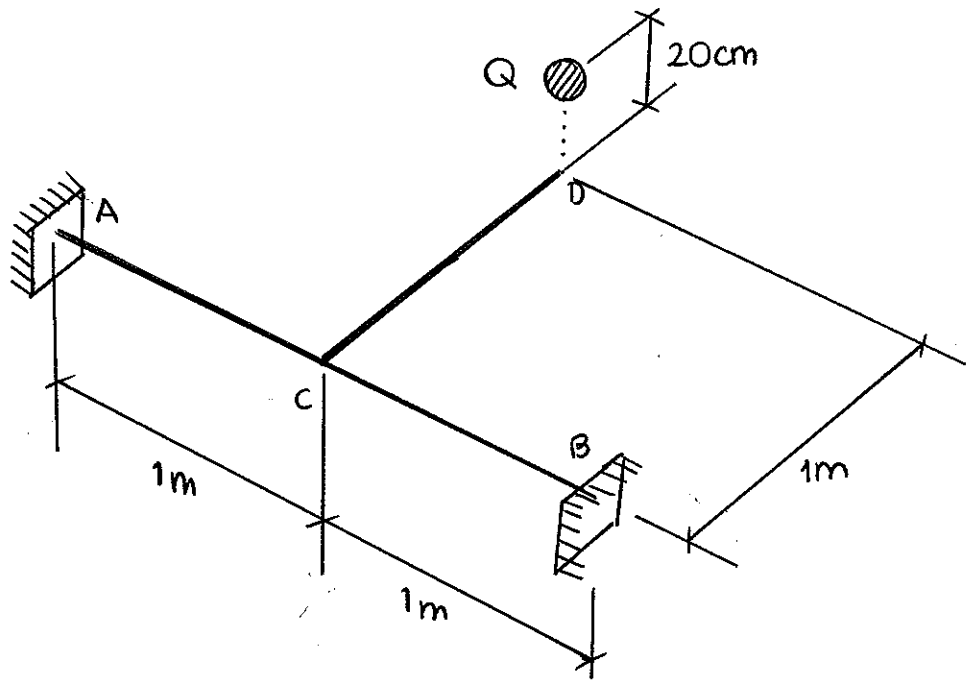
$$\sigma_2 = -10'22 \text{ MPa}$$

$$\sigma_3 = 0$$





6.4

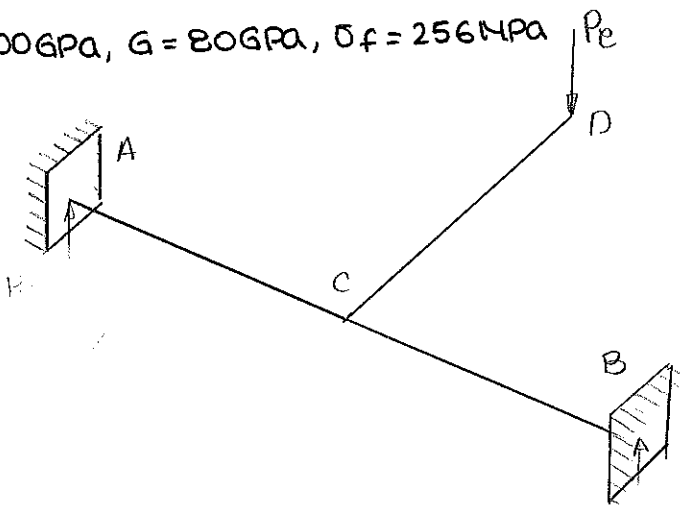


●  $d = 10\text{cm}$

●  $cd$ : indeformable.

●  $n = 1/6$  (tresca).

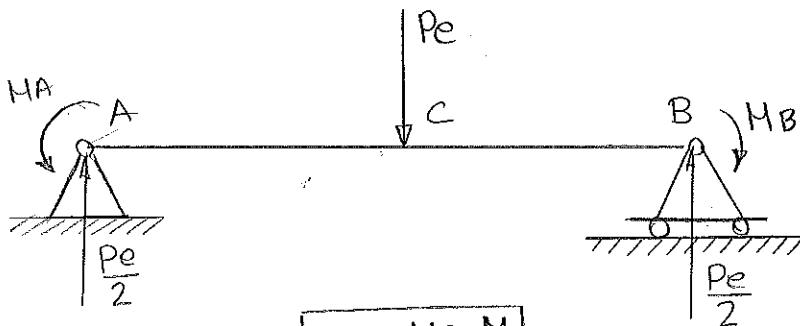
●  $E = 200\text{GPa}$ ,  $G = 80\text{GPa}$ ,  $\sigma_f = 256\text{MPa}$



$$\left. \begin{aligned} U_s &= W_e \\ U_s &= Q(h + \delta) \\ W_e &= \frac{1}{2} P_e \delta \end{aligned} \right\} \boxed{Q(h + \delta) = \frac{1}{2} P_e \delta}$$

$$\delta_D = \delta(\text{flexión } AB) + \delta(\text{torsión } AB)$$

HIPERESTATICIDAD DE GRADO ( $h=2$ )



Por simetría  $\boxed{M_A = M_B = M}$

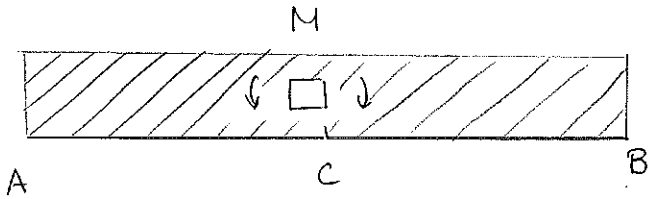
Ec. compatibilidad:

$$\theta_A = \theta_B = 0.$$

$$\rightarrow \boxed{\delta_{BA} = 0}$$

Diagrama de momentos flectores.

(M)



$$\delta_{BA_1}^{\downarrow} = \frac{1}{EI} [M \cdot 2 \cdot 1] = \frac{2M}{EI}$$

$$\delta_{BA_2}^{\uparrow} = \frac{1}{EI} \left[ \frac{1}{2} \frac{Pe}{2} \cdot 2m \cdot 1m \right] = \frac{Pe}{2EI}$$

$$\delta_B^{\downarrow} = \frac{2M}{EI} - \frac{Pe}{2EI} = 0 \Rightarrow \boxed{M_A = M_B = \frac{Pe}{4}}$$

(Pe)

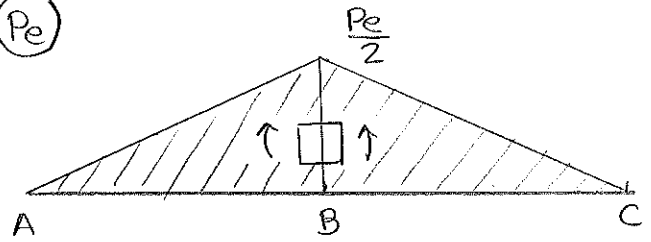
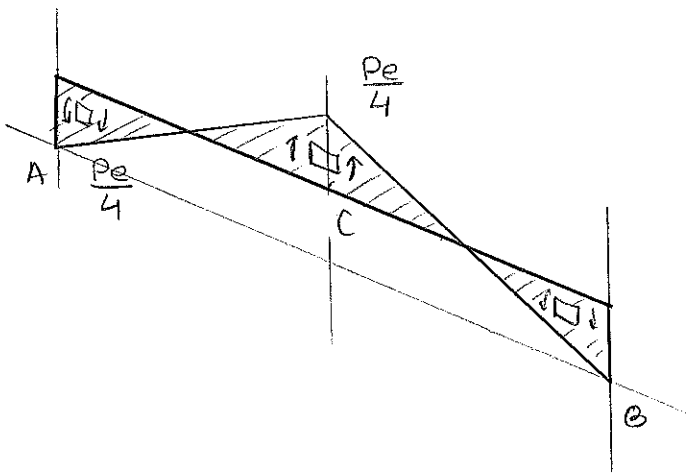


DIAGRAMA DE ESFUERZOS FLECTORES (AB)



$$\delta_c^{\downarrow} = \delta_{c,M}^{\downarrow} + \delta_{c,Pe}^{\downarrow}$$

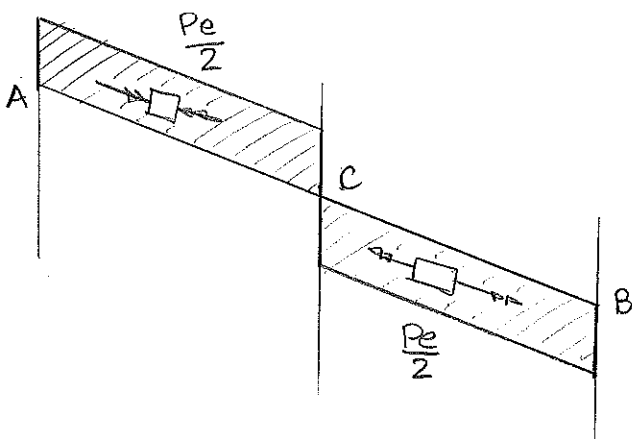
$$\delta_{c,M} = \frac{1}{EI} [M \cdot 1 \cdot 0.5] = \frac{M}{2EI} = \frac{Pe}{8EI}$$

$$\delta_{c,Pe}^{\downarrow} = -\frac{1}{EI} \left[ \frac{1}{2} \frac{Pe}{2} \cdot 1 \cdot \frac{1}{3} \right] = \frac{-Pe}{12EI}$$

$$\delta_c^{\downarrow} = \frac{Pe}{24EI} \text{ (flexión AB)}$$

$$\boxed{\delta_D \text{ (flexión AB)} = \frac{Pe}{24EI}}$$

DIAGRAMA DE ESFUERZOS TORSORES (AB)



$$\psi_{CA} = \frac{\frac{Pe}{2} \cdot 1m}{2GI} = \frac{Pe}{4GI}$$

$$\delta_D = \psi_{CA} \cdot 1m = \frac{Pe}{4GI}$$

$$\boxed{\delta_D \text{ (torsión AB)} = \frac{Pe}{4GI}}$$

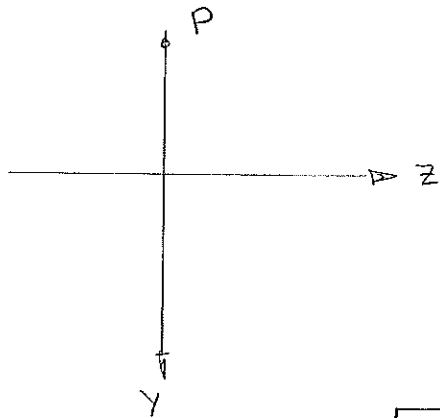
$$I = \frac{\pi R^4}{4} = 4,908738521 \cdot 10^{-6} \text{ m}^4$$

$$\delta = \delta(Pe) = \frac{Pe}{24EI} + \frac{Pe}{4GI} = 7,790610907 \cdot 10^{-7} Pe$$

$$Pe = Pe(\delta) = 1472621,556 \delta$$

SECCIÓN MÁS DESFAVORECIDA DE LA BARRA: EMPOTRAMIENTO (A)

Sección A



⊗ Punto más desfavorado: P.

$$\sigma_{xx} = \frac{Pe/4 \cdot 5\text{cm}}{I_z} = 2546,47909 \cdot Pe$$

$$\tau_{xz} = \frac{Pe/2 \cdot 5\text{cm}}{2I_z} = 2546,47909 \cdot Pe$$

⊗ Tresca :  $\sigma_{adm} = \sqrt{\sigma_{xx}^2 + 4\tau_{xz}^2} = 5694,100348 Pe \leq \frac{\sigma_f}{n}$

$$Pe \leq 28099,25892 \text{ N}$$

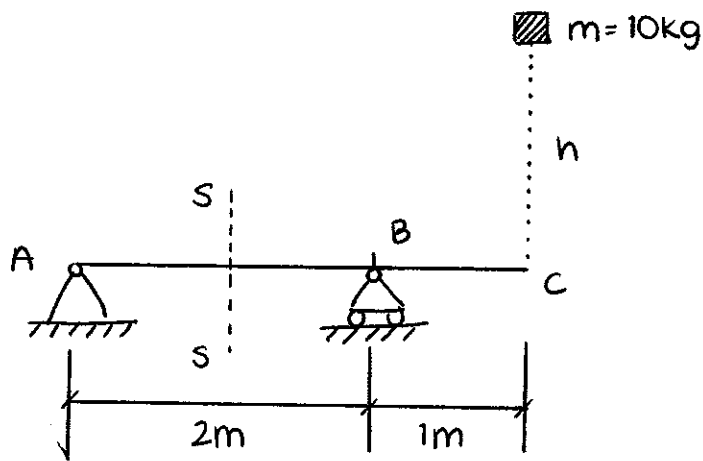
$$\delta = 0,01908111341 \text{ m}$$

$$Q(h+\delta) = \frac{1}{2} Pe \cdot \delta \Rightarrow Q = 1223,668 \text{ N} = 122,367 \text{ kg}$$

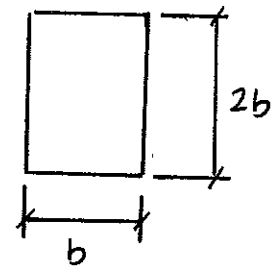
$$Q = 122,367 \text{ kg}$$



6.5



sección SS



$$\sigma_r = 100 \text{ MPa}$$

$$E = 100 \text{ GPa}$$

$$b = 6 \text{ cm}$$

$$I = \frac{1}{12} b(2b)^3 = \frac{8}{12} b^4 = 8'64 \cdot 10^{-6} \text{ m}^4$$

$$\left. \begin{array}{l} U_s = Q(h + \delta) \\ W_e = \frac{1}{2} P_e \cdot \delta \\ U_s = W_e \end{array} \right\} Q(h + \delta) = \frac{1}{2} P_e \cdot \delta$$

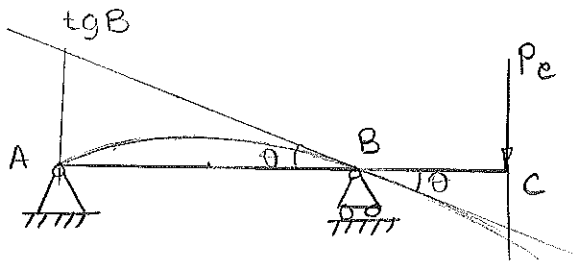
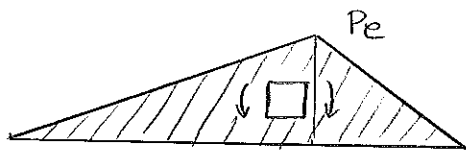


Diagrama de esfuerzos flectores.



$$\theta = \frac{2P_e}{3EI}$$

$$\delta_{CB} = \frac{1}{EI} \left[ \frac{1}{2} P_e \cdot 1 \cdot \frac{2}{3} \cdot 1 \right] = \frac{P_e}{3EI}$$

$$\delta_c = \frac{P_e}{3EI} + \frac{2P_e}{3EI} = \frac{P_e}{EI}$$

$$\delta_c = \delta_{CB} + \theta \cdot 1 \text{ m}$$

$$\theta = \frac{\delta_{AB}}{2 \text{ m}}$$

$$\delta_{AB} = \frac{1}{EI} \left[ \frac{1}{2} P_e \cdot 2 \cdot \frac{2}{3} \right] = \frac{4P_e}{3EI}$$

SECCIÓN MÁS DESFAVORECIDA: sección B

$$\sigma_{xx} = \frac{P_e \cdot 6\text{cm}}{I} < \sigma_r \Rightarrow P_e = 14400 \text{ N}$$

$$\delta = \frac{P_e}{EI} = 0'0167 \text{ m}$$

$$Q(h+\delta) = \frac{1}{2} P_e \cdot \delta \Rightarrow h = 1'2 \text{ m} \Rightarrow \boxed{h_{\text{máx}} = 120 \text{ cm}}$$

↑  
se desprecia

$$h = 100 \text{ cm}$$

$$\sigma_{xx} < 60 \text{ MPa}$$

$$\sigma_{xx} = \frac{P_e \cdot b}{I} = \frac{P_e \cdot b}{\frac{8}{12} b^4} = \frac{3P_e}{2b^3} = 60 \text{ MPa}$$

$$\boxed{P_e = 40 \cdot 10^6 \cdot b^3}$$

$$\delta = \frac{P_e}{EI} = \frac{40 \cdot 10^6 \cdot b^3}{100 \cdot 10^9 \cdot \frac{8}{12} b^4} = 6 \cdot 10^{-4} \frac{1}{b}$$

$$Q \cdot h = \frac{1}{2} P_e \delta = \frac{1}{2} \cdot 40 \cdot 10^6 b^3 \cdot 6 \cdot 10^{-4} \frac{1}{b} = 12000 b^2$$

$$b = 0,09120 \text{ m} = 9'129 \text{ cm}$$

$$\boxed{b = 9'129 \text{ cm}}$$

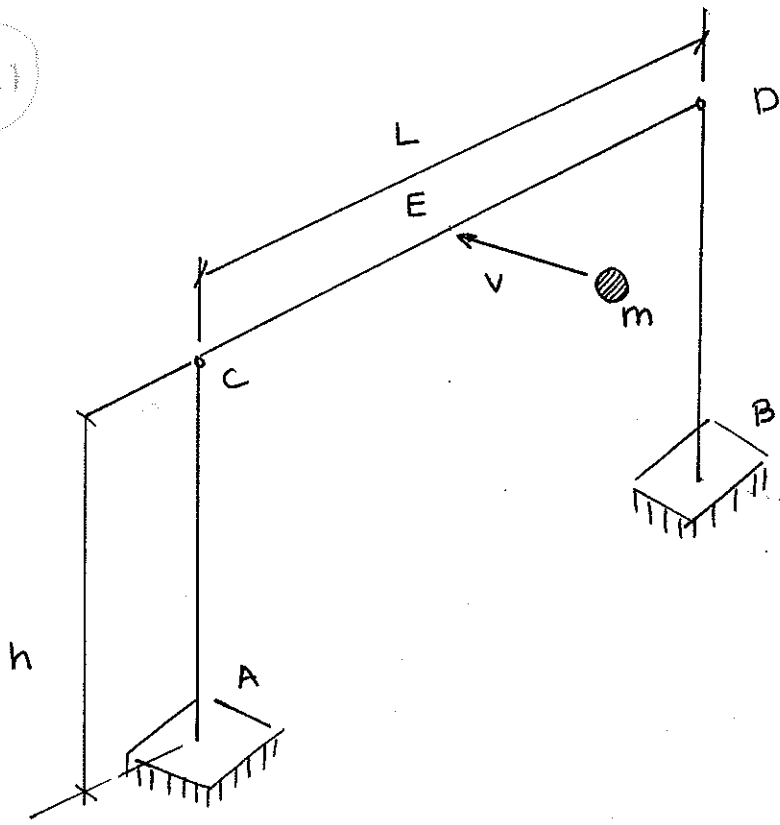
$$h = 50 \text{ cm}$$

$$b = 6 \text{ cm}$$

$$Qh = \frac{1}{2} P_e \delta = \frac{1}{2} EI \cdot \delta^2 \Rightarrow \delta = 0'010758 \text{ m} = 1'0758 \text{ cm}$$

$$\boxed{\delta = 1'0758 \text{ cm}}$$

C-6.1

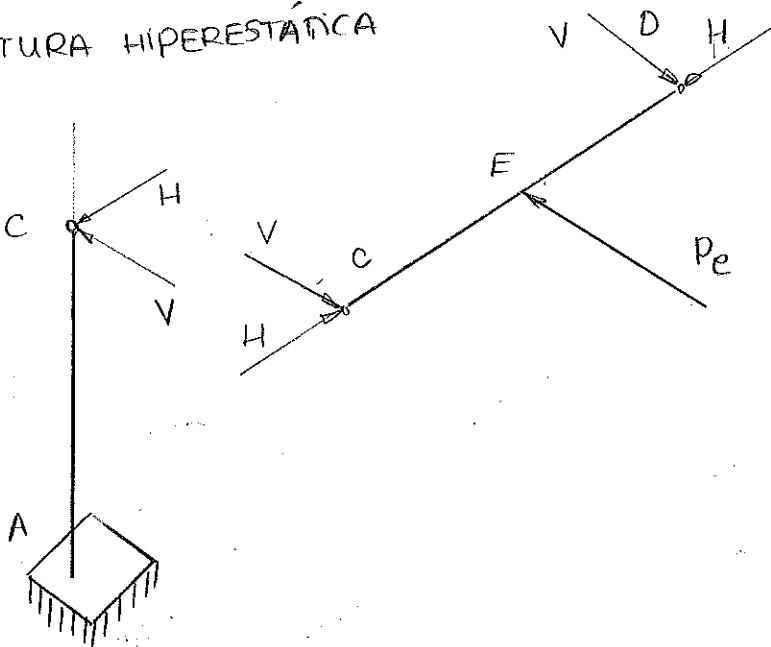


- ⊙ sección barras: 4cm x 4cm
- ⊙ L=30cm, h=20cm
- ⊙ m=1kg, v=10  $\frac{m}{s}$
- ⊙ g=10  $\frac{m}{s^2}$
- ⊙ E=100 GPa  $\sigma_r=200$  MPa

$U_s = W_e$

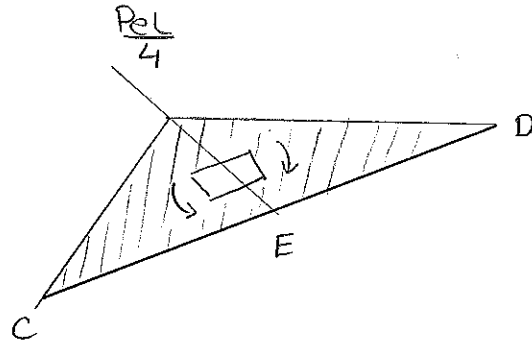
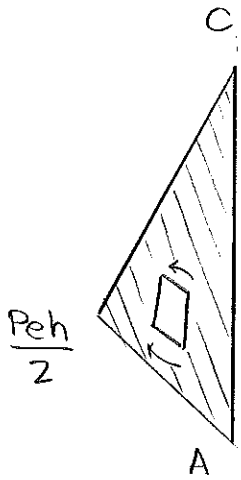
$$\left. \begin{aligned} U_s &= \frac{1}{2} m \cdot v^2 \\ W_e &= \frac{1}{2} P_e \cdot \delta \end{aligned} \right\} m \cdot v^2 = P_e \cdot \delta$$

ESTRUCTURA HIPERESTÁTICA

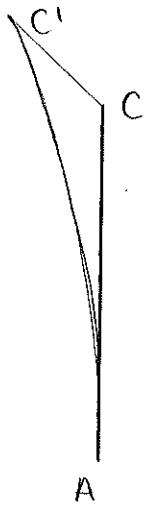


EQUILIBRIO:  $P_e = 2V \Rightarrow V = \frac{P_e}{2}$

Diagrama de esfuerzos flectores

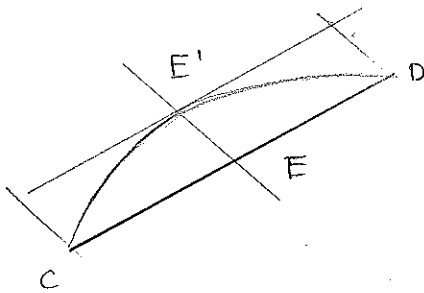


$\delta_E = \delta_E \text{ (flexión AC)} + \delta_E \text{ (flexión CED)}$



$$\delta_E = \delta_{CA} = \frac{1}{EI} \left[ \frac{1}{2} \cdot \frac{Pe \cdot h}{2} \cdot h \cdot \frac{2}{3} \cdot h \right] = \frac{Pe \cdot h^3}{6EI}$$

$$\delta_E = \delta_{CE} = \frac{1}{EI} \left[ \frac{1}{2} \cdot \frac{PeL}{4} \cdot \frac{L}{2} \cdot \frac{2}{3} \cdot \frac{L}{2} \right] = \frac{PeL^3}{48EI}$$



$$\delta = Pe \left( \frac{h^3}{6EI} + \frac{L^3}{48EI} \right)$$



Las secciones más frágiles: A y B (empotramientos)

$$\sigma_{xx, \text{máx}} = \frac{P_e \cdot \frac{h}{2} \cdot 2 \text{ cm}}{I} = 200 \text{ MPa} \Rightarrow P_e = 21333,33 \text{ N}$$

$$\delta = 1,895833 \cdot 10^{-3}$$

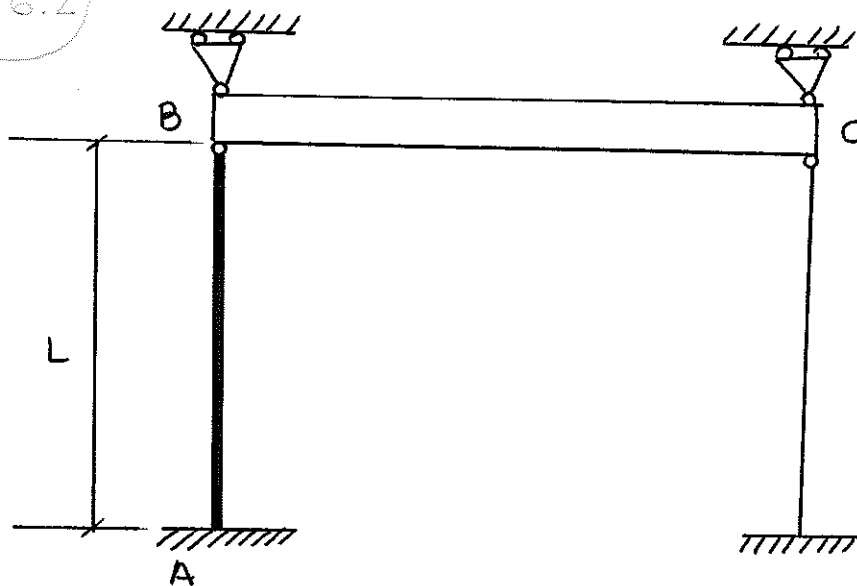
$$I = \frac{1}{12} (4 \text{ cm})^4 = 2,133 \cdot 10^{-7} \text{ m}^4$$

$$W_e = \frac{1}{2} P_e \cdot \delta = 20,222 \text{ N} \cdot \text{m} \quad \left. \begin{array}{l} U_s > W_e \\ U_s = 50 \text{ Nm} \end{array} \right\} \text{La estructura rompien A y B}$$

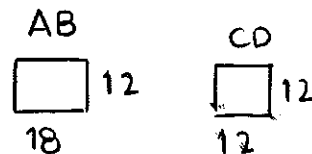
$$U_r = U_s - W_e = 29,778 \text{ Nm} = \frac{1}{2} m v_r^2 \Rightarrow v_r = 7,717 \frac{\text{m}}{\text{s}}$$



C-6.2



Secciones (cm)



$$I_{AB} = \frac{1}{12} 12 \text{ cm} (18 \text{ cm})^3 = 5,832 \cdot 10^{-5} \text{ m}^4$$

$$I_{CD} = \frac{1}{12} (12 \text{ cm})^4 = 1728 \cdot 10^{-5} \text{ m}^4$$

⊙  $L = 2 \text{ m}$

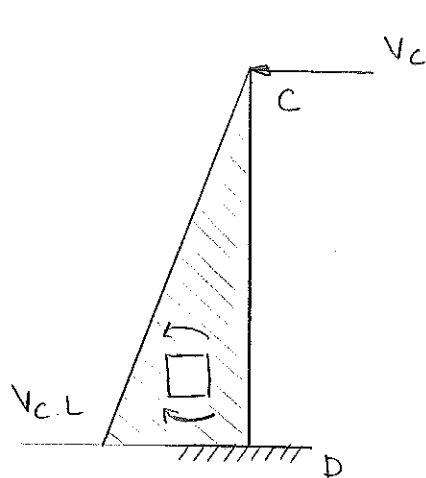
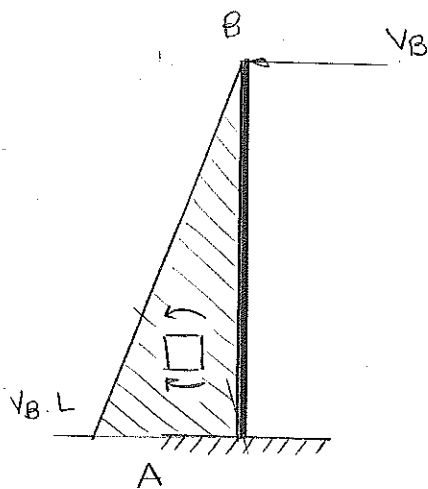
⊙  $\sigma_r = 200 \text{ MPa}$

⊙  $E = 200 \text{ GPa}$

$$U_s = \frac{1}{2} m v^2 \quad \left. \begin{array}{l} \\ \\ \end{array} \right\} U_s = W_e$$

$W_e$

⊙  $g = 10 \frac{\text{m}}{\text{s}^2}$



$$V_B + V_C = P_e$$

$$3,375 V_C + V_C = P_e$$

$$V_C = 0,22857 P_e$$

$$V_B = 0,77143 P_e$$

$$E_c \text{ compatibilidad: } \delta_B = \delta_C$$

$$\left. \begin{array}{l} \delta_B = \delta_{BA} = \frac{1}{EI_{AB}} \left[ \frac{1}{2} V_B L \cdot L \cdot \frac{2}{3} L \right] = \frac{V_B L^3}{3EI_{AB}} \\ \delta_C = \delta_{CD} = \frac{1}{EI_{CD}} \left[ \frac{1}{2} V_C L \cdot L \cdot \frac{2}{3} L \right] = \frac{V_C L^3}{3EI_{CD}} \end{array} \right\} V_B = \frac{I_{AB}}{I_{CD}} \cdot V_C = 3,375 V_C$$

⊙ SECCIÓN A

$$\sigma_{xx} = \frac{0'77143 P_e L \cdot 9\text{cm}}{I_{AB}} = 200\text{MPa} : P_e = 83999,84\text{N} \dots$$

⊙ SECCIÓN D

$$\sigma_{xx} = \frac{0'22857 P_e L \cdot 6\text{cm}}{I_{CD}} = 200\text{MPa} : P_e = 126000,7875\text{N}$$

⊙ Rompe primero la barra AB con  $P_e = 83999,84\text{N}$

$$W_e = \frac{1}{2} V_B \delta_B + \frac{1}{2} V_C \delta_C = 1622,22\text{Nm}$$

$$\delta = \frac{0'77143 P_e \cdot L^3}{3 E I_{AB}} = 0'0148148\text{m}$$

$$\boxed{U_s = W_e} \quad \frac{1}{2} m v^2 = W_e \Rightarrow \boxed{v = 11,155 \frac{\text{m}}{\text{s}}}$$

$$W_e(\text{fractura total}) = \frac{1}{2} (V_B \delta_B) + \frac{1}{2} P_e \delta_C = 799,999\text{Nm}$$

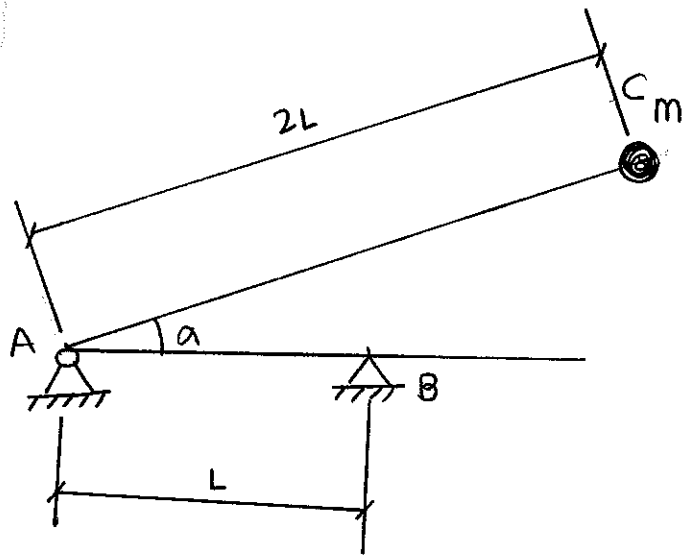
$$\frac{P_c \cdot L \cdot 6\text{cm}}{I_{CD}} = 200\text{MPa} \Rightarrow P_c = 28800\text{N}$$

$$\delta_C = 0'0222\text{m}$$

$$\boxed{U_s = W_e} \quad \frac{1}{2} m v^2 = W_e \Rightarrow \boxed{v = 12'649 \frac{\text{m}}{\text{s}}}$$

$$\boxed{\Delta = 2,22\text{cm}}$$

C-6.3



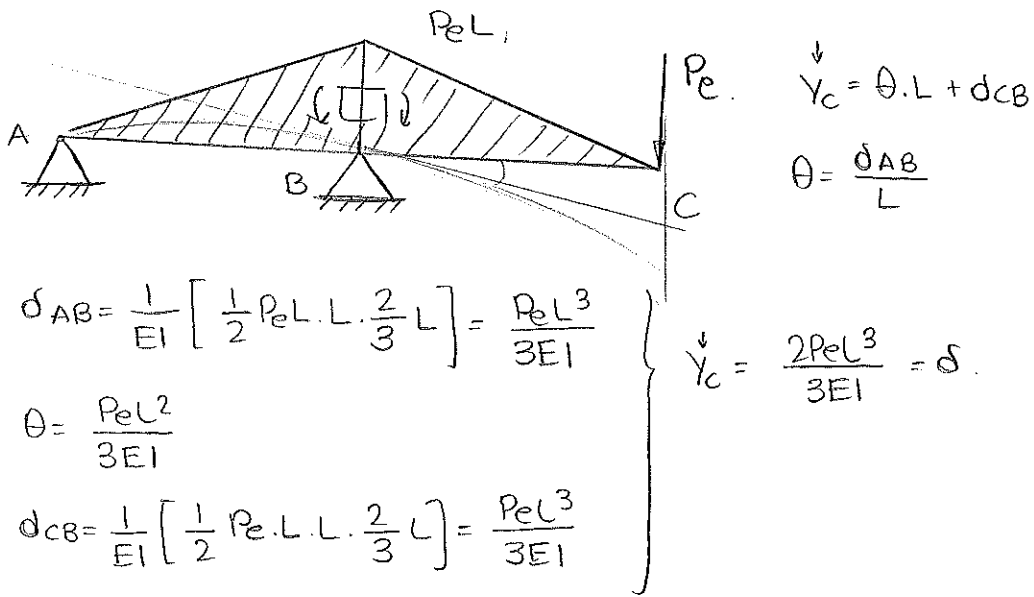
- $m = 10 \text{ kg}$
- $\sigma_r = 200 \text{ MPa}$
- $W_z = 36 \text{ cm}^3$
- $EI = 100 \text{ m}^2 \text{ kN}$

①  $U_s = W_e$

$$U_s = m \cdot (2L \cdot \sin \alpha + \delta)$$

$$W_e = \frac{1}{2} P_e \cdot \delta$$

$m(2L \cdot \sin \alpha + \delta) = \frac{1}{2} P_e \cdot \delta$



SECCIÓN MÁS DESFAVORECIDA: sección B

$$M_z = P_e \cdot L \Rightarrow \sigma_{xx} = \frac{M_z}{W_z} = \frac{P_e \cdot L}{W_z} = \sigma_r \Rightarrow \begin{cases} P_e = 7200 \text{ N} \\ \delta = 0.048 \text{ m} \end{cases}$$

$$m(2L \cdot \sin \alpha + \delta) = \frac{1}{2} P_e \cdot \delta$$

$\alpha = 57'140''$

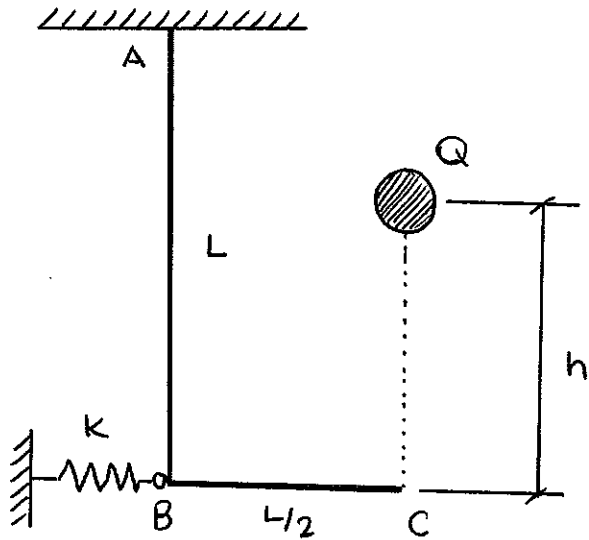
$$(2) \quad u_s = m_2 l \sin \alpha$$

$$W_e = \frac{1}{2} P_e \delta$$

$$m_2 l \sin \alpha = \frac{1}{2} P_e \delta$$

$$\alpha = 59'769$$

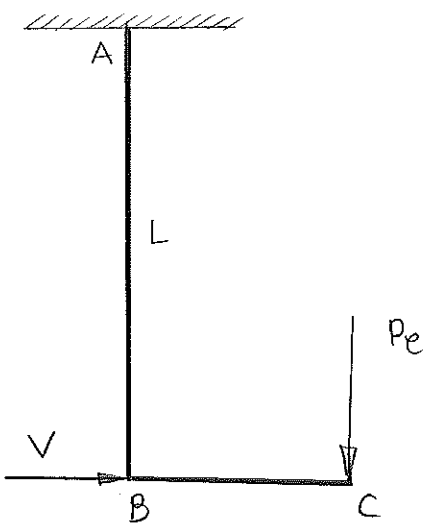
C-6.4



- $k(\text{muelle}) = 200 \frac{\text{KN}}{\text{m}}$
- $Q = 20 \text{kg}$
- $h = 80 \text{cm}$
- $d = 8 \text{cm}$
- $E = 200 \text{GPa}, \sigma_f = 280 \text{MPa}$
- $L = 1 \text{m}$

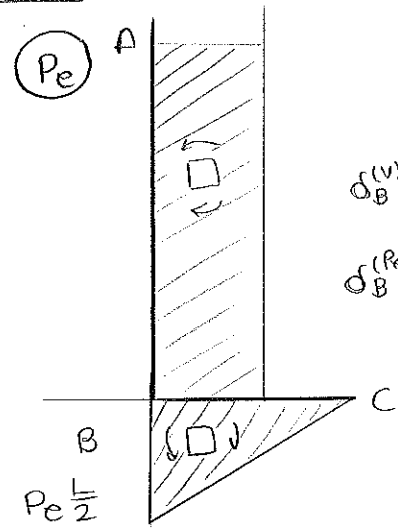
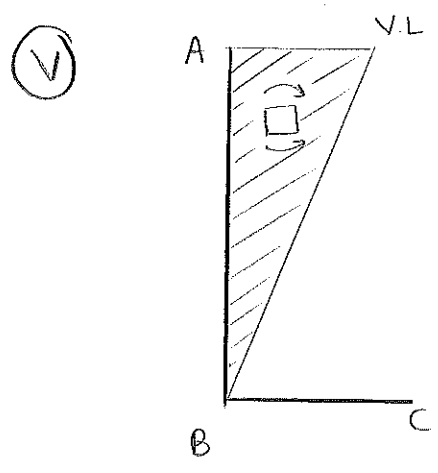
$$I = \frac{\pi (4 \text{cm})^4}{4} = 2,01061929810^{-6} \text{m}^4$$

$$\left. \begin{aligned} U_s &= W_e \\ U_s &= Q(h + \delta) \\ W_e &= \frac{1}{2} P_e \delta + U_m \end{aligned} \right\} Q(h + \delta) = \frac{1}{2} P_e \delta + U_m$$



Ec. compatibilidad:  $\delta_B = \delta_m = \frac{V}{K}$

DIAGRAMAS DE ESFUERZOS FLECTORES



$$\delta_B = \delta_B(\text{flexión } V) + \delta_B(\text{flexión } P_e)$$

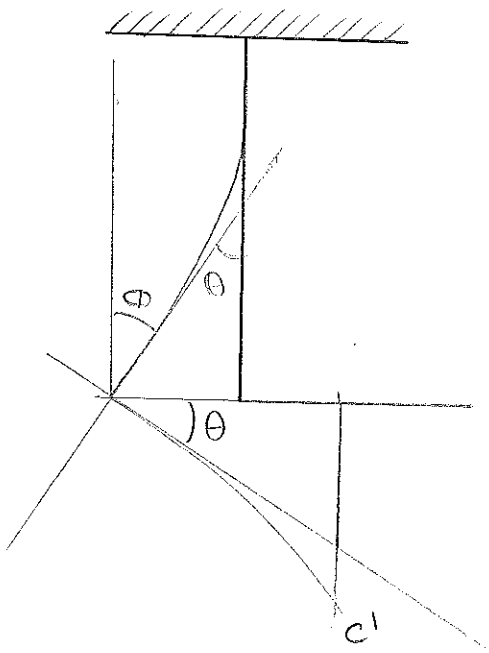
$$\delta_B^{(V)} = \frac{1}{EI} \left[ \frac{1}{2} V L \cdot L \cdot \frac{2}{3} L \right] = \frac{V L^3}{3EI}$$

$$\delta_B^{(P_e)} = \frac{1}{EI} \left[ P_e \frac{L}{2} \cdot L \cdot \frac{L}{2} \right] = \frac{P_e L^3}{4EI}$$

$$\sigma_B = \frac{VL^3}{3EI} + \frac{PeL^4}{4EI} = \frac{V}{K}$$

$$V \left( \frac{L^3}{3EI} + \frac{1}{K} \right) = + \frac{PeL^4}{4EI}$$

$$V = \frac{PeL^4}{4EI} \cdot \frac{1}{\frac{1}{K} + \frac{L^3}{3EI}} = 0,1066574455 Pe$$



$$y_c = \theta_{BA} \cdot \frac{L}{2} + \delta_{CB}$$

$$\theta = \theta_{BA} = \frac{1}{EI} \left[ Pe \frac{L}{2} \cdot L - \frac{1}{2} VL \cdot L \right] = \frac{0,4466712773 Pe}{EI}$$

$$\delta_{CB} = \frac{1}{EI} \left[ \frac{1}{2} Pe \cdot \frac{L}{2} \cdot \frac{L}{2} \cdot \frac{2}{3} \frac{L}{2} \right] = \frac{PeL^3}{24EI}$$

$$y_c = \frac{Pe}{EI} 0,2650023053 = \delta$$

$$Pe = Pe(\delta) = 1517435,326 \delta$$

$$Q(h+\delta) = \frac{1}{2} Pe \cdot \delta = 758717,6631 \delta^2$$

$$\delta = 0,01465 \text{ m}$$

$$\delta_c = 1,465 \text{ cm}$$

$$Pe = 22236,77164 \text{ N}$$

$$\delta_m = \frac{V}{K} = 0,01186 \text{ m}$$

$$\delta_m = 1,186 \text{ cm}$$

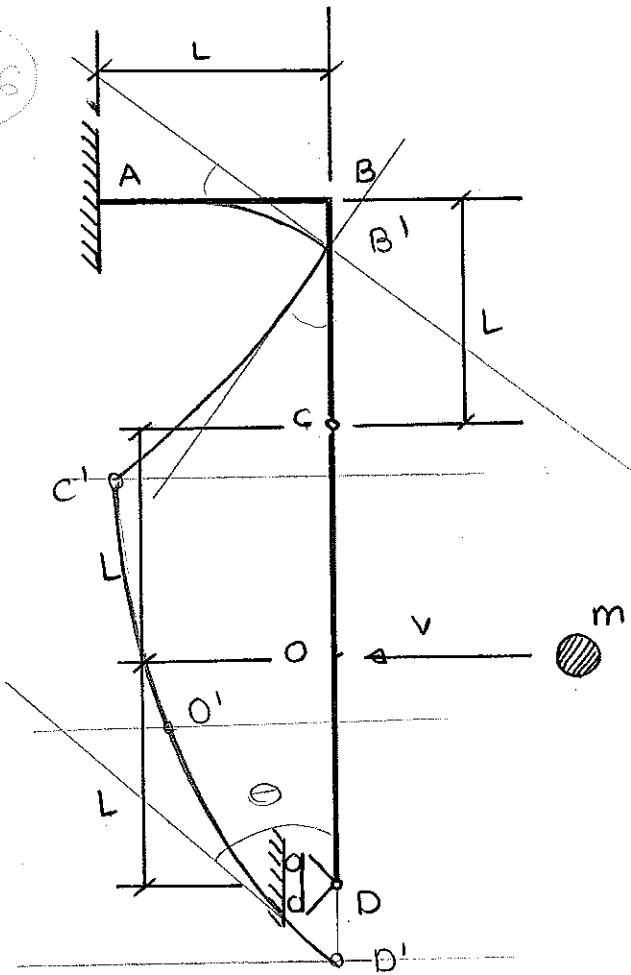
$$\sigma_{xx, \max} = \frac{M_{\max} R}{I} = 221,193 \text{ MPa}$$

$$n = 1,266$$

$$n = \frac{\sigma_c}{\sigma_{xx, \max}} = 1,266$$



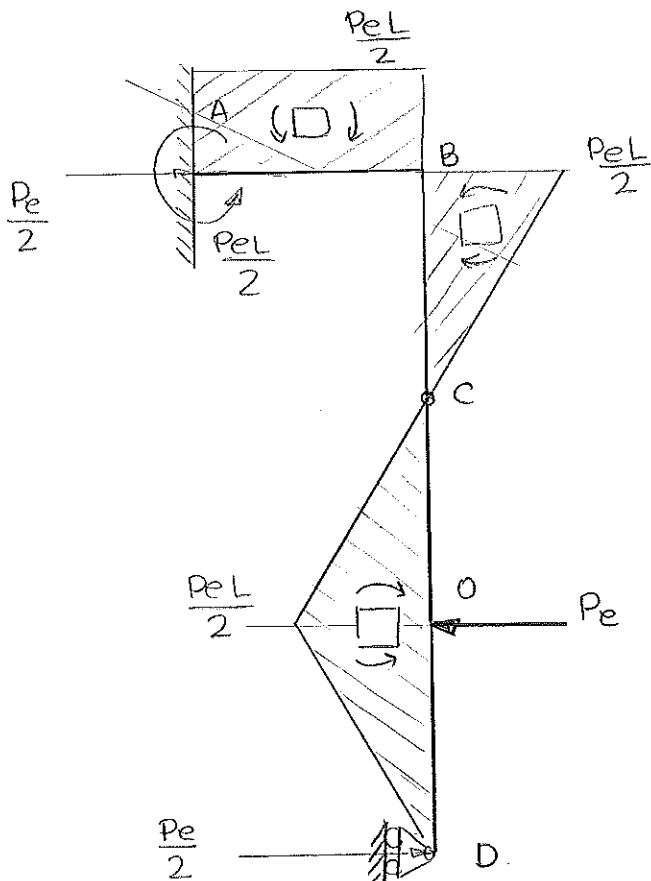
C-6.6



- $L = 100\text{cm}$
- sección:  $6 \times 6\text{cm}^2$
- $\sigma_r = 200\text{MPa}$
- $m = 1\text{kg}$
- $g = 10 \frac{\text{m}}{\text{s}^2}$

$$I = \frac{1}{12} (6\text{cm})^4 = 1,08 \cdot 10^{-6} \text{m}^4$$

$$\left. \begin{aligned} U_s &= W_e \\ U_s &= \frac{1}{2} m \cdot v^2 \\ W_e &= \frac{1}{2} P_e \cdot \delta_o \end{aligned} \right\} \frac{1}{2} m \cdot v^2 = \frac{1}{2} P_e \cdot \delta_o : \quad \boxed{mv^2 = P_e \cdot \delta_o}$$



$$M_A = P_e \cdot 2L - \frac{P_e}{2} \cdot 3L = \frac{P_e L}{2}$$

$$\delta_{xO} = \theta \cdot L - \delta_{OD}$$

$$\theta = \frac{\delta_{xc} + \delta_{cd}}{2L}$$

$$\delta_{xc} = \theta_{BA} \cdot L + \delta_{CB}$$

$$\theta_{BA} = \frac{1}{EI} \left[ \frac{P_e L}{2} \cdot L \right] = \frac{P_e L^2}{2EI}$$

$$\delta_{CB} = \frac{1}{EI} \left[ \frac{1}{2} \cdot \frac{P_e L}{2} \cdot L \cdot \frac{2}{3} L \right] = \frac{P_e L^3}{6EI}$$

$$\delta_{xc} = \frac{2P_e L^3}{3EI}$$

$$\delta_{cd} = \frac{1}{EI} \left[ \frac{1}{2} \cdot \frac{P_e L}{2} \cdot 2L \cdot L \right] = \frac{P_e L^3}{2EI}$$

$$\theta = \frac{7 PeL^2}{12 EI}$$

$$\delta_{00} = \frac{1}{EI} \left[ \frac{1}{2} \frac{PeL}{2} \cdot L \cdot \frac{1}{3} L \right] = \frac{PeL^3}{12EI}$$

$$\delta_{x0} = \frac{PeL^2}{2EI}$$

⊗ Esfuerzo flector máximo:

$$\sigma_{xx} = \frac{M_{\max} \cdot 3cm}{I} = \frac{Pe \frac{L}{2} \cdot 3cm}{1'08 \cdot 10^6 m^4} = \sigma_r \Rightarrow Pe = 14400N$$

$$\delta_{x0} = 0'033m.$$

$$m \cdot v^2 = Pe \cdot \delta \Rightarrow v = 21'909 \frac{m}{s}$$

$$v = 25 \frac{m}{s}$$

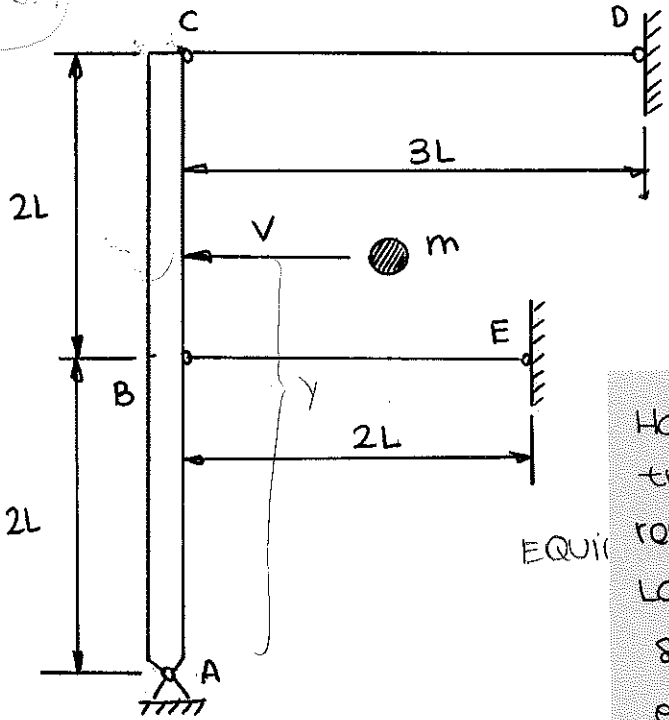
$$U_s = \frac{1}{2} m v^2 = 312,5 \text{ Nm.}$$

$$W_r = \frac{1}{2} Pe \cdot \delta_{x0} = 240 \text{ Nm}$$

$$U_r = U_s - W_r = 72'5 \text{ Nm} = \frac{1}{2} m v_r^2$$

$$v_r = 12'042 \frac{m}{s}$$

C-8.7

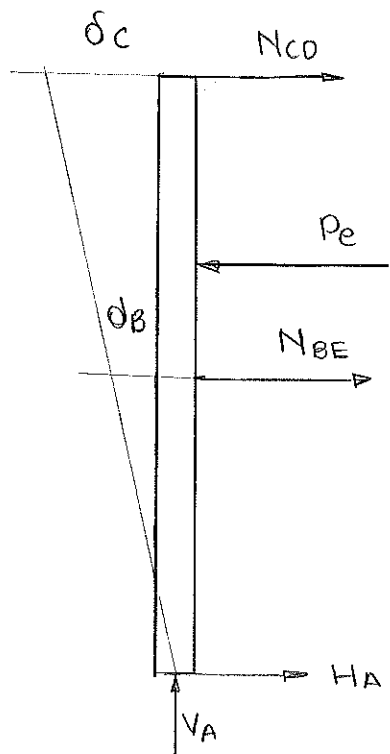


- ⊙  $L = 1\text{m}$
- ⊙  $\sigma_r = 200\text{MPa}$
- ⊙  $E = 200\text{GPa}$
- ⊙  $A_T = 2\text{cm}^2$
- ⊙  $m = 10\text{Kg}$

Hay que calcular el trabajo que supone romper cada tirante. La energía de la bola se va a traducir en eso.

$L = Pe \cdot y$

EQUIV



EC COMPATIBILIDAD:  $\delta_c = 2 \cdot \delta_b$

$$\left. \begin{aligned} \delta_c &= \frac{N_{cd} L_{cd}}{E \cdot A} \\ \delta_b &= \frac{N_{be} L_{be}}{E \cdot A} \end{aligned} \right\} N_{cd} L_{cd} = 2 N_{be} L_{be}$$

$N_{be} = \frac{3}{4} N_{cd}$

⊙ Para que rompa tirante (CD) (Rompe primero CD)

$$\sigma_{cd} = \frac{N_{cd}}{A} = 200\text{MPa} \Rightarrow \begin{cases} N_{cd} = 40000\text{N} \\ N_{be} = 30000\text{N} \end{cases}$$

⊙ Para que rompa tirante (BE)

$$\sigma_{xx} = \frac{N_{be}}{A} = 200\text{MPa} \Rightarrow \begin{cases} N_{be} = 40000\text{N} \\ N_{cd} = 53333,33\text{N} \end{cases} \quad (*)$$

Ambos tirantes rompen para  $N_{be} = 40000\text{N}$  y  $N_{cd} = 53333,33\text{N}$

$$\delta_c(\text{rotura}) = 3 \cdot 10^{-3} \text{ m}$$

$$\delta_B(\text{rotura}) = 2 \cdot 10^{-3} \text{ m}$$

$$W_e = W_c(\text{rotura}) + W_B(\text{rotura}) = \frac{1}{2} 40.000 \cdot 3 \cdot 10^{-3} + \frac{1}{2} 40.000 \cdot 2 \cdot 10^{-3} = 100 \text{ Nm}$$

$$U_s = W_e$$

$$U_s = \frac{1}{2} mv^2$$

$$\frac{1}{2} mv^2 = 100$$

$$v = 4,472 \frac{\text{m}}{\text{s}}$$

$$v = 6 \frac{\text{m}}{\text{s}}$$

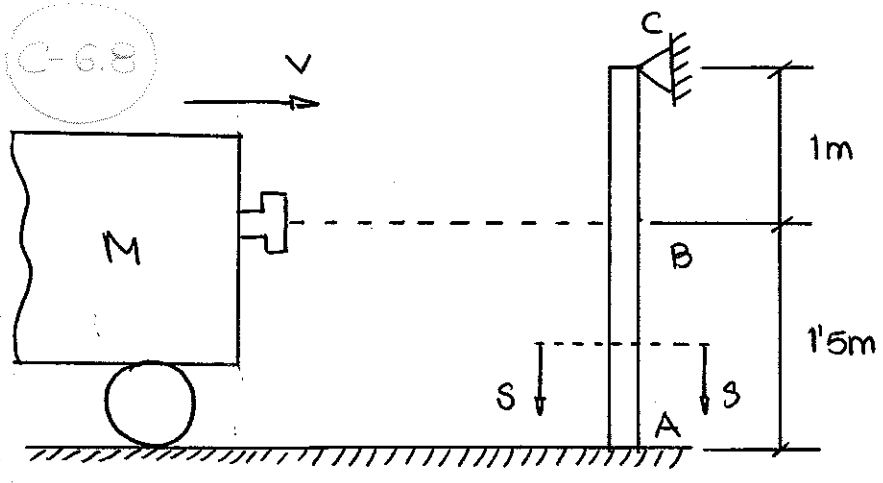
$$U_s = \frac{1}{2} mv^2 = 180 \text{ Nm}$$

$$W_e = W_c(\text{rotura}) + W_B(\text{deformación}) = \frac{1}{2} 40.000 \cdot 3 \cdot 10^{-3} + \frac{1}{2} 30.000 \cdot 15 \cdot 10^{-3} = 82,5 \text{ Nm}$$

$$U_r = U_s - W_e = 97,5 \text{ Nm} = \frac{1}{2} mv^2$$

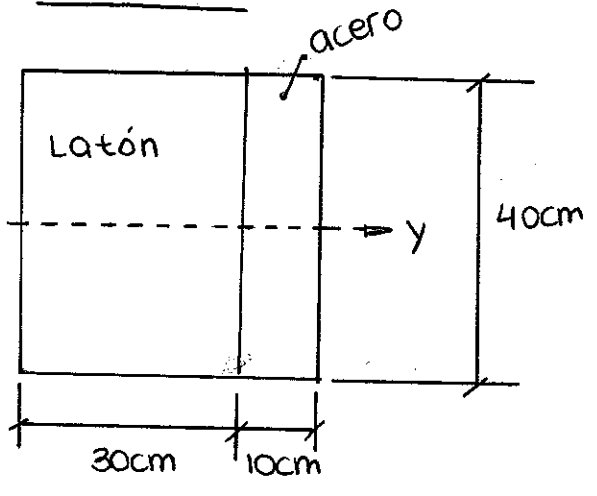
$$v = 4,416 \frac{\text{m}}{\text{s}}$$

C-6.8

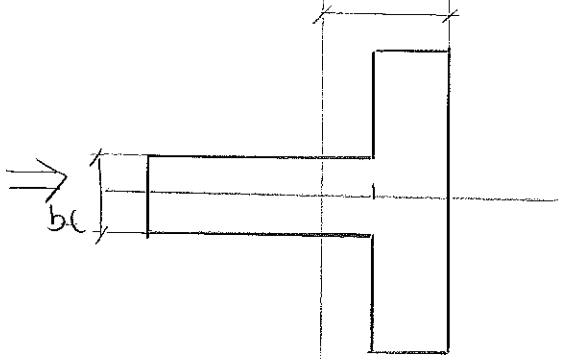


$\odot M = 50000 \text{ kg} ; v = 0.3 \frac{\text{m}}{\text{s}}$   
 $\odot E_a = 200 \text{ GPa} ; E_l = 100 \text{ GPa}$

sección s-s



sección de acero equivalente



$b_e = b \cdot \frac{E_l}{E_a} = 20 \text{ cm.}$

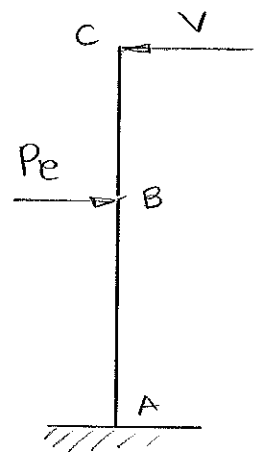
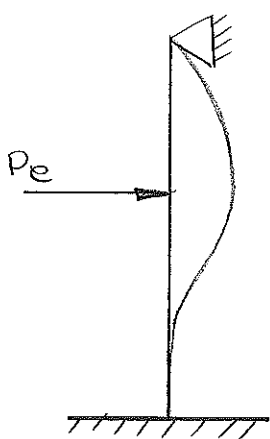
$y_G = \frac{30 \text{ cm} \cdot 20 \text{ cm} \cdot 25 \text{ cm} + 10 \text{ cm} \cdot 40 \text{ cm} \cdot 5 \text{ cm}}{(30 \text{ cm} \cdot 20 \text{ cm} + 10 \text{ cm} \cdot 40 \text{ cm})} = 17 \text{ cm}$

$I_2^1 = 4.333 \cdot 10^{-3} \text{ m}^4 \Rightarrow I_2 = 1.44333 \cdot 10^{-3} \text{ m}^4$

$U_s = \frac{1}{2} M \cdot v^2$   
 $W_e = \frac{1}{2} P_e \cdot \delta_e$

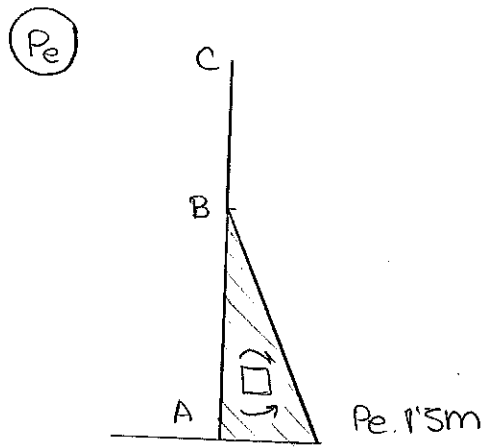
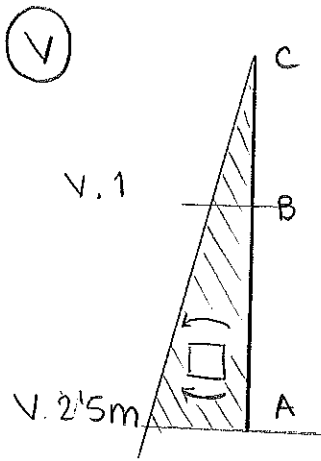
$U_s = W_e$

ESTRUCTURA HIPERESTÁTICA: (h=1) {H}



Eccompatibilidad:  $\delta_c = 0$

## Diagramas de esfuerzos flectores



$$\delta_c = \delta_c^{(V)} - \delta_c^{(Pe)} = 0.$$

$$\delta_c^{(V)} = \frac{1}{EI} \left[ \frac{1}{2} \cdot V \cdot 2.5m \cdot 2.5m \cdot \frac{2}{3} \cdot 2.5m \right] = \frac{125V}{24EI}$$

$$\delta_c^{(Pe)} = \frac{1}{EI} \left[ \frac{1}{2} Pe \cdot 1.5m \cdot 1.5m \cdot \left( \frac{2}{3} \cdot 1.5m + 1 \right) \right] = \frac{9Pe}{4EI}$$

$$\delta_c^{(V)} = \delta_c^{(Pe)} \Rightarrow \frac{125V}{24} = \frac{9Pe}{4} \Rightarrow \boxed{V = 0.432 Pe}$$

$$\delta_B = \delta_B^{(Pe)} - \delta_B^{(V)}$$

$$\delta_B^{(Pe)} = \frac{1}{EI} \left[ \frac{1}{2} Pe \cdot 1.5m \cdot 1.5m \cdot \frac{2}{3} \cdot 1.5m \right] = \frac{9Pe}{8EI}$$

$$\delta_B^{(V)} = \frac{1}{EI} \left[ V \cdot 1.5m \cdot \frac{1.5m}{2} + \frac{1}{2} V \cdot 1.5m \cdot 1.5m \cdot \frac{2}{3} \cdot 1.5m \right] = \frac{9V}{4EI} = \frac{0.972 Pe}{EI}$$

$$\boxed{\delta_B = 0.153 \frac{Pe}{EI}}$$

$$U_s = W_e$$

$$M \cdot V^2 = Pe \cdot \delta = Pe \cdot 0.153 \cdot \frac{Pe}{EI}$$

$$Pe = 2918794,104N$$

Esfuerzo flector máximo  $\Rightarrow$  A, B

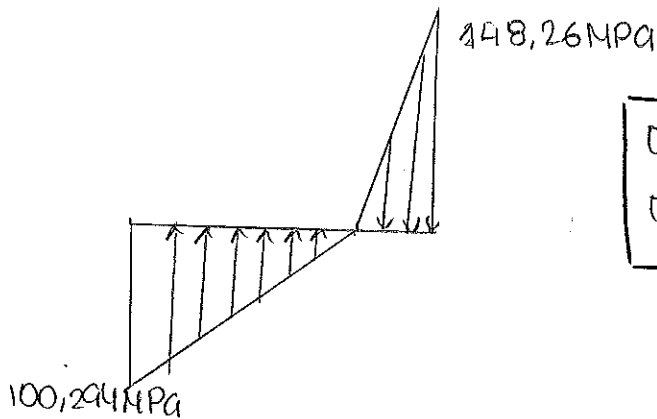
$$\left. \begin{aligned} M_2(A) &= 1223793,524 \text{ Nm} \\ M_2(B) &= 1258759,053 \text{ Nm} \end{aligned} \right\} \textcircled{B}$$

sección B

$$\sigma_{xx, \text{máx}}(\text{acero}) = \frac{M_2(B) \cdot 17 \text{ cm}}{I_2} = 148,26 \text{ MPa}$$

$$\sigma_{xx, \text{máx}}^a(\text{latón}) = \frac{M_2(B) \cdot 23 \text{ cm}}{I_2} = 200,588 \text{ MPa}$$

$$\sigma_{xx, \text{máx}}^e(\text{latón}) = \frac{E_L}{E_A} \cdot \sigma_{xx, \text{máx}}^a(\text{latón}) = 100,294 \text{ MPa}$$



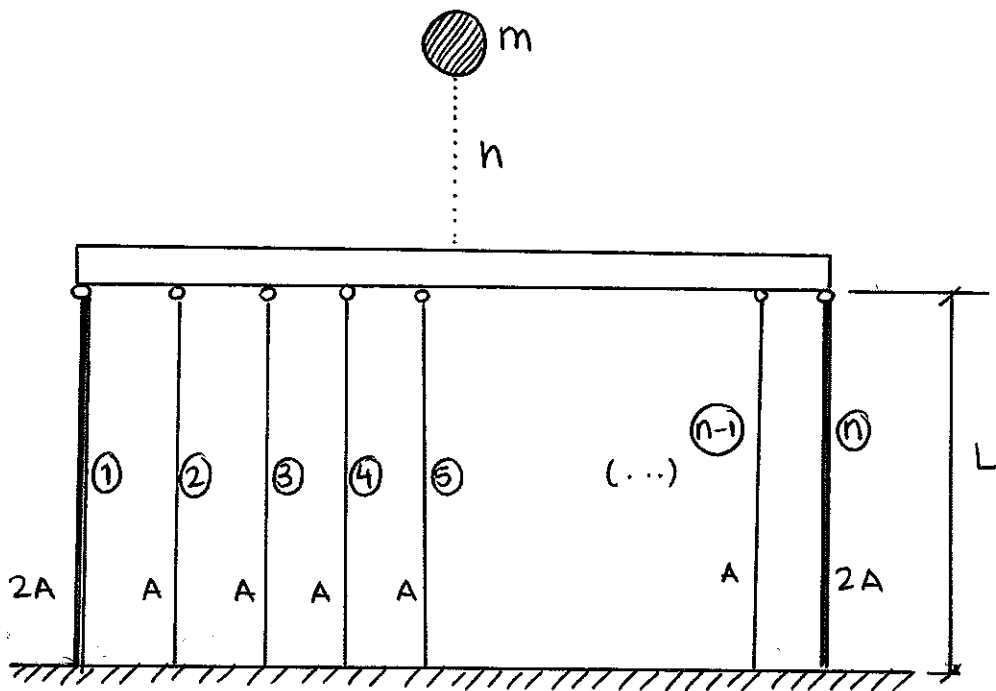
$$\sigma_{\text{máx}}(\text{acero}) = 148,26 \text{ MPa}$$

$$\sigma_{\text{máx}}(\text{latón}) = 100,294 \text{ MPa}$$





2.8.9



●  $L = 1\text{m}$ ,  $h = 1.5\text{m}$

●  $g = 10 \frac{\text{m}}{\text{s}^2}$

●  $E = 100\text{GPa}$

●  $m = 14\text{kg}$

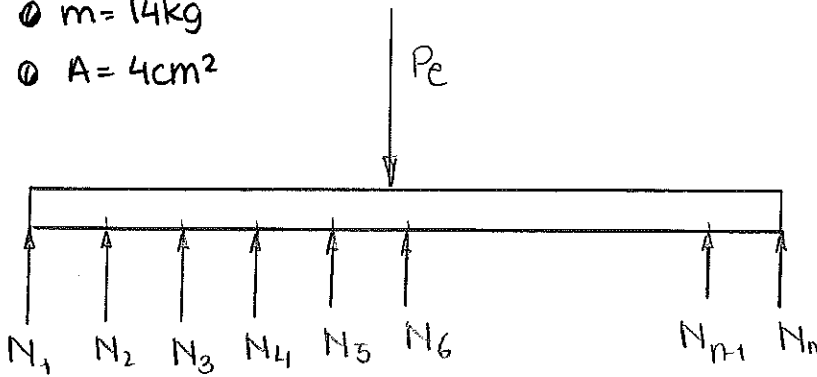
●  $A = 4\text{cm}^2$

$U_s = W_e$

$I_e = \frac{1}{12} (2\sqrt{2} \cdot \text{cm})^4 = 5,333 \cdot 10^{-8} \text{m}^4$

$I_i = \frac{1}{12} (2\text{cm})^4 = 1,333 \cdot 10^{-8} \text{m}^4$

$L_k = 0.7L = 0.7\text{m}$



$\Delta_1 = \Delta_2 = \dots = \Delta_n = \Delta$

$\frac{N_1 \cdot L}{E \cdot A_1} = \frac{N_2 \cdot L}{E \cdot A_2} = (\dots) = \frac{N_n \cdot L}{E A_n}$

$\begin{cases} N_1 = N_n = N_e \\ N_2 = N_3 = (\dots) = N_{n-1} = \frac{1}{2} N_1 = N_i = \frac{1}{2} N_e \end{cases}$

COLUMNAS EXTERIORES

$P_{cr_e} = \frac{\pi^2 \cdot E \cdot I_e}{L_k^2} = 107424,2656 \text{ N} = N_e \Rightarrow N_i = 53712,1328 \text{ N} > P_{cr_i}$

COLUMNAS INTERIORES

$P_{cr_i} = \frac{\pi^2 E \cdot I_i}{L_k^2} = 26856,0664 \text{ N} =$

Las columnas interiores fallan primero por pandeo.

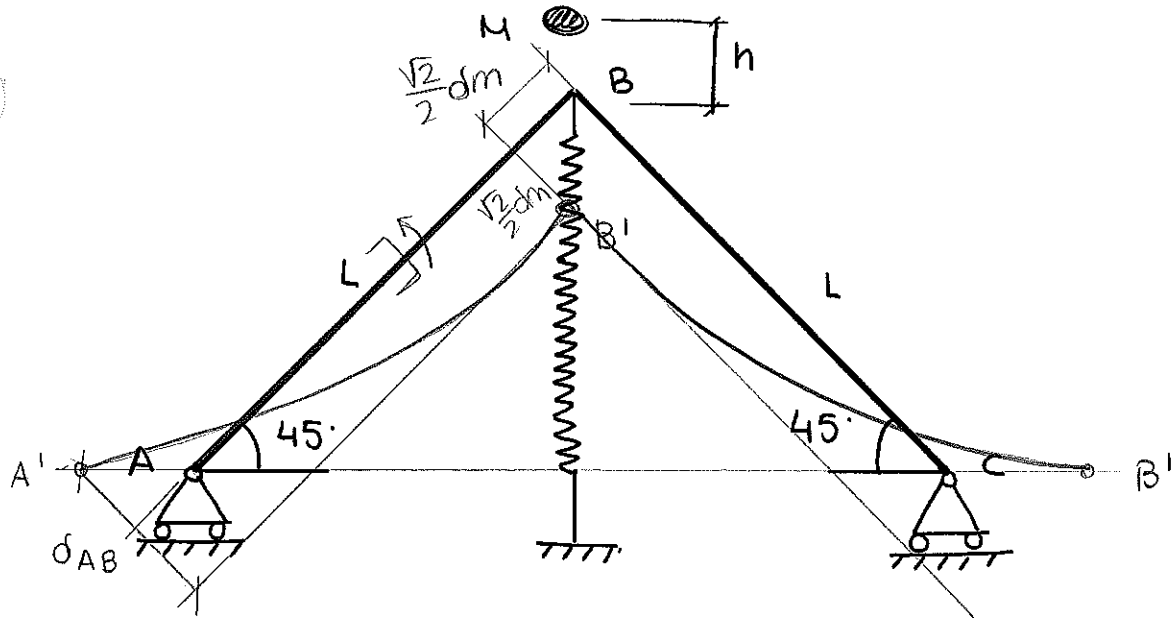
⊗ Para que la estructura se derrumbe: las columnas exteriores han de fallar.

$$N_e = 107424,2656 \text{ N.}$$

$$W_e = 2 \cdot \frac{1}{2} (107424,2656 \text{ N})^2 \frac{L}{E \cdot A_e} + (n-2) \frac{1}{2} (26856,0664 \text{ N})^2 \frac{L}{EA_i} = U_s$$

$$n = 9,293 \Rightarrow \boxed{n=10}$$

Caso



● Sección cuadrada:  $S = 10 \times 10 \text{ cm}^2$

●  $L = 1 \text{ m}$

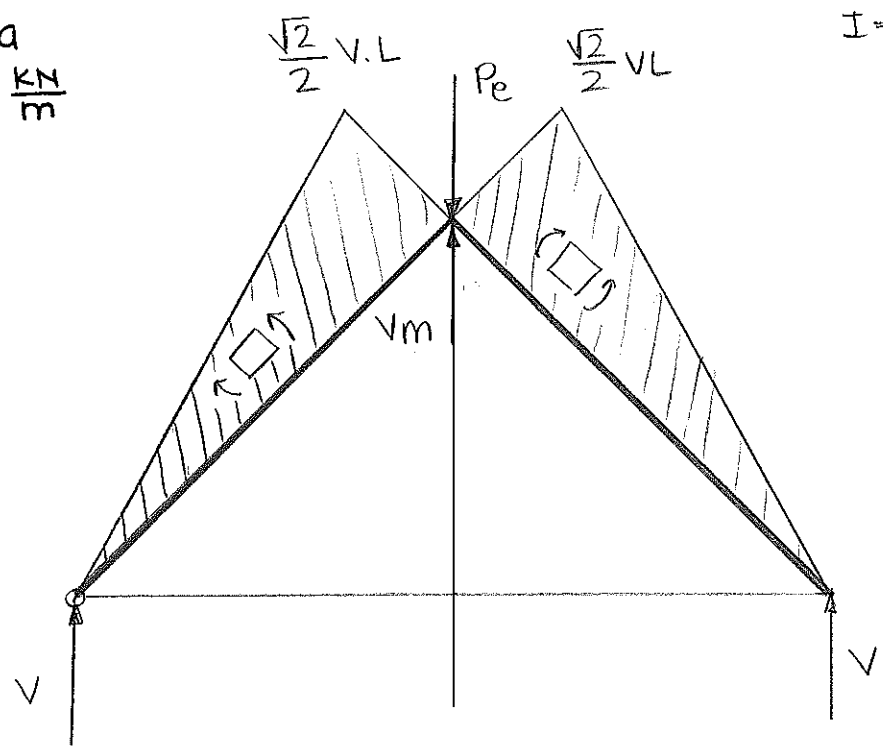
●  $E = 200 \text{ GPa}$

●  $k = 5000 \frac{\text{KN}}{\text{m}}$

●  $h = 2 \text{ m}$

●  $M = 10 \text{ kg}$

$$I = \frac{1}{12} (10 \text{ cm})^4 = 8,333 \cdot 10^6 \text{ m}^4$$



$$2V + V_m = P_e \Rightarrow \boxed{V_m = P_e - 2V}$$

Ec. compatibilidad:  $\boxed{\delta_m = \delta_B}$  :  $\frac{\sqrt{2}}{2} \delta_m \cdot 2 = \delta_{AB}$  :  $\boxed{\delta_{AB} = \sqrt{2} \delta_m}$

$$\delta_{AB} = \frac{1}{EI} \left[ \frac{\sqrt{2}}{4} V L \cdot L \cdot \frac{2}{3} L \right] = \frac{\sqrt{2} V L^3}{6EI} \quad \left. \begin{array}{l} \\ \end{array} \right\} \frac{P_e - 2V}{k} \cdot \sqrt{2} = \frac{\sqrt{2} V L^3}{6EI}$$

$$\delta_m = \frac{V_m}{k} = \frac{P_e - 2V}{k} \quad \left. \begin{array}{l} \\ \end{array} \right\} \boxed{P_e = \frac{V L^3}{6EI} \cdot k + 2V}$$

$$U_s = W_e$$

$$U_s = mgh$$

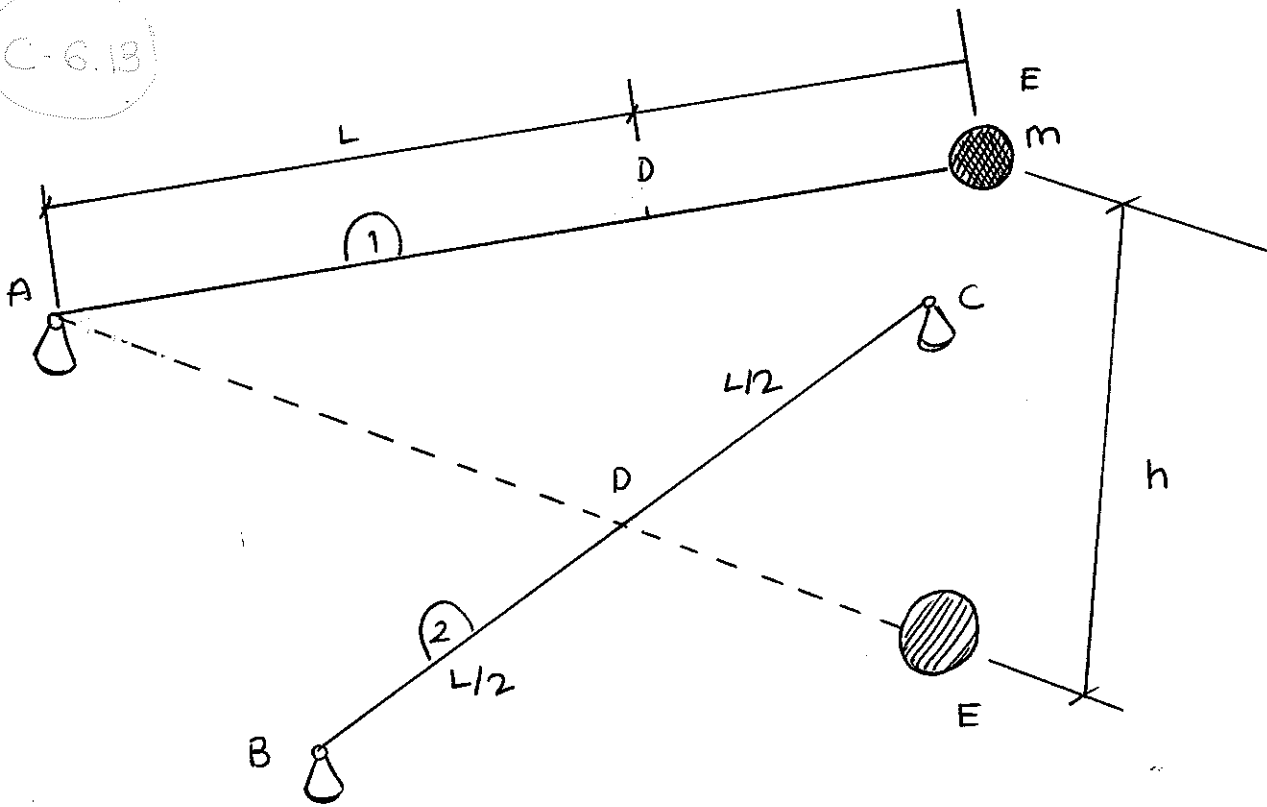
$$W_e = \frac{1}{2} P_e \cdot \delta_m = \frac{1}{2} \left( \frac{VL^3}{6EI} k + 2V \right) \cdot \frac{VL^3}{6EI}$$

$$\frac{VL^3}{12EI} \left( \frac{VL^3}{6EI} k + 2V \right) = 10 \cdot 10 \cdot 2m \Rightarrow V = 40000N$$

$$\delta_m = 4 \cdot 10^{-3} m$$

$$\delta_m = 4mm$$

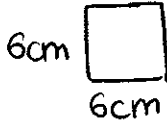
C-6.13



sección ①



sección ②



①  $L = 1m$

②  $E = 200GPa$

③  $\sigma_r = 200MPa$

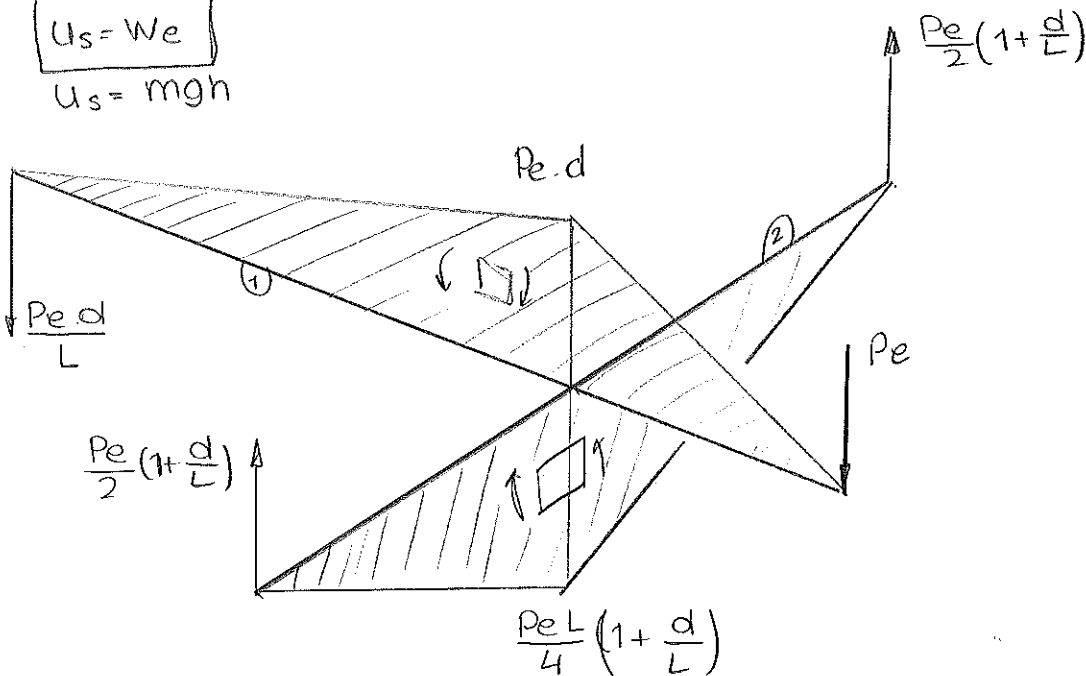
④  $m = 10kg, h = 0.5m$

$$I_1 = \frac{1}{12} 6cm (8cm)^3 = 2.56 \cdot 10^{-6} m^4$$

$$I_2 = \frac{1}{12} (6cm)^4 = 1.08 \cdot 10^{-6} m^4$$

$$U_s = W_e$$

$$U_s = mgh$$



⑤ Para que rompan simultáneamente  $\sigma_{xx, \max}^{(ade)} = \sigma_{xx, \max}^{(bde)}$

$$\sigma_{xx, \max}^{(ade)} = \frac{Pe \cdot d \cdot 4cm}{I_1} = Pe \cdot d \cdot 15625$$

$$\sigma_{xx, \max}^{(bde)} = \frac{(Pe/L)(1 + d/L) 3cm}{I_2} = 6944 \cdot PeL(1 + d/L)$$

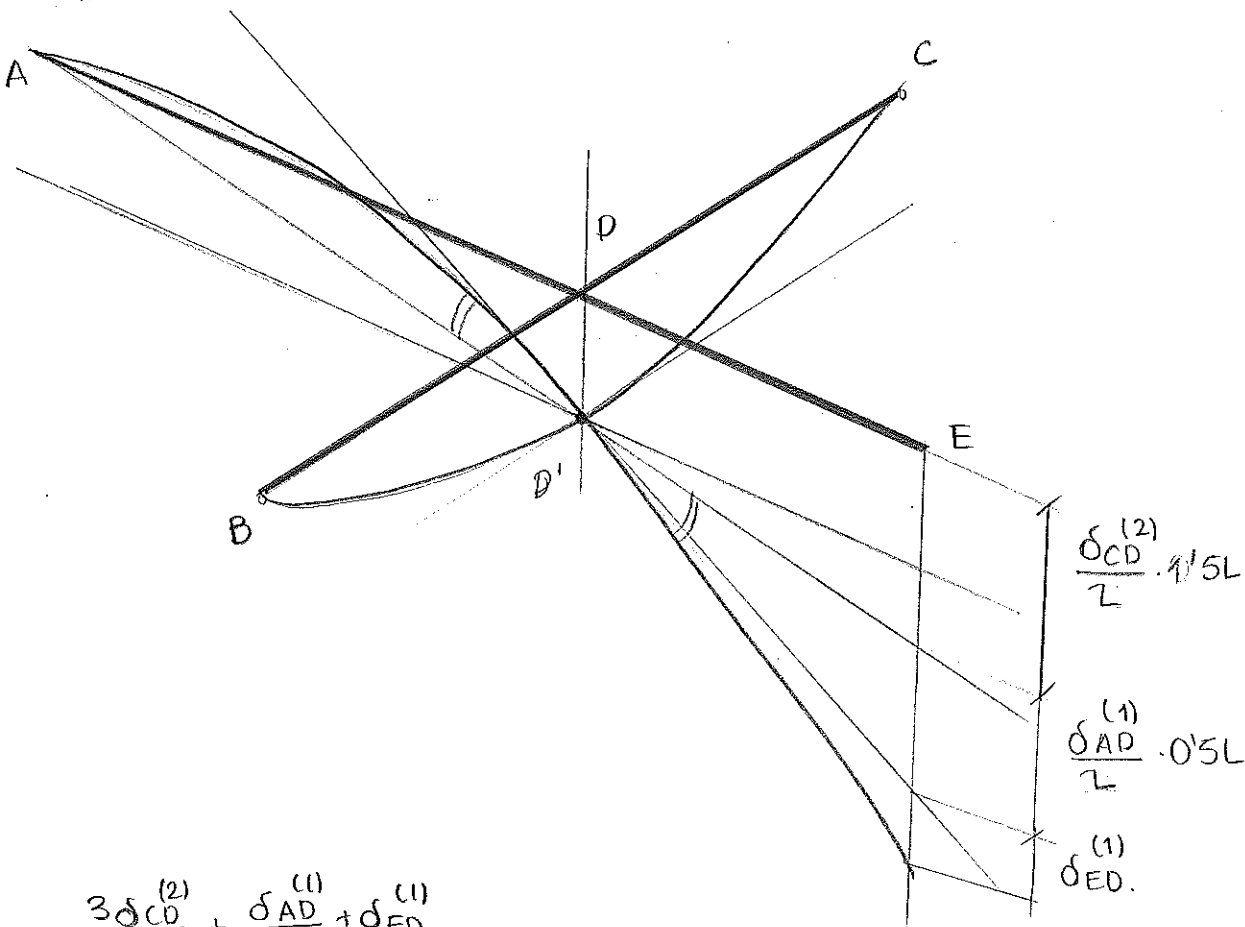
$$15625d = 6944.444 (1 + d/L) \cdot L$$

$$15625d = 6944,444 \cdot L + 6944,444 d$$

$$d = 0,8L$$

$$d = 0,5L$$

$$\left. \begin{aligned} W_e &= \frac{1}{2} P_e \cdot \delta \\ U_s &= mgh \end{aligned} \right\} U_s = W_e$$



$$Y_E = \frac{3\delta_{CD}^{(2)}}{2} + \frac{\delta_{AD}^{(1)}}{2} + \delta_{ED}^{(1)}$$

$$\delta_{CD}^{(2)} = \frac{1}{EI} \left[ \frac{1}{2} \frac{P_e \cdot L \cdot 1,5}{4} \cdot \frac{L}{2} \cdot \frac{2}{3} \frac{L}{2} \right] = \frac{P_e L^3}{32EI_2}$$

$$\delta_{AD}^{(1)} = \frac{1}{EI} \left[ \frac{1}{2} P_e \cdot 0,5L \cdot L \cdot \frac{2}{3} L \right] = \frac{P_e L^3}{6EI_1}$$

$$\delta_{ED}^{(1)} = \frac{1}{EI} \left[ \frac{1}{2} P_e \cdot 0,5L \cdot 0,5L \cdot \frac{2}{3} \cdot 0,5L \right] = \frac{P_e L^3}{24EI_1}$$

$$\delta = \frac{P_e}{E} \left( \frac{L^3}{8I_1} + \frac{3L^3}{64I_2} \right)$$

$$mgh = \frac{1}{2E} P_e^2 \left( \frac{L^3}{8I_1} + \frac{3L^3}{64I_2} \right)$$

$$P_e = 14725,72779 \text{ N}$$

$$\sigma_{xx, \max}^{(abc)} = 115,045 \text{ MPa}$$

$$\sigma_{xx, \max}^{(bdc)} = 153,398 \text{ MPa} \implies n = \frac{\sigma_f}{\sigma_{\max}} = 1'304$$

$$n = 1'304$$

