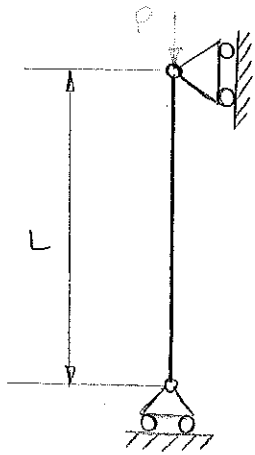


PANDEO

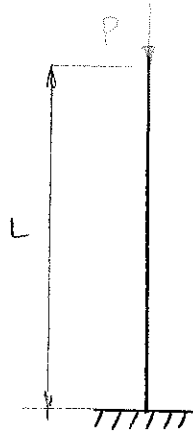
① COLUMNA DE EULER



$$P_{cr} = \frac{\pi^2 EI}{L^2}$$

$$L_k = L$$

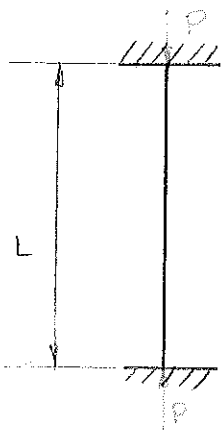
② COLUMNA EMPOTRADA Y LIBRE



$$P_{cr} = \frac{\pi^2 EI}{4L^2}$$

$$L_k = 2L$$

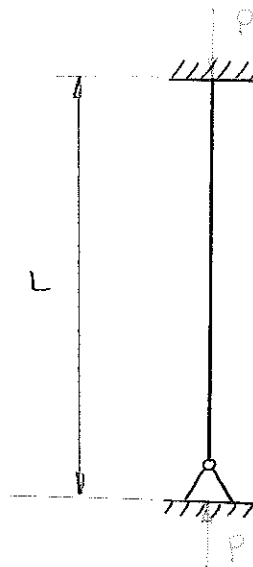
③ COLUMNA CON AMBOS EXTREMOS EMPOTRADOS



$$P_{cr} = \frac{4\pi^2 EI}{L^2}$$

$$L_k = \frac{L}{2}$$

④ COLUMNA EMPOTRADA EN UN EXTREMO Y ARTICULADA EN EL OTRO.



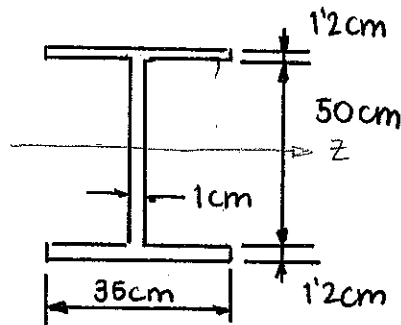
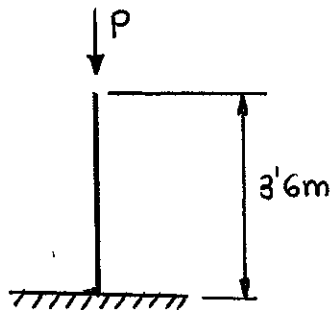
$$P_{cr} = \frac{\pi^2 EI}{(0.7L)^2}$$

$$L_k \approx 0.7L$$



4. Inestabilidad: Pandeo.

4.1



⊗ Acero S-235

⊗ $E = 210 \text{ GPa}$

⊗ $\sigma_f = 240 \text{ MPa}$

TABLA 4.2

Perfil laminado en I
 S-235
 $\frac{h}{b} = \frac{52.4 \text{ cm}}{35 \text{ cm}} = 1.4971 > 1.2$
 $t = 1.2 \text{ cm} < 40 \text{ mm}$

$$I_z = \frac{1}{12} 35 \text{ cm} (52.4 \text{ cm})^3 - 2 \cdot \frac{1}{12} 17 \text{ cm} (50 \text{ cm})^3 = 6.547699 \cdot 10^{-4} \text{ m}^4$$

$$I_y = \frac{1}{12} 50 \text{ cm} (1 \text{ cm})^3 + 2 \cdot \frac{1}{12} 1.2 \text{ cm} (35 \text{ cm})^3 = 8.579167 \cdot 10^{-5} \text{ m}^4$$

$I_y < I_z \rightarrow$ Se producirá el fenómeno de pandeo en el plano xz

⊗ La curva de pandeo es la b: $\alpha = 0.84$

$$L_k = 2L = 7.2 \text{ m}; \quad A = 2 \cdot 1.2 \text{ cm} \cdot 35 \text{ cm} + 1 \text{ cm} \cdot 50 \text{ cm} = 0.0134 \text{ m}^2$$

$$P_{cr} = \frac{\pi^2 EI}{L_k^2} = 3430039.877 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 0.968297 \rightarrow A \gamma$$

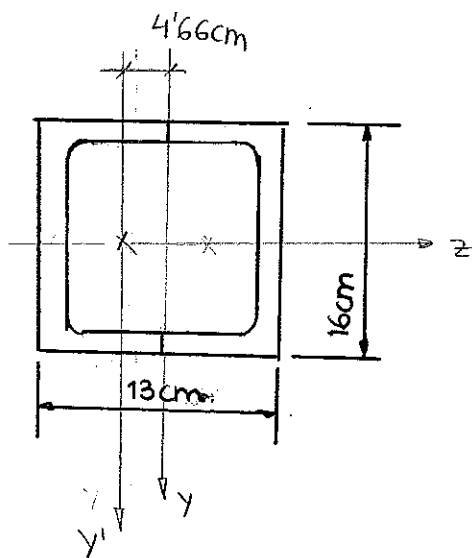
$$\Phi = 0'5 [1 + \alpha \cdot (\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 1'09941$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0'61725$$

$$P_b = \chi \cdot A \cdot \sigma_f = 1985080,652 \text{ N}$$

$$P_b = 1985,081 \text{ kN}$$

(4.2)



- ⊗ Dos perfiles UPN-160
- ⊗ Extremos articulados: $L=8\text{m}$
- ⊗ Acero S-235
- ⊗ $E=210\text{GPa}$
- ⊗ ~~190~~ $\sigma_f=240\text{MPa}$

TABLA 4.2 → Agrupación de perfiles laminados soldados → $c: \alpha = 0.49$

$$I_2 = 2.925\text{cm}^4 = 1.85 \cdot 10^{-5}\text{m}^4$$

$$\left. \begin{array}{l} I_{y_1} = 85.3\text{cm}^4 \\ A_1 = 24\text{cm}^2 \end{array} \right\} I_{y_1} = 85.3\text{cm}^4 + 24\text{cm}^2 \cdot (4.66\text{cm})^2 = 606.4744\text{cm}^4 = 6.064744 \cdot 10^{-6}\text{m}^4$$

$$I_y = 2I_{y_1} = 1.2129488 \cdot 10^{-5}\text{m}^4$$

- ⊗ $I_y < I_2$: El pandeo se produce en la sección xz

$$\left. \begin{array}{l} P_{cr} = \frac{\pi^2 EI_y}{L_k^2} \\ L_k = L = 8\text{m} \end{array} \right\} P_{cr} = 392809.0955\text{N}$$

$$\left. \begin{array}{l} \bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} \\ A = 2A_1 = 48\text{cm}^2 \end{array} \right\} \bar{\lambda} = 1.7125191 \rightarrow Ay$$

$$\Phi = 0.5 [1 + \alpha(\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 2.3369281 \rightarrow By$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0.25464$$

$$P_b = \chi \cdot A \cdot \sigma_f = 293.349\text{kN}$$

$$P_b = 293.349\text{kN}$$



4.3

● Perfil IPN

● $L = 4\text{m}$

● $P = 300\text{ kN}$

● S-235 ; $E = 210\text{ GPa}$; $\sigma_f = 240\text{ MPa}$

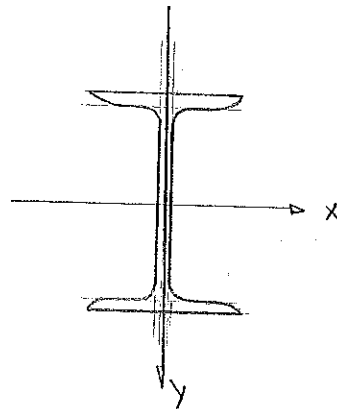
1) EXTREMOS EMPOTRADOS.

$$L_k = \frac{L}{2} = 2\text{m}$$

① Elegimos IPN-100

$$\begin{cases} I_x = 171\text{ cm}^4 \\ I_y = 12'2\text{ cm}^4 \end{cases}$$

$I_y < I_x$ El pandeo se produce en el plano xz



$$\begin{aligned} A &= 10'6\text{ cm}^2 \\ h &= 100 \\ b &= 50 \\ t &< 40\text{ mm} \end{aligned}$$

$$P_{cr} = \frac{\pi^2 E I_y}{L_k^2} = 6321481619\text{ N}$$

TABLA 4.2

$$\begin{cases} \frac{h}{b} = 2 > 1'2 \\ t < 40\text{ mm} \\ \text{eje de pandeo } y \end{cases}$$

⇒ curva b

$$\alpha = 0'34$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 2,006 \quad \rightarrow A \gamma$$

$$\Phi = 0'5 [1 + \alpha (\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 2'8192 \quad \rightarrow B \gamma$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0'2083$$

$$P_b = \chi \cdot A \cdot \sigma_f = 52999,59\text{ N} < 300\text{ kN} \text{ (No nos sirve)}$$

② Elegimos IPN-180

$$I_x = 1450\text{ cm}^4$$

$$A = 27'9\text{ cm}^2$$

$$I_y = 81'3\text{ cm}^4$$

$$h = 180\text{ cm}$$

$$b = 82\text{ cm}$$

$$t < 40\text{ mm}$$

$I_y < I_x$: El pandeo se produce en el plano xz

$$P_{cr} = \frac{\pi^2 EI}{L_k^2} = 421259,3898$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 1,26076 \rightarrow A \}$$

TABLA 4.2

$$\left\{ \begin{array}{l} \frac{h}{b} = 2,195 > 1,2 \\ t < 40 \text{ mm} \\ \text{Eje de pandeo y} \end{array} \right.$$

\Rightarrow curva a
 $\alpha = 0,34$

$$\Phi = 0,5 [1 + \alpha(\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 1,47511 \rightarrow B \}$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \lambda^2}} = 0,44626$$

$P_b = \chi \cdot \sigma_f \cdot A = 298817,012 \text{ kN} > P$. No nos sirve; el inmediatamente superior sí.

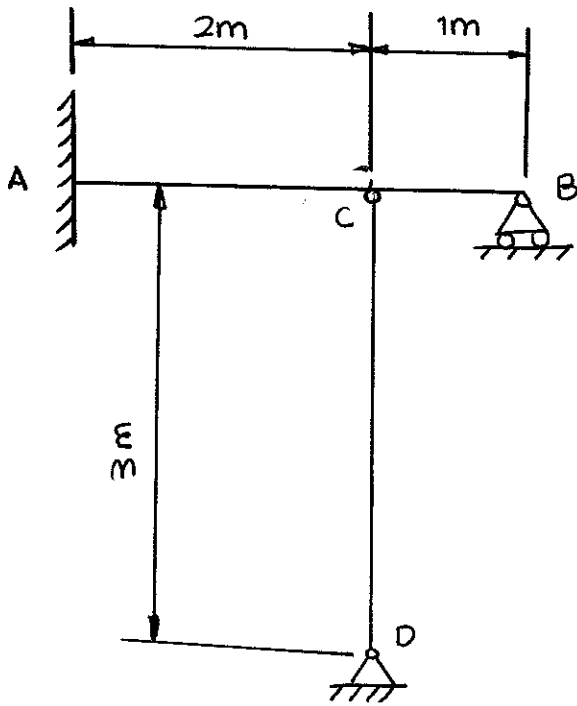
(habría que repetir el proceso y comprobar)

Necesitamos un perfil IPN-200

2) EXTREMOS ARTICULADOS

sería lo mismo pero con $L_k = L = 8 \text{ m}$.

4.4



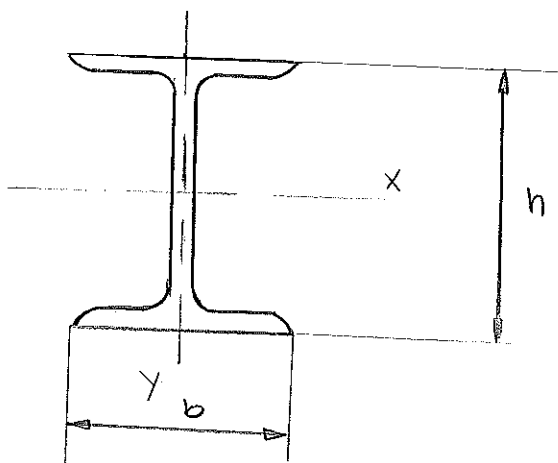
● VIGA AB: $I_z = 19610 \text{ cm}^4$

● SOPORTE CD: IPN-200

● Acero S-275; $G = 210 \text{ GPa}$; $\sigma_f = 260 \text{ MPa}$
 $\alpha = 10^{-5} \cdot \text{C}^{-1}$

$T(CD) \uparrow$

SOPORTE CD



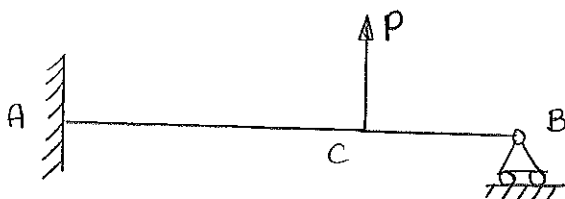
- $h = 200 \text{ cm}$
- $b = 90 \text{ cm}$
- $t < 40 \text{ mm}$
- $A = 33.5 \text{ cm}^2$
- $I_x = 2140 \text{ cm}^4$
- $I_y = 117 \text{ cm}^4$

TABLA 2.4

$\left\{ \begin{array}{l} \frac{h}{b} = 2.22 > 1.2 \\ t < 40 \text{ mm} \\ I_y < I_x: \text{ el pandeo se produce en el plano } xz: \text{ eje } y \end{array} \right.$

⇒ curva b
 $\alpha = 0.34$

①

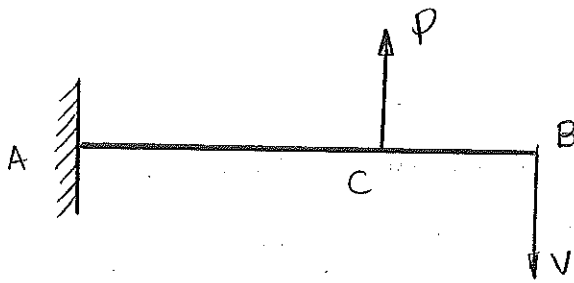


Ec compatibilidad: $y_{c1} = y_{c2}$

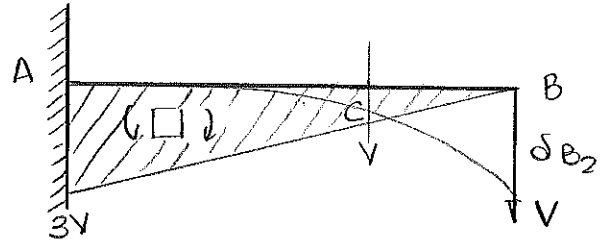
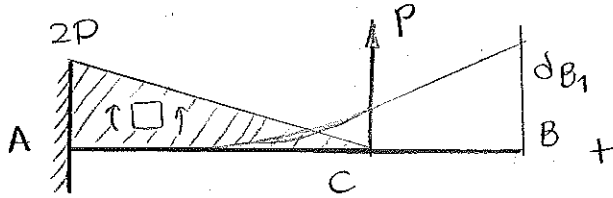
②



①



Ec. Compatibilidad: $\delta_B = 0$



$$\delta_B = \delta_{B1} - \delta_{B2} = 0$$

$$\delta_{B1} = \delta_{BA} = \frac{1}{EI} \left[\frac{1}{2} \cdot 2P \cdot \left(1 + \frac{2}{3} \cdot 2\right) \right] = \frac{14P}{3EI}$$

$$\delta_{B2} = \delta_{BA} = \frac{1}{EI} \left[\frac{1}{2} \cdot 3 \cdot 3V \cdot \frac{2}{3} \cdot 2 \right] = \frac{9V}{EI}$$

$$\left. \begin{array}{l} \frac{14P}{3EI} = \frac{9V}{EI} \\ V = \frac{14}{27} P \end{array} \right\}$$

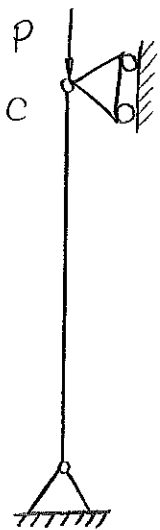
$$\downarrow \gamma_{C1} = \delta_{C2} - \delta_{C1}$$

$$\delta_{C1} = \frac{1}{EI} \left[\frac{1}{2} \cdot 2 \cdot 2P \cdot \frac{2}{3} \cdot 2 \right] = \frac{8P}{3EI}$$

$$\delta_{C2} = \frac{1}{EI} \left[\frac{1}{2} \cdot 2 \cdot 2V \cdot \frac{2}{3} \cdot 2 + 2 \cdot V \cdot 1 \right] = \frac{14V}{3EI} = \frac{14 \cdot 14P}{3 \cdot 27EI} = \frac{196P}{81EI}$$

$$\downarrow \gamma_{C1} = -\frac{20P}{81EI}$$

②



$$L_k = L = 3m$$

$$P_{cr} = \frac{\pi^2 EI}{L_k^2} = 269440,2001 N$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 1,79795 \rightarrow A4$$

$$\Phi = 0,5 \left[1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2 \right] = 2,38797 \rightarrow B4$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0,252556$$

$$P_b = \chi \cdot \sigma_f \cdot A = 219976,6906 N$$

$$\boxed{\delta_{c2} = \frac{P.L}{AE} - \alpha \Delta T . L = \downarrow \gamma_{c2}}$$

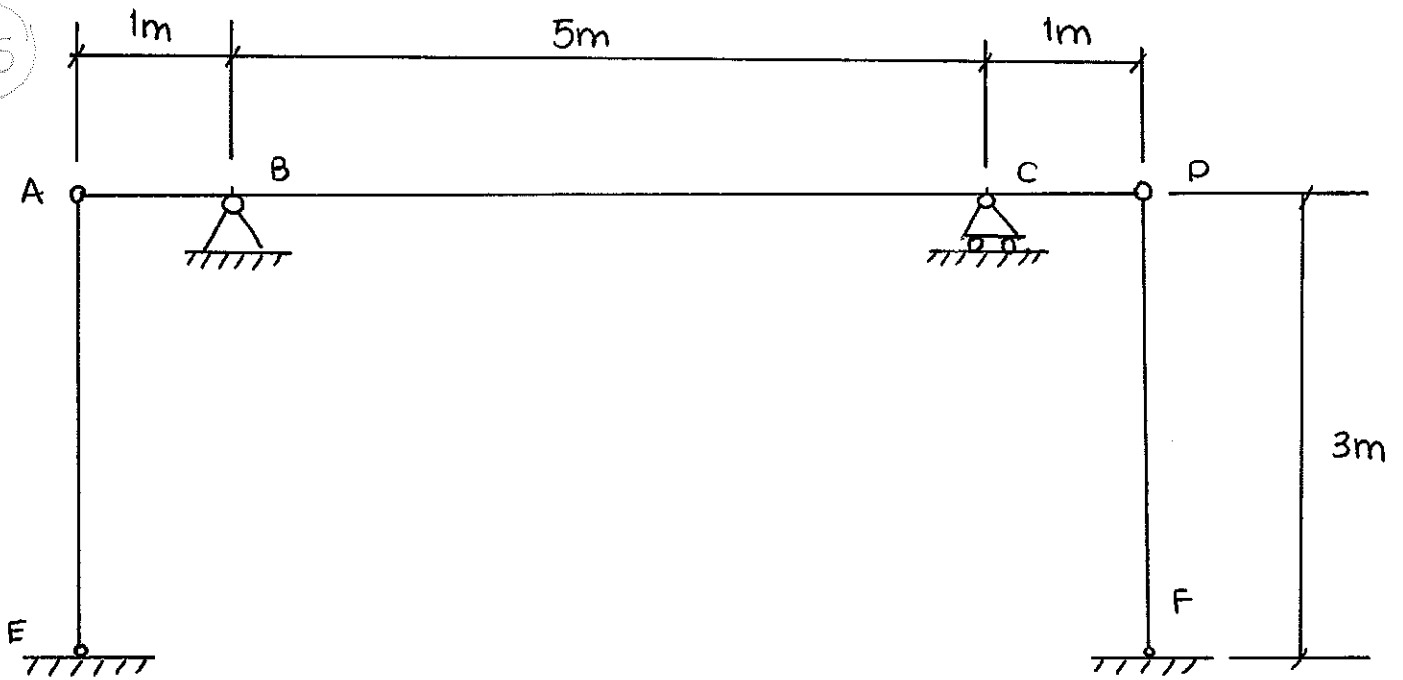
$$- \frac{20P}{81EI} = \frac{PL}{AE} - \alpha \Delta T . L$$

$$\frac{20 \cdot 219976,6906}{81 \cdot 210 \cdot 10^9 \cdot 19610 \cdot \frac{1}{100^4}} = \frac{219976,6906 \cdot 3m}{33'5 \cdot \frac{1}{100^2} \cdot 210 \cdot 10^9} - 10^{-5} \Delta T \cdot 3$$

$$\boxed{\Delta T = 75'234'C}$$



45



① $I(ABCD) = 19610 \text{ cm}^4$

② Bamas AE y DF: sección tubular $\begin{cases} \phi_i = 60 \text{ mm} \\ \phi_e = 70 \text{ mm} \end{cases}$

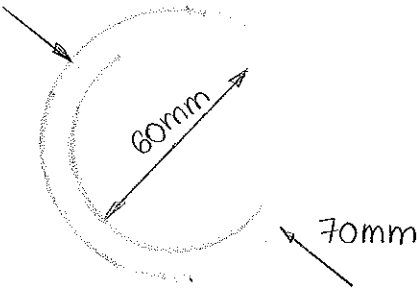
③ Acero S-355

④ $E = 210 \text{ GPa}, \sigma_f = 360 \text{ MPa}, \alpha = 10^{-5} \text{ C}^{-1}$

BARRAS AE y CF

① suponemos conformado en frío \rightarrow

curva C
 $\alpha = 0.49$

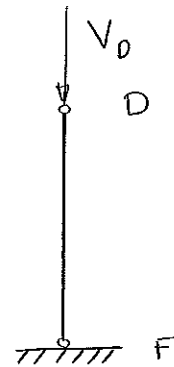


$$I = \frac{\pi (35 \text{ mm})^4 - \pi (30 \text{ mm})^4}{4} = 5.424156 \cdot 10^{-7} \text{ m}^4$$

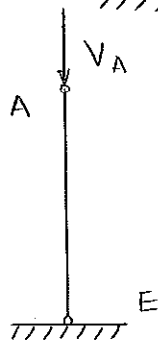
$$A = \pi (35 \text{ mm})^2 - \pi (30 \text{ mm})^2 = 1.021017612 \cdot 10^{-3} \text{ m}^2$$



②



③

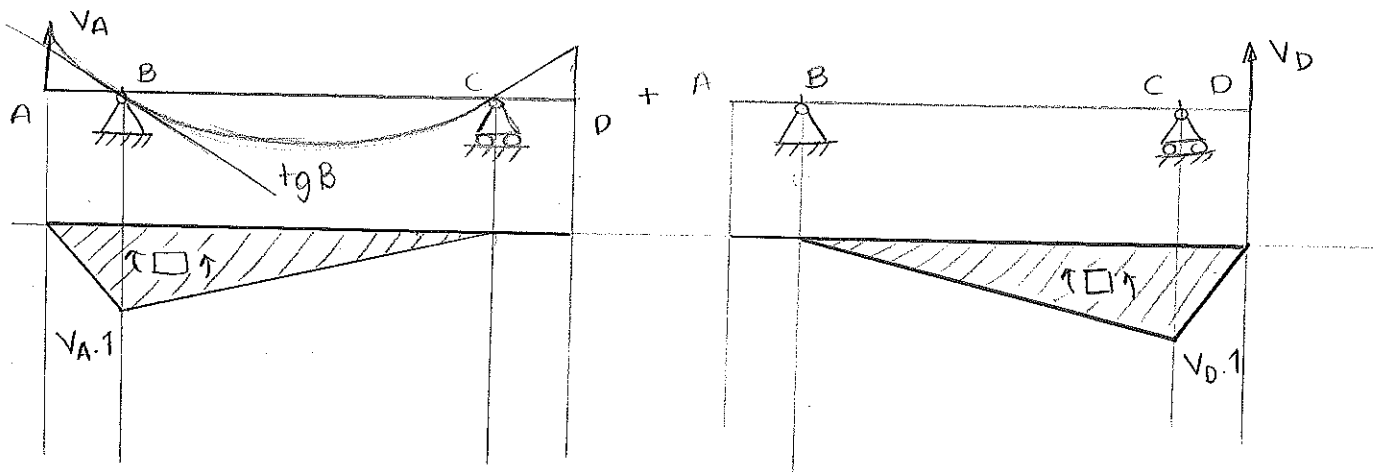


EC compatibilidad: (1) $\uparrow \gamma_{A1} = \uparrow \gamma_{A3}$
(2) $\uparrow \gamma_{D1} = \uparrow \gamma_{D2}$

1



Superposición



EQUILIBRIO: $V_A = R_{BA} + R_{BC} \Rightarrow R_{BA} = \frac{4}{5} V_A$

$V_A \cdot 1m = R_{BC} \cdot 5m \Rightarrow R_{BC} = \frac{1}{5} V_A$

EQUILIBRIO: $V_D = R_{BD} + R_{CD} \Rightarrow R_{CD} = \frac{4}{5} V_D$

$V_D \cdot 1m = R_{BD} \cdot 5m \Rightarrow R_{BD} = \frac{1}{5} V_D$

$\uparrow Y_{A1} = \delta_{AB} + \theta_B \cdot 1m$

$\theta_B = \frac{\delta_{CB}}{5m}$

$\delta_{CB} = \frac{1}{EI} \left[\frac{1}{2} V_A \cdot 5m \cdot \frac{2}{3} 5m \right] = \frac{25V_A}{3EI}$

$\theta_B = \frac{5V_A}{3EI}$

$\delta_{AB} = \frac{1}{EI} \left[\frac{1}{2} V_A \cdot 1m \cdot \frac{2}{3} 1m \right] = \frac{V_A}{3EI}$

$\uparrow Y_{A1M} = \frac{2V_A}{EI}$

$\uparrow Y_{D1} = \theta_C \cdot 1m$

$\theta_C = \frac{\delta_{BC}}{5m}$

$\delta_{BC} = \frac{1}{EI} \left[\frac{1}{2} V_D \cdot 5m \cdot \frac{1}{3} 5m \right] = \frac{25V_D}{6EI}$

$\theta_C = \frac{5V_D}{6EI}$

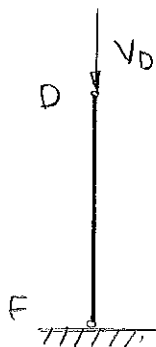
$\uparrow Y_{D1M} = \frac{5V_D}{6EI}$

Por simetría:

$$\begin{aligned} \uparrow Y_{D21} &= \frac{2V_D}{EI} \\ \uparrow Y_{A21} &= \frac{5V_D}{6EI} \end{aligned}$$

$$\begin{aligned} \uparrow Y_{D1} &= \frac{5V_A}{6EI} + \frac{2V_D}{EI} = \frac{1}{EI} \left(\frac{5}{6}V_A + 2V_D \right) \\ \uparrow Y_{A1} &= \frac{2V_A}{EI} + \frac{5V_D}{6EI} = \frac{1}{EI} \left(2V_A + \frac{5}{6}V_D \right) \end{aligned}$$

2)



$$L_k = L = 3\text{m}$$

$$P_{cr} = \frac{\pi^2 E \cdot I_{DF}}{L_k^2} = \frac{\pi^2 \cdot 210 \text{ GPa} \cdot 5'424156 \cdot 10^{-7} \text{ m}^4}{(3\text{m})^2} = 124,9133058 \text{ kN}$$

$$\bar{\lambda} = \sqrt{\frac{A \cdot \sigma_f}{P_{cr}}} = 1'715392537 \text{ } \lambda \rightarrow A \lambda$$

$$\phi = 0'5 \left[1 + \alpha(\bar{\lambda} - 0'2) + \bar{\lambda}^2 \right] = 2'3425569491 \rightarrow B \lambda$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0'2539448405$$

$$P_b = \chi \cdot \sigma_f \cdot A = 933,4157567 \text{ kN} \text{ (Para ambas barras)}$$

$$\uparrow Y_{D2} = + \frac{V_D \cdot 3\text{m}}{AE} + \alpha \cdot \Delta T \cdot 3\text{m}$$

3)

$$\uparrow Y_{A2} = - \frac{V_A \cdot 3\text{m}}{AE} + \alpha \cdot \Delta T \cdot 3\text{m}$$

Ec. compatibilidad

$$\frac{1}{EI} \left[2V_A + \frac{5}{6}V_D \right] = - \frac{V_A \cdot 3\text{m}}{AE} + 3\alpha \Delta T$$

por simetría $V_A = V_D$

Pandeo cuando $V_A = P_b$

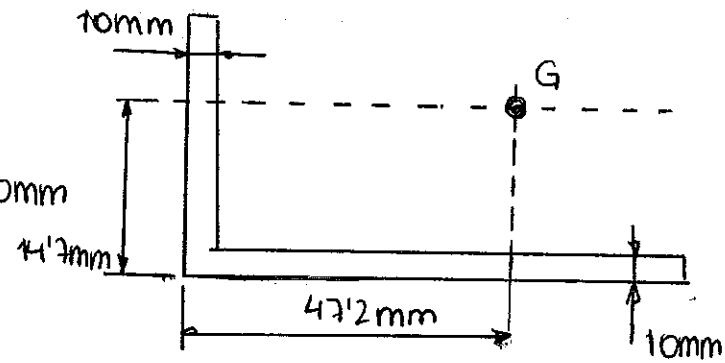
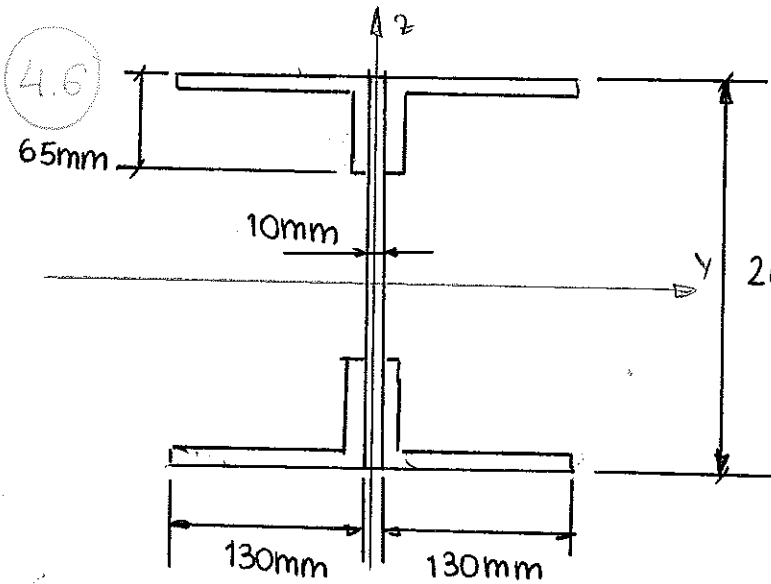
$$\Delta T = 257'608 \text{ } ^\circ\text{C}$$

Si lo haces suponiendo que esta laminado en caliente:

curva a $\Rightarrow \alpha = 0'21$

$$\Delta T = 298'904 \text{ } ^\circ\text{C}$$





● Acero S-235

● $E = 210 \text{ GPa}$, $\sigma_f = 240 \text{ MPa}$.

● $L = 3.90 \text{ m}$ (viga articulada en los extremos)

$$L_k = L = 3.90 \text{ m}$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I}{L_k^2} = 4505432,231 \text{ N}$$

$$I_y = \frac{1}{12} \left[(130 \text{ mm} \cdot 2 + 10 \text{ mm}) (260 \text{ mm})^3 \right] - 2 \cdot \frac{1}{12} 120 \text{ mm} (260 \text{ mm} - 2 \cdot 10 \text{ mm})^3$$

$$- 2 \cdot \frac{1}{12} [10 \text{ mm} \cdot (260 \text{ mm} - 65 \text{ mm} \cdot 2)^3] = 1.153183 \cdot 10^{-4} \text{ m}^4$$

$$I_z = 2 \cdot \frac{1}{12} (10 \text{ mm}) \cdot (130 \text{ mm} \cdot 2 + 10 \text{ mm})^3 + \frac{1}{12} (65 \text{ mm} - 10 \text{ mm}) \cdot (3 \cdot 10 \text{ mm})^3 +$$

$$+ \frac{1}{12} (260 \text{ mm} - 2 \cdot 65 \text{ mm}) (10 \text{ mm})^3 = 3.3063 \cdot 10^{-5} \text{ m}^4$$

$I_z < I_y$: El pandeo se produce en el plano xy : eie z } curva c $\Rightarrow \alpha = 0.49$
 Agrupación de perfiles laminados soldados.

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 0.7298563486 \rightarrow A_4$$

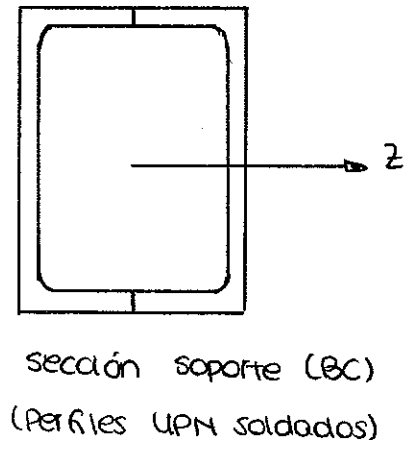
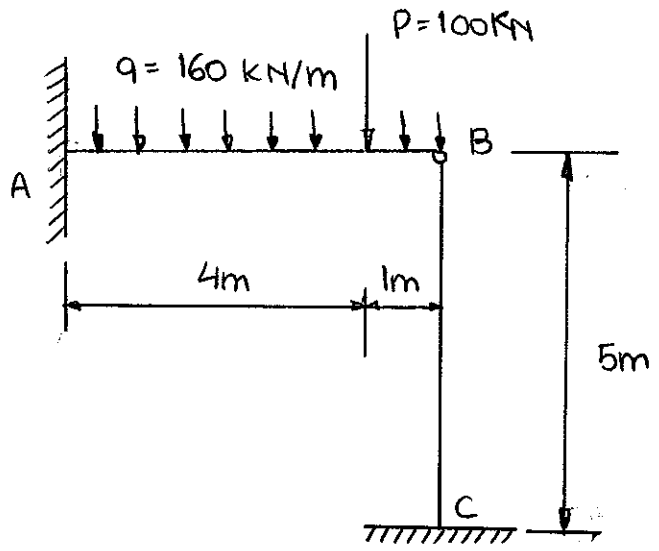
$$A = 4 [5 \text{ mm} \cdot 130 \text{ mm} + 10 \text{ mm} \cdot 65 \text{ mm} + 120 \text{ mm} \cdot 10 \text{ mm}] = 0.01 \text{ m}^2$$

$$\phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 0.8564207241 \rightarrow B_4$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.7665830494$$

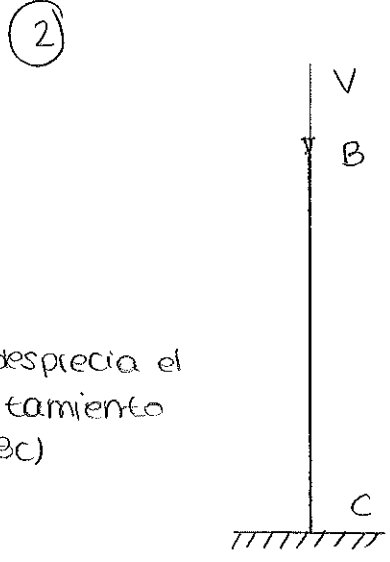
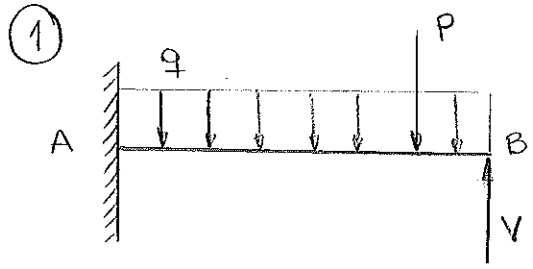
$$P_b = \gamma \cdot \sigma_f \cdot A = 1694,709 \text{ KN}$$

4.7

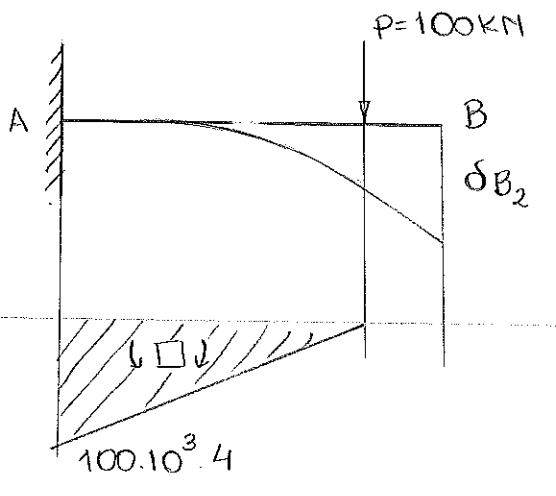
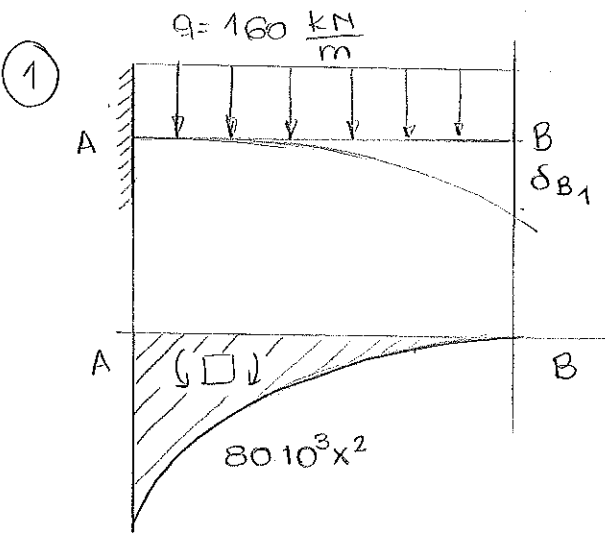


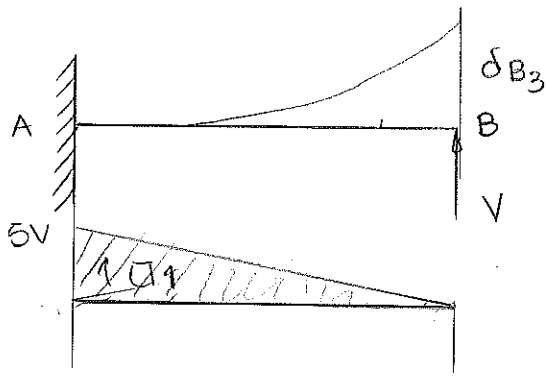
Acero S-235

$E = 210 \text{ GPa}$ $\sigma_f = 240 \text{ MPa}$



Ec. compatibilidad $\downarrow y_{B1} = \downarrow y_{B2} = 0$ (se desprecia el acortamiento en BC)





$$\downarrow \gamma_{B1} = \delta_{B1} + \delta_{B2} - \delta_{B3}$$

$$\delta_{B1} = \frac{1}{EI} \int_0^L 80 \cdot 10^3 x^2 dx = \frac{1}{EI} \cdot 125 \cdot 10^5$$

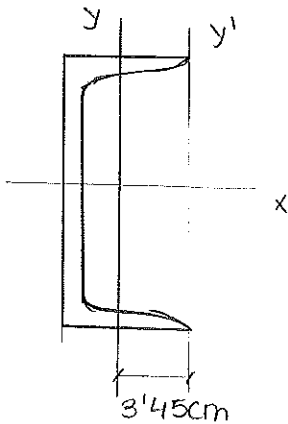
$$\delta_{B2} = \frac{1}{EI} \left[\frac{1}{2} \cdot 400 \cdot 10^3 \cdot 4 \cdot \left(1 + \frac{2}{3} \cdot 4\right) \right] = \frac{88 \cdot 10^5}{3EI}$$

$$\delta_{B3} = \frac{1}{EI} \left[\frac{1}{2} \cdot V \cdot 5 \cdot \frac{2}{3} \cdot 5 \right] = \frac{125V}{3EI}$$

$$\downarrow \gamma_{B1} = \frac{463 \cdot 10^5 + 125V}{3EI} = 0 \Rightarrow \boxed{V = 26997,84017N}$$

② $L_k = L = 5m$

Suponemos UPN-100.



$$\begin{cases} I_x = 206 \text{ cm}^4 \\ I_y = 29'3 \text{ cm}^4 \\ A = 13'5 \text{ cm}^2 \rightarrow A_1 = 27 \text{ cm}^2 \end{cases}$$

$$I_{y'} = I_y + A(3'45 \text{ cm})^2 = 189'98375 \text{ cm}^4$$

$$I_{x, \text{TOTAL}} = 412 \text{ cm}^4$$

$$I_{y', \text{TOTAL}} = 379'9675 \text{ cm}^4$$

$I_x > I_{y'} \Rightarrow$ El pandeo se produce en el plano yz \rightarrow eje y'

$$\downarrow \text{ curva } c \Rightarrow \boxed{\alpha = 0'49}$$

$$P_{cr} = \frac{\pi^2 EI_{y'}}{L_k^2} = 315010,8285N$$

$$\bar{\lambda} = \sqrt{\frac{A \cdot \sigma_f}{P_{cr}}} = 1'434249679 \lambda \rightarrow A_4$$

$$\phi = 0'5 [1 + \alpha(\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 1'830927243 \lambda \rightarrow B_4$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.3368131028$$

$$P_b = \chi \cdot A \cdot \sigma_f = 218254,8906 \text{ N} < V \rightarrow \text{No nos sirve}$$

⊗ Suponemos UPN-120.

$$I_x = 364 \text{ cm}^4$$

$$I_y = 43.2 \text{ cm}^4$$

$$A = 17 \text{ cm}^2 \implies A_T = 34 \text{ cm}^2$$

$$I_{y'} = I_y + A \cdot (5.5 \text{ cm} - 1.6 \text{ cm})^2 = 301.77 \text{ cm}^4$$

$$\left. \begin{array}{l} I_{x, \text{TOTAL}} = 728 \text{ cm}^4 \\ I_{y', \text{TOTAL}} = 603.54 \text{ cm}^4 \end{array} \right\} I_x > I_{y'} \Rightarrow \text{El pandeo se produce en el plano } y'z' \Rightarrow \text{eje } z'$$

curva c \Rightarrow $\boxed{\alpha = 0.49}$

$$P_{cr} = \frac{\pi^2 E I_{y'}}{L_k^2} = 500362,8874 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 1.277031218 \text{ } \leftarrow A_f$$

$$\phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 1.579281581 \text{ } \leftarrow B_f$$

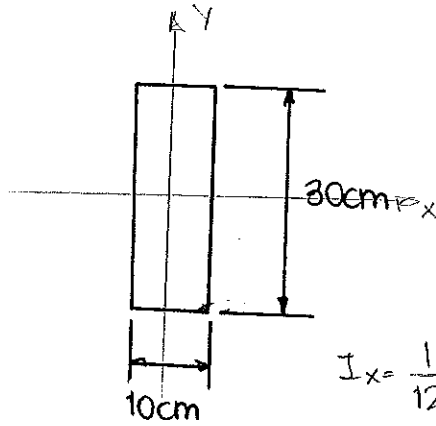
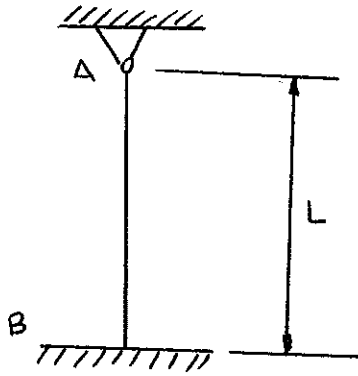
$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.3986559782$$

$$P_b = \chi \cdot A \cdot \sigma_f = 325303,2782 \text{ N} < V \Rightarrow \text{Nos sirve.}$$

El perfil más adecuado es el UPN-120.



4.8



● Acero S-355

● $E = 210 \text{ GPa}$

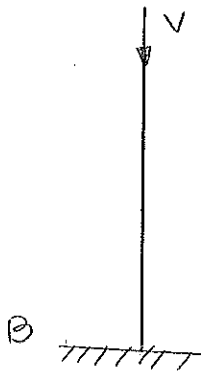
● $\sigma_f = 300 \text{ MPa}$

$\alpha = 10^{-5} \text{ C}^{-1}$

$$I_x = \frac{1}{12} 10 \text{ cm} (30 \text{ cm})^3 = 2.25 \cdot 10^{-4} \text{ m}^4$$

$$I_y = \frac{1}{12} 30 \text{ cm} (10 \text{ cm})^3 = 2.5 \cdot 10^{-5} \text{ m}^4$$

1) PLASIFICACIÓN



● Ec. compatibilidad $\delta_{AB} = 0$.

$$\delta = -\frac{V \cdot L}{EA} + \Delta T \cdot \alpha \cdot L = 0$$

$$\sigma_f = 300 \text{ MPa} \Rightarrow V_{\text{máx}} = \sigma_f \cdot A = 9 \text{ MN}$$

$$-\frac{\sigma_f \cdot L}{E} + \Delta T \cdot \alpha \cdot L = 0 \Rightarrow \Delta T = 142,857^\circ \text{C}$$

2) PANDEO.

$$L_k = 0,7L$$

$$P_{cr} = \frac{\pi^2 \cdot EI}{(0,7L)^2} = \frac{1057457614}{L^2}$$

$I_y < I_x \Rightarrow$ el pandeo se produce en el eje y.

● Perfil simple \Rightarrow curva c: $\alpha = 0,49$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 0,2917358296 \cdot L = A \cdot L$$

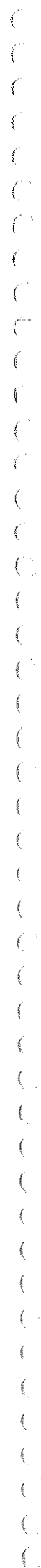
$$-\frac{V_k}{EA} + \Delta T \cdot \alpha \cdot k = 0 \Rightarrow V = \Delta T \cdot \alpha \cdot E = V = \chi \cdot A \cdot \sigma_f$$

$$\Delta T = \frac{\chi \cdot A \cdot \sigma_f}{\alpha \cdot E}$$

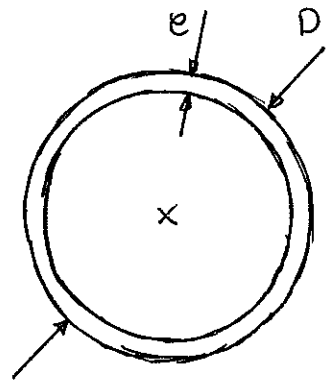
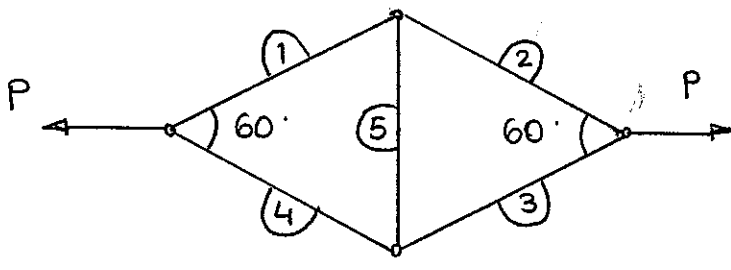
$$\phi = 0,5(1 + \alpha(\bar{\lambda} - 0,2) + \bar{\lambda}^2) = 0,5(1 + 0,49 \cdot (A \cdot L - 0,2) + A^2 L^2)$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}}$$

17

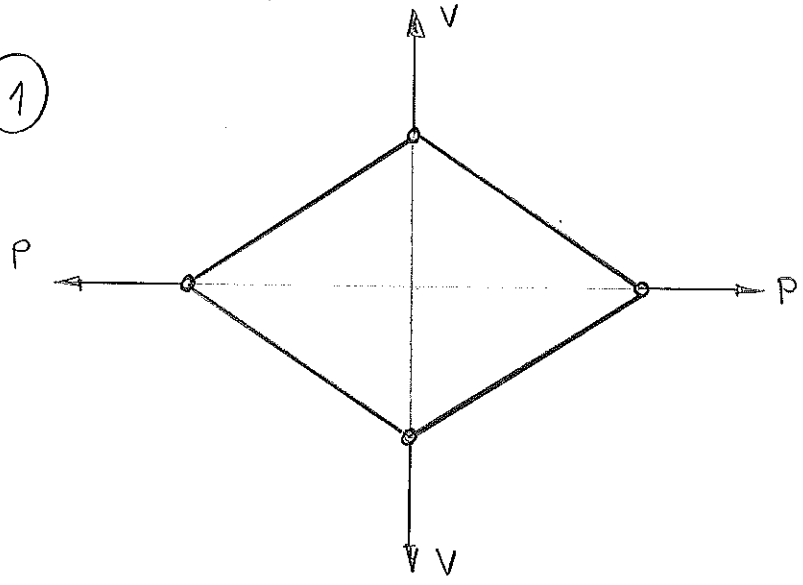


4.9



- $L_1 = L_2 = L_3 = L_4 = L_5 = L = 2\text{m}$
- Acero S-275
- $E = 210\text{GPa}$
- $\sigma_f = 260\text{MPa}$
- $e \leq 5\text{mm}$
- $P = 260\sqrt{3}\text{ kN}$.

1

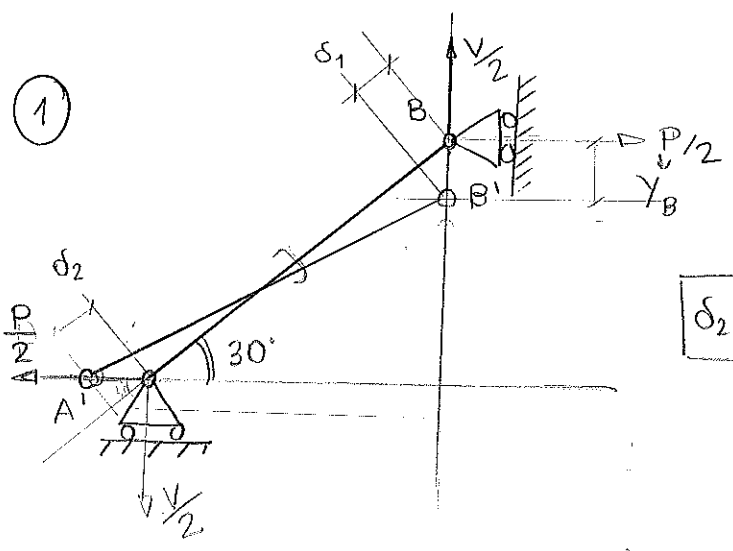


2



● Ec. compatibilidad $L'_{51} = L'_{52}$ ($\Delta L'_{51} = \Delta L'_{52}$)

1

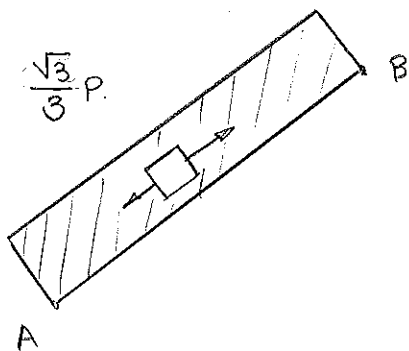


EQUILIBRIO: $\frac{P}{2} \frac{L}{2} = \frac{V \cdot L}{2} \cdot \cos 30$
 $\frac{P}{2} \frac{1}{2} = \frac{V}{2} \frac{\sqrt{3}}{2} \Rightarrow V = \frac{P}{\sqrt{3}}$

$\delta_2 - \delta_1 = \Delta L_1$

Diagrama de esfuerzos

Esfuerzos normales



$$N = \frac{P}{2} \cos 30 + \frac{V}{2} \cdot \cos 60 = \frac{\sqrt{3} P}{4} + \frac{P}{2\sqrt{3}} \cdot \frac{1}{2} = \frac{\sqrt{3}}{3} P$$

$$\frac{\sqrt{3}}{3} P = \sigma_f \cdot A_1$$

$$A_1 = 10^{-3} \text{ m}^2$$

$$\Rightarrow \text{TABLAS: } \begin{cases} D = 90 \text{ mm} \\ e = 4 \text{ mm} \end{cases}$$

La barra no trabaja a flexión

$$\Delta L_1 = \frac{N \cdot L_{AB}}{E \cdot A} = \sigma_2 \cdot \frac{Y_{B1}}{\cos 30} \Rightarrow Y_{B1} = \frac{\sigma_2 - \Delta L_1}{\cos 30} = \frac{\sigma_2 - \frac{N \cdot L}{EA}}{\cos 30} \Rightarrow \Delta L_{S1} = 2 \cdot \frac{\sigma_2 - \frac{N \cdot L}{EA}}{\cos 30}$$

2



$$\Delta L_{S2} = \frac{\frac{2D}{\sqrt{3}} \cdot L}{EA_2} = \frac{2PL}{\sqrt{3}EA_2}$$

suponemos laminado en caliente: curva a: $\alpha = 0.21$

$$L_k = 2 \text{ m}$$

suponemos $D = 100^{\text{mm}}$, $e = 4 \text{ mm}$, $A = 12.06 \text{ cm}^2$

$$I = 139 \text{ cm}^4$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I}{L_k^2} = 720234.3812 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \cdot \sigma_f}{P_{cr}}} = 0.6598168519 \rightarrow A$$

$$\phi = 0.15 [1 + \alpha(\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 0.7659599085 \rightarrow B$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.8658161746$$

$P_b = \chi \cdot \sigma_f \cdot A = 271485.3197 \text{ N}$ } Nos sirve, probamos con otra de menor resistencia
 $V = 260000 \text{ N}$

suponemos $D=94\text{ mm}$, $e=4\text{ mm}$, $A=11'30\text{ cm}^2$

$$I = 114\text{ cm}^4$$

$$P_{cr} = \frac{\pi^2 E I}{L_k^2} = 590695,8234\text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 0'7052513817 \quad \leftarrow A_f$$

$$\Phi = 0'5 [1 + \alpha(\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 0'8019411507 \quad \leftarrow B_f$$

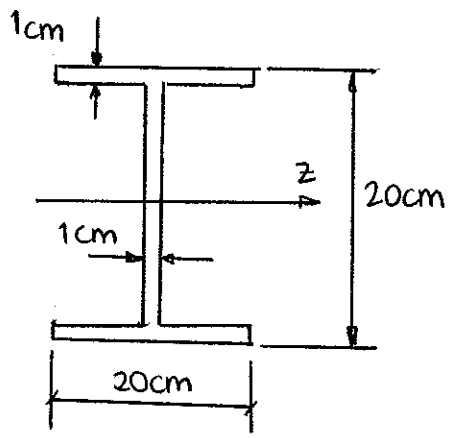
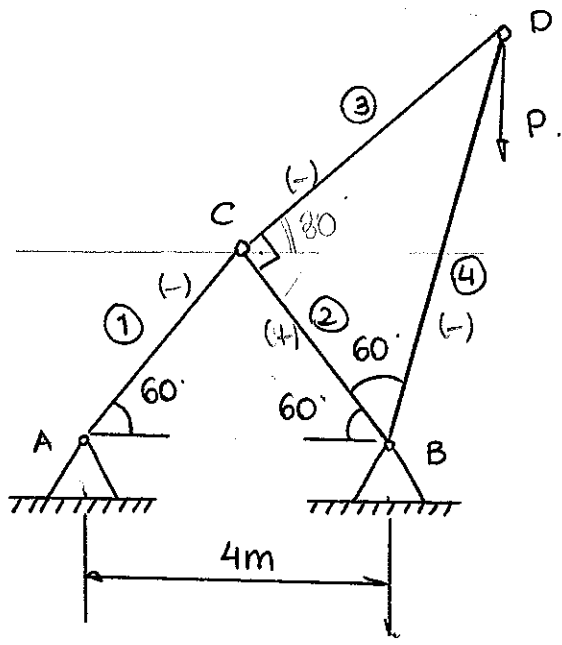
$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0'8452607049$$

$$P_b = \chi \cdot \sigma_f \cdot A = 248337,5951\text{ N} < V: \text{No nos sirve.}$$

Las bamas 1, 2, 3 y 4: $D=90\text{ mm}$ $e=4\text{ mm}$	y la barra 5: $D=100\text{ mm}$ $e=4\text{ mm}$
--	--



4.10

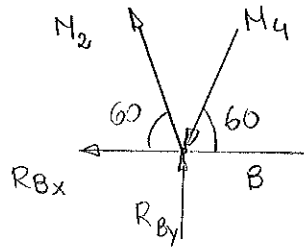
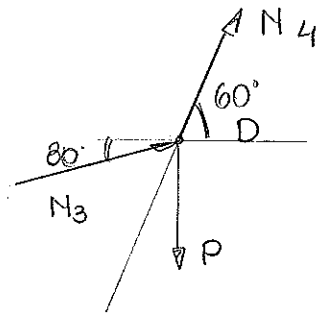
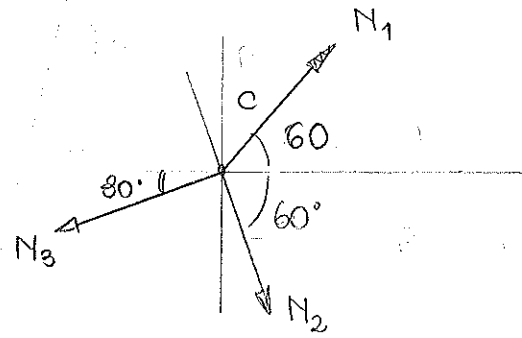
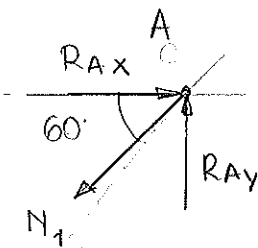


● Acero S-235

● $E = 210 \text{ GPa}$

● $\sigma_f = 240 \text{ MPa}$

①



(A) $N_1 \cdot \cos 60 = R_{Ax}$
 $N_1 \cdot \sin 60 = R_{Ay}$

(B) $N_2 \cdot \cos 60 + N_4 \cdot \cos 60 = R_{Bx}$
 $N_2 \cdot \sin 60 - N_4 \cdot \sin 60 + R_{By} = 0$

$$N_1 = -\frac{2\sqrt{3}P}{3}$$

$$N_2 = -\frac{P}{\sqrt{3}}$$

(C) $N_1 \cos 60 + N_2 \cos 60 = N_3 \cos 30 \Rightarrow N_1 \cos 60 + N_2 \cos 60 = -P \cos 30$
 $N_1 \sin 60 = N_2 \sin 60 + N_3 \sin 30 \Rightarrow N_1 \sin 60 - N_2 \sin 60 = -P \sin 30$

(D) $N_4 \cos 60 + N_3 \cos 30 = 0 \Rightarrow N_4 = -N_3 \frac{\cos 30}{\cos 60} = -\sqrt{3} \cdot N_3 \Rightarrow N_4 = \sqrt{3} P$
 $N_4 \sin 60 + N_3 \sin 30 = P$

$$-\sqrt{3} \cdot N_3 \sin 60 + N_3 \sin 30 = P$$

$$-N_3 = P \Rightarrow N_3 = -P$$

⊗ Barras

$$(1) N_1 = \sigma_f \cdot A$$

$$\frac{2\sqrt{3}}{3} P = \sigma_f \cdot A \Rightarrow P = 1205,507 \text{ kN}$$

$$A = 2.1 \text{ cm} \cdot 20 \text{ cm} + 1 \text{ cm} \cdot 18 \text{ cm} = 5.8 \cdot 10^{-3} \text{ m}^2$$

$$(2) N_2 = \sigma_f A$$

$$\frac{P}{\sqrt{3}} = \sigma_f A \Rightarrow P = 2411,015 \text{ kN}$$

$$I_2 = 4,0993 \cdot 10^{-5} \text{ m}^4$$

$$I_y = 1,33483 \cdot 10^{-5} \text{ m}^4$$

$$I_y < I_2 \Rightarrow \text{Eje } z$$

$$(3) N_3 = \sigma_f A$$

$$P = \sigma_f A \Rightarrow P = 1392 \text{ kN}$$

$$(4) N_4 = \sigma_f A$$

$$\sqrt{3} P = \sigma_f A \Rightarrow P = 803,672 \text{ kN}$$

⊗ Barras a compresión: barra (2), barra (4)

BARRA 2

$$L_k = 4 \text{ m}$$

$$\frac{h}{b} = 1 \leq 1,2$$

$$t \leq 100 \text{ mm}$$

$$3-235$$

eje z

$$\text{curva b} \Rightarrow \alpha = 0,34$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_y}{L_k^2} = 1729123,849 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 0,8972357453 \rightarrow A \gamma$$

$$\phi = 0,5 [1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 0,021046062 \rightarrow B \gamma$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0,6629589776$$

$$P_b = \chi \cdot \sigma_f \cdot A = 922,839 \text{ kN} = N_2 = \frac{P}{\sqrt{3}} \Rightarrow P = 1598,404 \text{ kN}$$

BARRA 4

$$L_k = 8m$$

$$\frac{h}{b} = 1 \leq 1'2$$

$$t \leq 40mm$$

$$S-235$$

$$eje z$$

$$\left. \begin{array}{l} L_k = 8m \\ \frac{h}{b} = 1 \leq 1'2 \\ t \leq 40mm \\ S-235 \\ eje z \end{array} \right\} \text{curva a} \Rightarrow \alpha = 0'21$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_y}{L_k^2} = 432280,9621N$$

$$\bar{\lambda} = \sqrt{\frac{A \cdot \sigma_f}{P_{cr}}} = 1'7944714913 - A4$$

$$\phi = 0'5 [1 + \alpha (\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 2'381124119$$

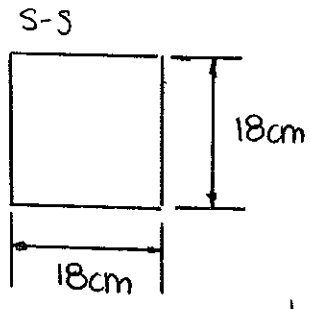
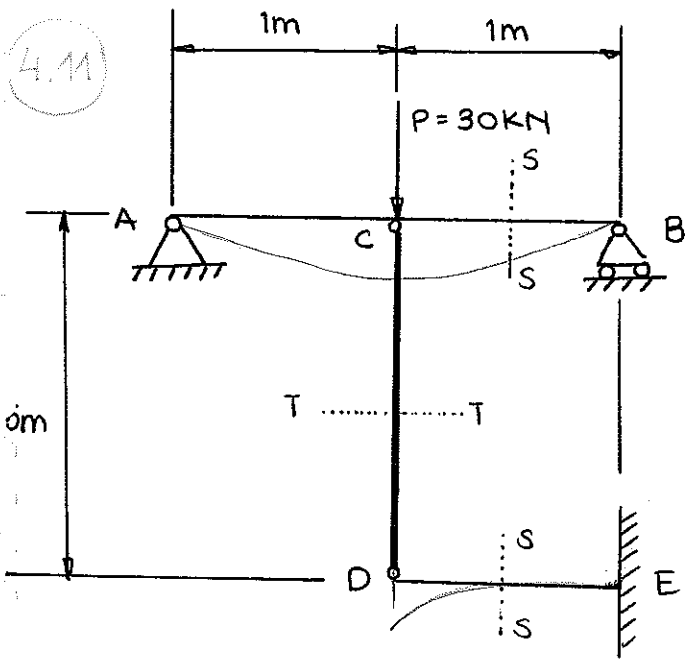
$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0'2534050265$$

$$P_b = \chi \cdot \sigma_f \cdot A = 352,740kN = \sqrt{3} P$$

$$P = 203,654kN$$



4.11



$$I_s = \frac{1}{12} (18\text{cm})^4 = 8.748 \cdot 10^{-5} \text{m}^4$$

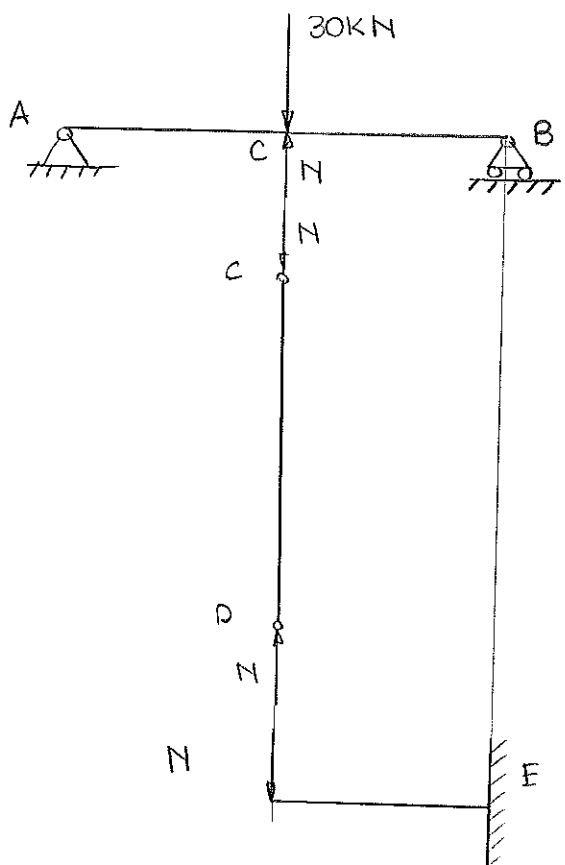
T-T



$$I_T = \frac{\pi (1\text{cm})^4}{4} = 7.853981634 \cdot 10^{-9} \text{m}^4$$

- Acero S-275
- $E = 210\text{GPa}$
- $\sigma_f = 260\text{MPa}$

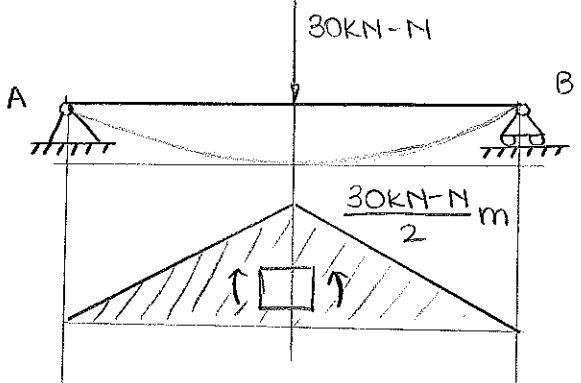
1



2

3

1



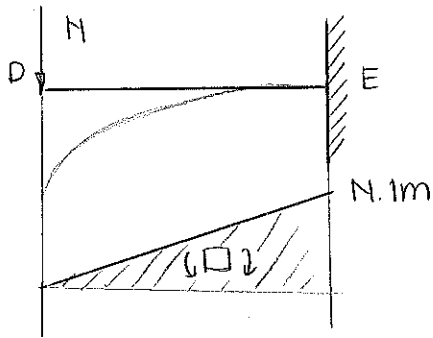
● Ec. compatibilidad

$$\delta_c - \delta_D = \Delta L_{CD}$$

$$\delta_B = \delta_{AB} = \frac{1}{EI_s} \left[\frac{1}{2} \frac{30\text{KN-N}}{2} \text{m} \cdot 1\text{m} \cdot \frac{2}{3} \text{m} \right] = \frac{1}{6EI_s} (30\text{KN-N}) \text{m}^3$$

$$\textcircled{2} \quad \Delta L_{CD} = \frac{N \cdot 1'5m}{E \cdot A_T} = \frac{N \cdot 1'5m}{E \cdot \pi (1cm)^2} = 2'273642044 \cdot 10^{-8} \frac{m}{N}$$

3



$$\delta_D = \frac{1}{E I_s} \left[\frac{1}{2} N \cdot 1m \cdot 1m \cdot \frac{2}{3} 1m \right] = \frac{N}{3 E I_s} m^3$$

⊗ Ec. compatibilidad

$$\frac{1}{6 E I_s} (30kN - N) m^3 - \frac{N}{3 E I_s} m^3 = 2'273642044 \cdot 10^{-8} N \frac{m}{N}$$

$$N = 5448,485356 N$$

Pandeo en CD

$$L_k = 1'5m$$

⊗ redondo macizo: curva c $\Rightarrow \alpha = 0'49$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_T}{L_k^2} = 7234,797892 N$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 3'360070646 \text{ } \rightarrow \text{A} \gamma$$

$$\Phi = 0'5 [1 + \alpha (\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 6'919254688 \text{ } \rightarrow \text{B} \gamma$$

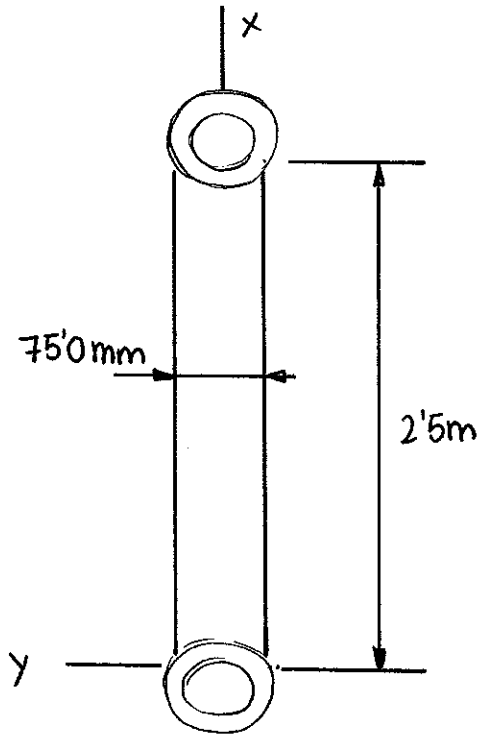
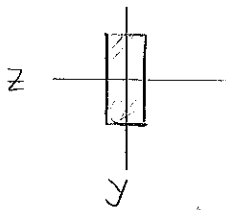
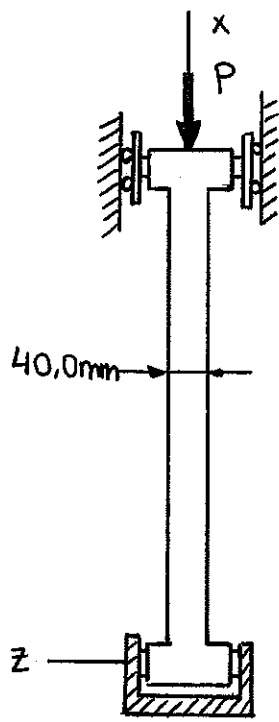
$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0'07711353686$$

$$P_b = \chi \cdot \sigma_f \cdot A = 6298,742$$

$$n = \frac{P_b}{N} = 1'156$$

$$n = 1'156$$

C-4.2



● $E = 70 \text{ GPa}$

● $n = 2.5$

$$I_z = \frac{1}{12} (40 \text{ mm}) (75 \text{ mm})^3 = 1.40625 \cdot 10^{-6} \text{ mm}^4$$

$$I_y = \frac{1}{12} (75 \text{ mm}) (40 \text{ mm})^3 = 4 \cdot 10^{-7} \text{ mm}^4$$

PLANO XY

$$L_k = 2.5 \text{ m}$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_z}{L_k^2} = 155,446 \text{ kN}$$

PLANO XZ

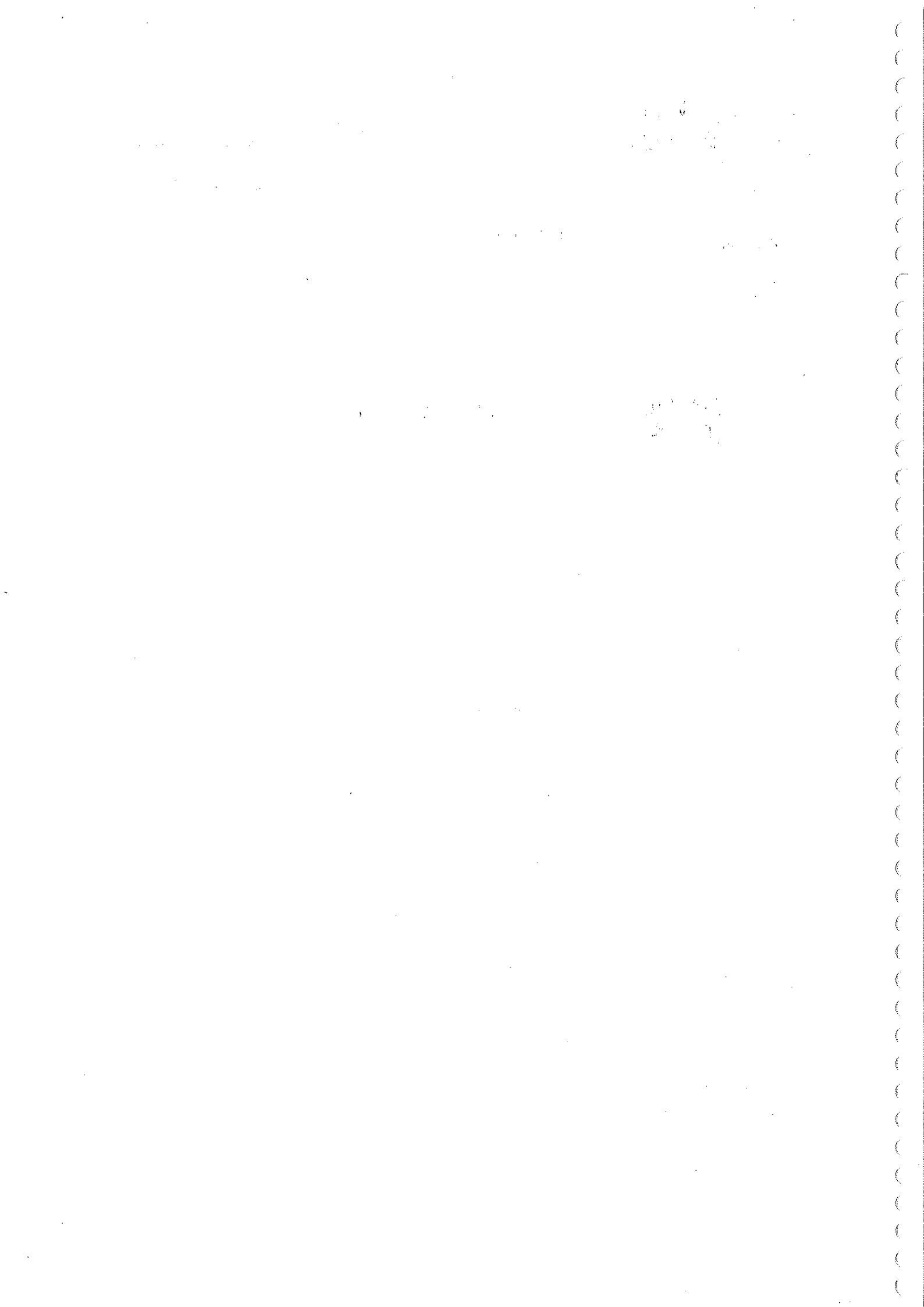
$$L_k = 1.25 \text{ m}$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_y}{L_k^2} = 176,863 \text{ kN}$$

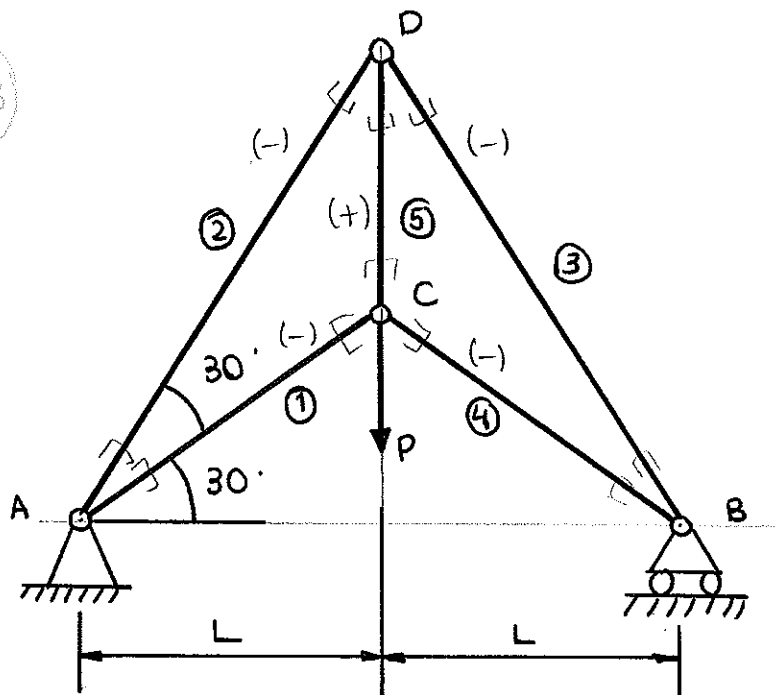
El pandeo ocurre en el plano XY (He copiado los ejes al revés :))

$$P = \frac{P_{cr}}{n} = 62,178 \text{ kN}$$

$$P = 62,178 \text{ kN}$$



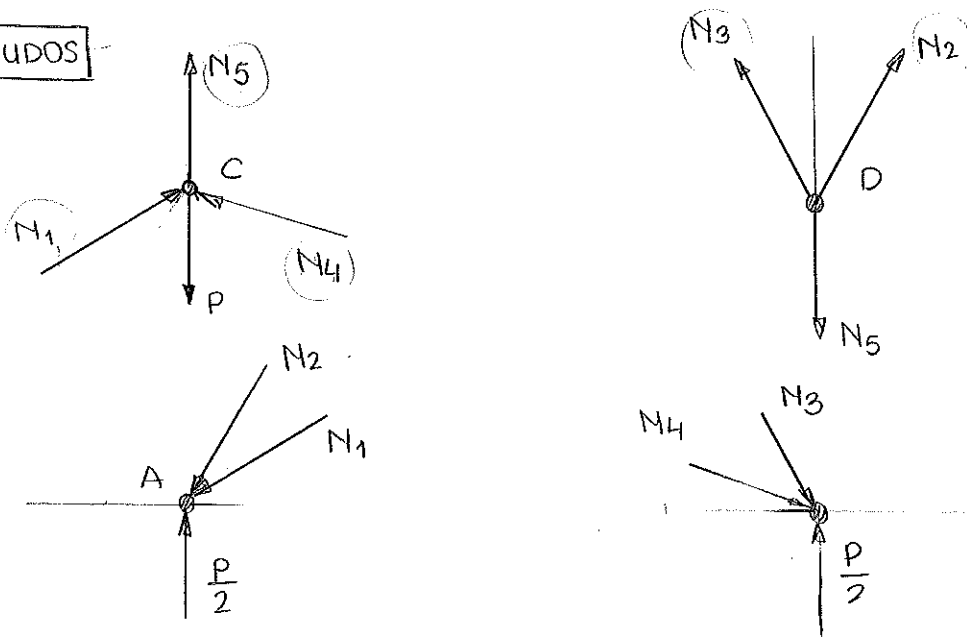
C-43



- ① $L = 2\text{m}$
- ② $P = 250\text{kN}$
- ③ Acero S=235
- ④ $E = 210\text{GPa}$, $\sigma_f = 240\text{MPa}$
- (UPN)

● HIPÓTESIS: Barras ①, ②, ③, ④ : Compresión
Barras ⑤ : Tracción

NUDOS



● Por simetría: $N_2 = N_3$
 $N_1 = N_4$

NUDO A: $N_2 \cdot \text{sen}60 + N_1 \cdot \text{sen}30 = \frac{P}{2}$
 $N_1 \cdot \text{cos}30 + N_2 \cdot \text{sen}30 = 0$

$N_1 = -125\text{ kN} = N_4$
 $N_2 = 216,506\text{ kN} = N_3$

NUDO D: $N_5 = 2N_2 \cdot \text{sen}60 = 374,999\text{ kN} \approx 375\text{ kN}$

⊗ RESISTENCIA A TRACCIÓN/COMPRESIÓN

$$\frac{N_5}{A} = \sigma_f \Rightarrow A = 1,5625 \cdot 10^{-3} \text{ m}^2 = 15,625 \text{ cm}^2 \Rightarrow \boxed{\text{UPN-120}}$$

PANDEO (Perfil UPN \rightarrow curva c $\Rightarrow \alpha = 0,49$)

⊗ trabajan a compresión las barras 2 y 3

$$L_k = 4 \text{ m} ; N_2 = 216,506 \text{ kN}$$

⊗ suponemos $\boxed{\text{UPN-120}}$

$$\left. \begin{array}{l} I_x = 364 \text{ cm}^4 \\ I_y = 43,2 \text{ cm}^4 \\ A = 17 \text{ cm}^2 \end{array} \right\} I_y < I_x$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_y}{L_k^2} = 55960,65695 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 2,900154902 \leftarrow A \quad \Phi = 0,5 [1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 4,757956199 \leftarrow B$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0,1152668581$$

$$P_b = \chi \cdot \sigma_f \cdot A = 47028,87811 \ll N_2 \quad \boxed{\text{No nos sirve}}$$

⊗ suponemos $\boxed{\text{UPN-200}}$

$$\left. \begin{array}{l} I_x = 1910 \text{ cm}^4 \\ I_y = 148 \text{ cm}^4 \\ A = 32,2 \text{ cm}^2 \end{array} \right\} I_y < I_x$$

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_y}{L_k^2} = 191717,0655 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 2,007720116 \leftarrow A \quad \Phi = 0,5 [1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 2,95836146 \leftarrow B$$

$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \bar{\lambda}^2}} = 0,1948885014$$

$$P_b = \chi \cdot A \cdot \sigma_f = 150609,8338 \text{ N} < N_2 \quad \boxed{\text{No nos sirve}}$$

Suponemos UPN-220

$$\left. \begin{aligned} I_x &= 2690 \text{ cm}^4 \\ I_y &= 197 \text{ cm}^4 \\ A &= 37.4 \text{ cm}^2 \end{aligned} \right\} I_y < I_x$$

$$P_{cr} = \frac{\pi^2 E I_y}{L_k^2} = 255190,9588 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 1.875464207 \rightarrow A \quad \phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 2.669171727 \rightarrow B$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.2188944669$$

$$P_b = \chi \sigma_f A = 196479,6735 \text{ N} < N_2: \text{ No nos sirve}$$

Suponemos UPN-240

$$\left. \begin{aligned} I_x &= 3600 \text{ cm}^4 \\ I_y &= 248 \text{ cm}^4 \\ A &= 42.3 \text{ cm}^2 \end{aligned} \right\} I_y < I_x$$

$$P_{cr} = \frac{\pi^2 E I_y}{L_k^2} = 321255,6233 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{A \sigma_f}{P_{cr}}} = 1.777667102 \rightarrow A \quad \phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2] = 2.466578603 \rightarrow B$$

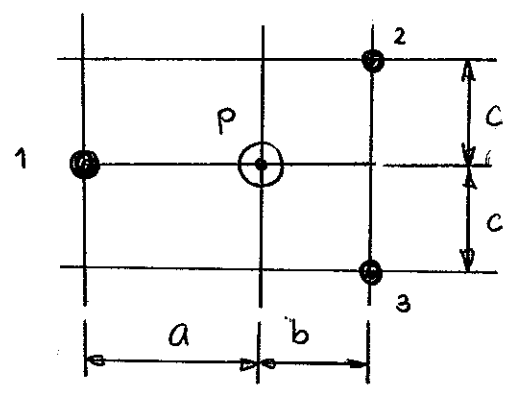
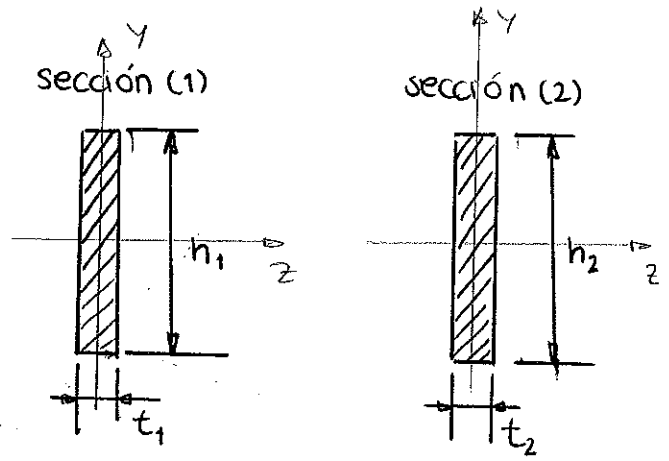
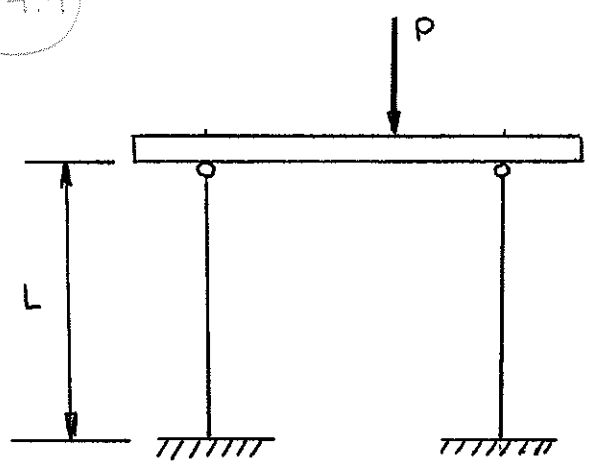
$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0.2394336177$$

$$P_b = \chi \sigma_f A = 243073,0087 \text{ N} > N_2 \quad \boxed{\text{Nos sirve}}$$

El perfil más adecuado es el UPN-240.

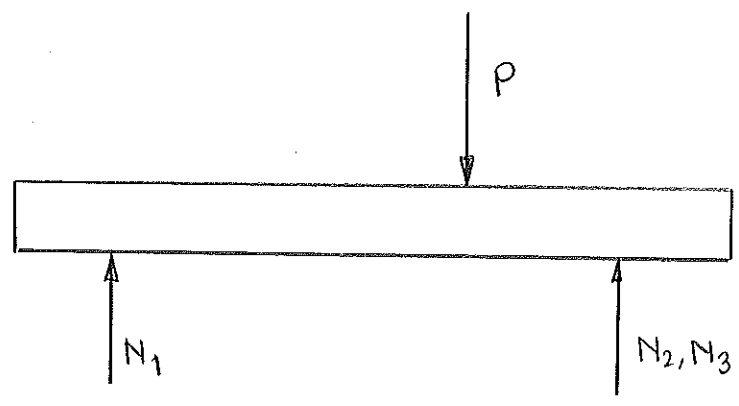


C-4.4



- ⊙ $L = 20 \text{ cm}$
- ⊙ $h_1 = 25 \text{ mm}; t_1 = 5 \text{ mm}; h_2 = 25 \text{ mm}, t_2 = 3 \text{ mm}$
- ⊙ $a = 10 \text{ cm}, b = 4.5 \text{ cm}, c = 5 \text{ cm}$
- ⊙ $E = 110 \text{ GPa}, \sigma_f = 495 \text{ MPa}$

A)



⊙ Por simetría $N_2 = N_3$

$$\sum F_y: N_1 + 2N_2 = P \Rightarrow N_2 = 0.344828 P$$

$$\sum M = 0: N_1(a+b) = P \cdot b \Rightarrow N_1 = \frac{b}{a+b} P = 0.310345 P$$

RESISTENCIA COMPRESIÓN

$$\frac{N_2}{A_2} = \sigma_f : 0.344828 P = 3 \text{ mm} \cdot 25 \text{ mm} \cdot 495 \text{ MPa} \Rightarrow \boxed{107,662 \text{ kN} = P}$$

$$\frac{N_1}{A_1} = \sigma_f : 0.310345 P = 5 \text{ mm} \cdot 25 \text{ mm} \cdot 495 \text{ MPa} \Rightarrow \boxed{P = 199,375 \text{ kN}}$$

PANDEO.

⊙ Pilar (1)

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_{y_1}}{L_k^2} = 14424,645 \text{ N.}$$

$$N_1 = P_{cr}$$

$$0,310345P = 14424,645 \text{ N}$$

$$P = 46,479 \text{ KN}$$

$$I_{z_1} = \frac{1}{12} 5 \text{ mm} (25 \text{ mm})^3 = 6,510416667 \cdot 10^{-9} \text{ m}^4$$

$$I_{y_1} = \frac{1}{12} 25 \text{ mm} (5 \text{ mm})^3 = 2,604166667 \cdot 10^{-10} \text{ m}^4$$

$$A = 25 \text{ mm} \cdot 5 \text{ mm} = 125 \cdot 10^{-4} \text{ m}^2$$

$$I_{y_1} < I_{z_1}$$

$$L_k = 0,7L = 0,14 \text{ m}$$

⊙ Pilar (2)

$$P_{cr} = \frac{\pi^2 \cdot E \cdot I_{y_2}}{L_k^2} = 3115,723328 \text{ N}$$

$$N_2 = P_{cr}$$

$$0,344828P = 3115,723328 \text{ N}$$

$$P = 9035,587 \text{ N}$$

$$I_{z_2} = \frac{1}{12} 3 \text{ mm} (25 \text{ mm})^3 = 3,90625 \cdot 10^{-9} \text{ m}^4$$

$$I_{y_2} = \frac{1}{12} 25 \text{ mm} (3 \text{ mm})^3 = 5,625 \cdot 10^{-11} \text{ m}^4$$

$$A = 25 \text{ mm} \cdot 3 \text{ mm} = 75 \cdot 10^{-6} \text{ m}^2$$

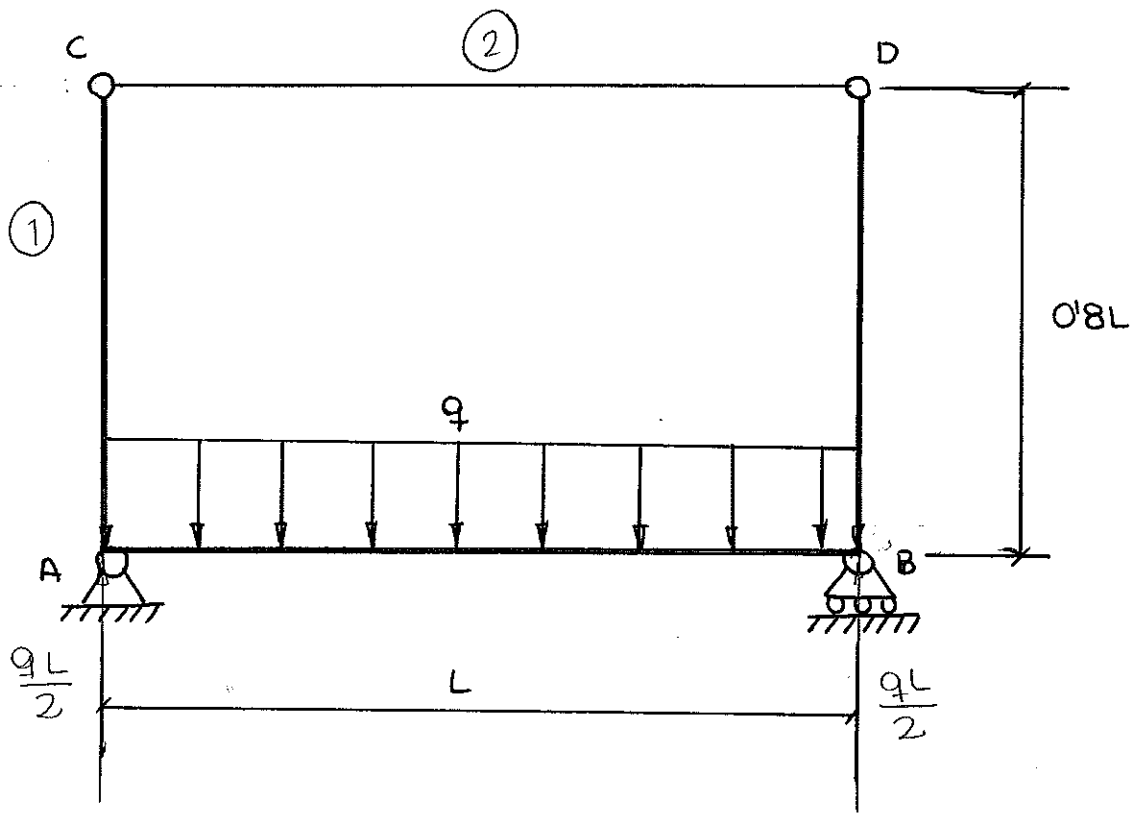
$$L_k = 0,7L = 0,14 \text{ m}$$

$$I_{y_2} < I_{z_2}$$

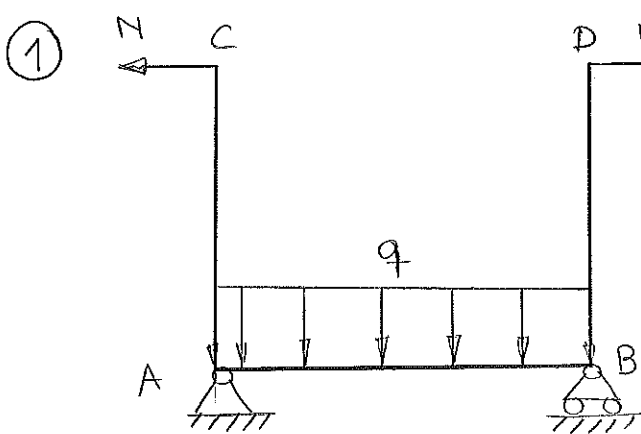
La máxima carga P que puede resistir el soporte es de 9035,587 N

b)

C-4.5



- Barras AB, AC y CD: $S = 6 \times 8 \text{ cm}^2$
- Barra CD: sección circular: $d = 2 \text{ cm}$
- $L = 1 \text{ m}$
- Acero S-235 (curva c: $\alpha = 0.49$)
- $E = 210 \text{ GPa}$, $\sigma_f = 240 \text{ MPa}$.

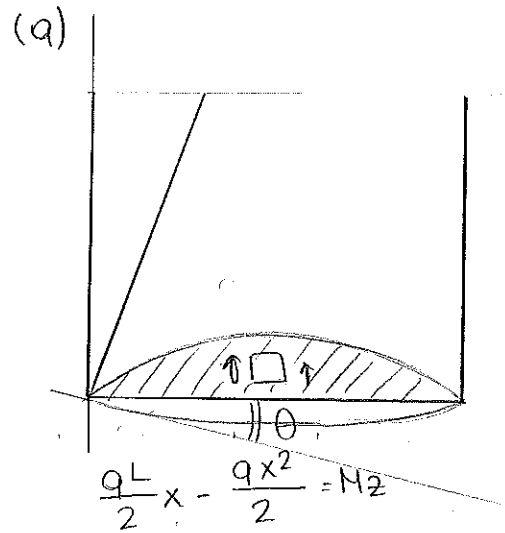
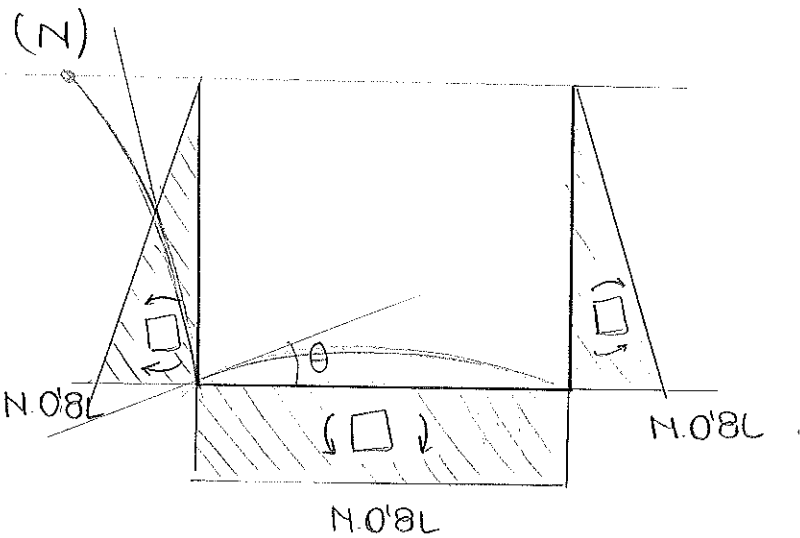


$$dc_2 = \frac{\Delta L}{2} = \frac{NL}{2EA}$$

$$I = 2,56 \cdot 10^{-6} \text{ m}^4$$

Ec. compatibilidad: $dc_1 = dc_2$

① Diagramas de esfuerzos flectores



$$\delta_c = \delta_c^{(N)} - \delta_c^{(q)}$$

$$\delta_c^{(N)} = \theta_A \cdot 0.8L + \delta_{CA}$$

$$\theta_A = \frac{\delta_{BA}}{L}$$

$$\delta_{BA} = \frac{1}{EI} \left[N \cdot 0.8L \cdot L \cdot \frac{L}{2} \right] = \frac{2NL^3}{5EI}$$

$$\theta_A = \frac{2NL^2}{5EI}$$

$$\delta_{CA} = \frac{1}{EI} \left[\frac{1}{2} N \cdot 0.8L \cdot \frac{2}{3} \cdot 0.8L \cdot 0.8L \right] = \frac{64NL^3}{375EI}$$

$$\delta_c^{(N)} = \frac{184NL^3}{375EI}$$

$$\delta_c^{(q)} = \theta_A \cdot 0.8L$$

$$\theta_A = \frac{\delta_{BA}}{L}$$

$$\delta_{BA} = \frac{1}{EI} \int_0^L \left(\frac{qLx}{2} - \frac{qx^2}{2} \right) x dx = \frac{1}{EI} \left[\frac{qL}{2} \frac{L^3}{3} - \frac{qL^4}{8} \right] = \frac{qL^4}{24EI}$$

$$\delta_c^{(q)} = \frac{qL^4}{30EI}$$

$$\delta_c^{(1)} = -\frac{184NL^3}{375EI} + \frac{qL^3}{30EI}$$

$$\frac{-184NL^3}{375EI} + \frac{qL^3}{30EI} = \frac{NL}{2EA}$$

$$q = \frac{30I}{L^3} \left(\frac{184NL^3}{375I} + \frac{NL}{2A} \right) = 14,8422N \text{ (B)}$$

⊙ Esfuerzo flector máximo: (C)

$$\sigma_{xx} = \frac{0,8 N \cdot 4cm}{I} = 240 \text{ MPa} : \begin{cases} N_{\text{máx}} = 19200 \text{ N} \\ q_{\text{máx}} = 284970,8357 \text{ N/m} \end{cases}$$

() Pandeo: tirante

$$L_k = L \quad I = 7'853981634 \cdot 10^{-9} \text{ (A)}$$

$$P_{cr} = \frac{\pi^2 EI}{L_k^2} = 16278,29526 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{\sigma_c \cdot A}{P_{cr}}} = 2,152167637 \text{ (C)}$$

$$\phi = 0,5 [1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 3,294119384 \text{ (D)}$$

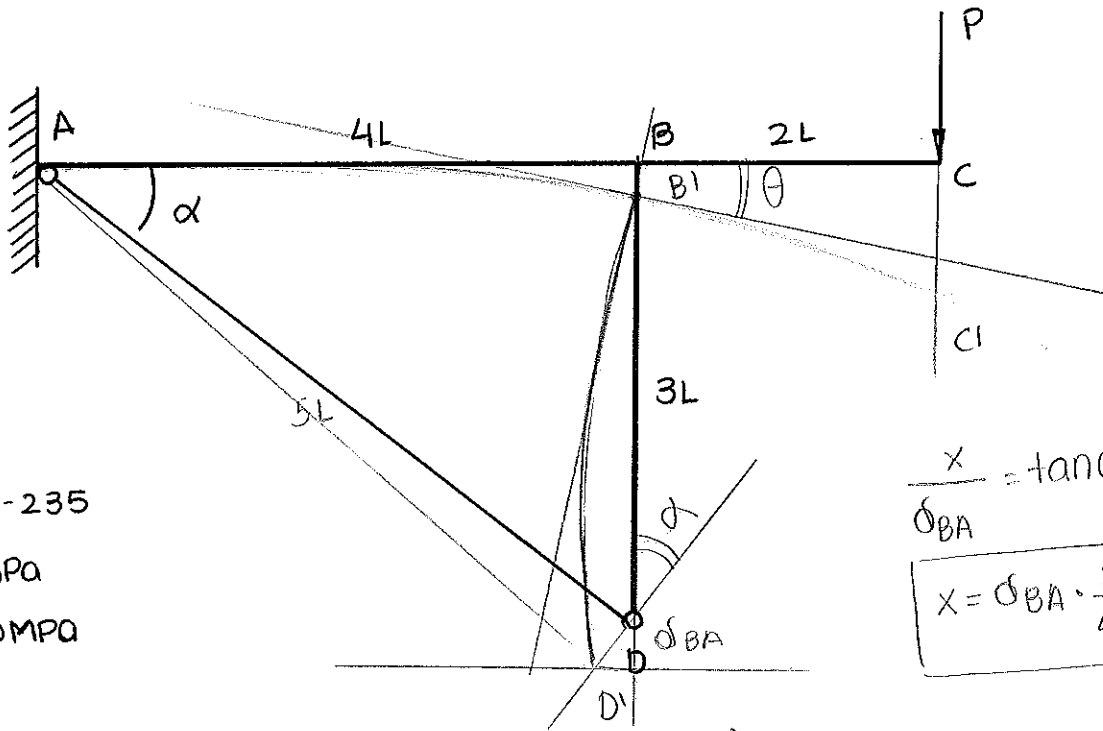
$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0,1727663535$$

$$P_b = \chi \cdot A \cdot \sigma_c = 13026,27616 \text{ N} = N \Rightarrow q = 193,33 \text{ kN/m}$$

$$q_{\text{máx}} = 193,33 \frac{\text{kN}}{\text{m}}$$



C-4.6



● Acero S-235

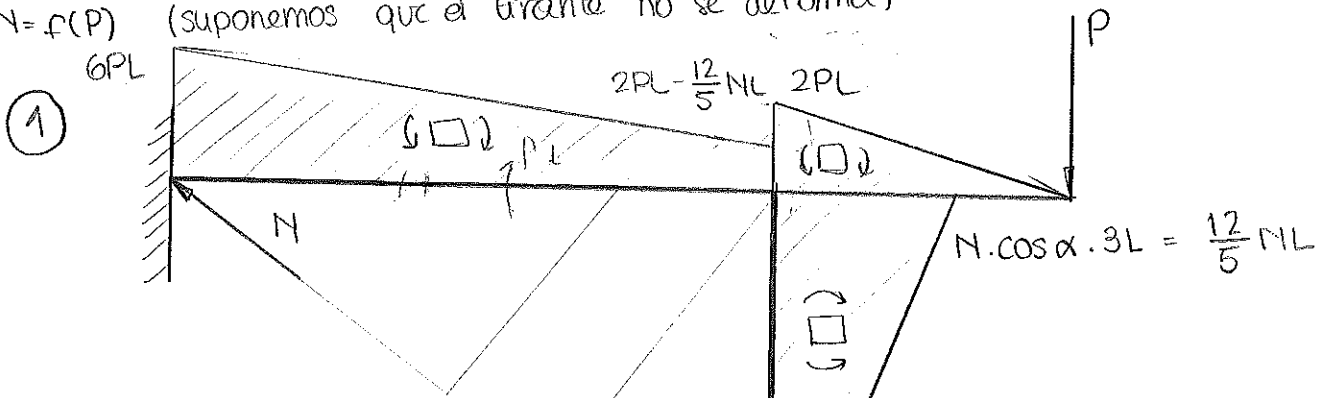
● $E = 210 \text{ GPa}$

● $\sigma_f = 240 \text{ MPa}$

$$\frac{x}{\delta_{BA}} = \tan \alpha$$

$$x = \delta_{BA} \cdot \frac{3}{4}$$

$N = f(P)$ (suponemos que el tirante no se deforma)



Ec. compatibilidad

$$\theta \cdot 3L - \delta_{DB} = x$$

$$\theta = \theta_{BA} = \frac{1}{EI} \left[(2PL - \frac{12}{5}NL) 4L + \frac{1}{2} (4PL + \frac{12}{5}NL) 4L \right]$$

$$\delta_{DB} = \frac{1}{EI} \left[\frac{1}{2} \frac{12}{5}NL \cdot 3L \cdot \frac{2}{3} 3L \right] = \frac{36NL^3}{5EI}$$

$$\delta_{BA} = \frac{1}{EI} \left[(2PL - \frac{12}{5}NL) 4L \cdot 2L + \frac{1}{2} (4PL + \frac{12}{5}NL) \cdot 4L \cdot \frac{2}{3} 4L \right] =$$

$$= \frac{1}{EI} \left[\frac{112}{3} PL^3 - \frac{32}{5} NL^3 \right]$$

$$\frac{12L^3}{EI} (4P - \frac{6}{5}N) - \frac{36NL^3}{5EI} = \frac{3}{4EI} \left[\frac{112PL^3}{3} - \frac{32}{5}NL^3 \right]$$

$$20P = \frac{24}{5}N$$

$$N = 1'19P$$

$$P = 10\text{KN}$$

$$L = 1'5\text{m}$$

$$\alpha = 0'49$$

sección 1: $b = 50\text{mm}$

$$N = 1'19P = 11904,7619\text{N}$$

$$P_{cr} = \frac{E \pi^2 I}{L_k^2} = 479772,4362\text{N}$$

$$\bar{\lambda} = \sqrt{\frac{\sigma_f A}{P_{cr}}} = 1,118299108 \text{ (A)}$$

$$\Phi = 0'5 [1 + \alpha (\bar{\lambda} - 0'2) + \bar{\lambda}^2] = 1'350279729$$

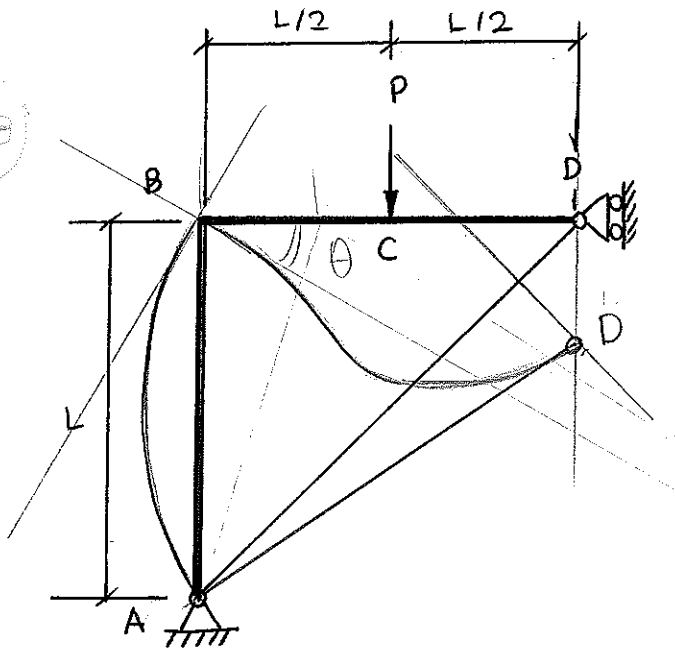
$$\chi = \frac{1}{\Phi + \sqrt{\Phi^2 + \bar{\lambda}^2}} = 0'37029$$

$$P_b = \chi \cdot \sigma_f \cdot A = 22276,1858\text{N}$$

(...)

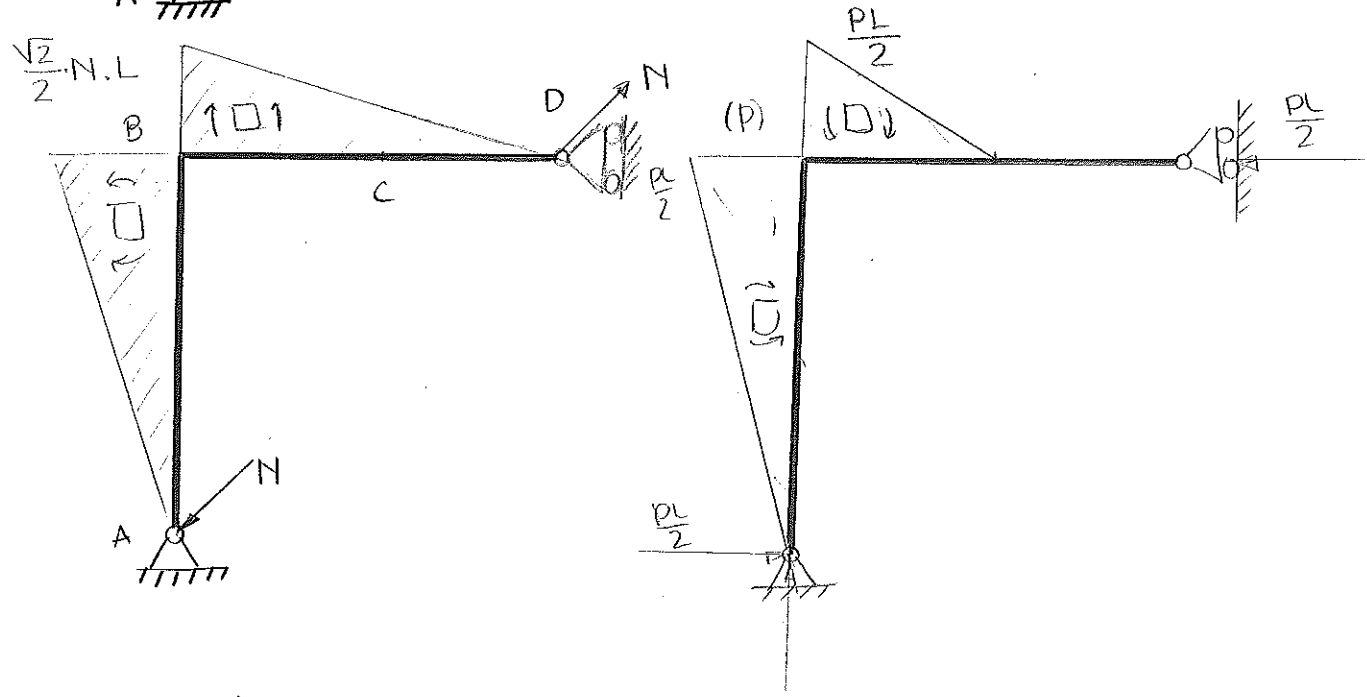
$$b = 45\text{mm}$$

C-4.8



- $L = 1\text{m}$
- Sección: $s = 6 \times 8\text{cm}^2$
- Tirante: $s = 3 \times 3\text{cm}^2$
- Acero S-275
- $E = 210\text{GPa}$ $I = 2,56 \cdot 10^{-6}\text{m}^4 \text{ (A)}$
- $\sigma_f = 260\text{MPa}$

1
(N)



Ec. compatibilidad

$$\downarrow y_D = \Delta t \cdot \sqrt{2}$$

$$\Delta t = \frac{N \cdot \sqrt{2} L}{2EA}$$

$$\downarrow y_D = \theta \cdot L - \delta_{DB}$$

$$\theta = \frac{\delta_{AB}}{L}$$

$$\delta_{AB} = \delta_{AB}^{(P)} - \delta_{AB}^{(N)}$$

$$\delta_{AB}^{(P)} = \frac{1}{EI} \left[\frac{1}{2} \frac{PL}{2} \cdot L \cdot \frac{2}{3} L \right] = \frac{PL^3}{6EI}$$

$$\delta_{AB}^{(N)} = \frac{1}{EI} \left[\frac{1}{2} \frac{\sqrt{2} NL}{2} \cdot L \cdot \frac{2}{3} L \right] = \frac{\sqrt{2} NL^3}{6EI}$$

$$\delta_{DB}^{(P)} = \frac{1}{EI} \left[\frac{1}{2} \frac{PL}{2} \cdot \frac{L}{2} \cdot \left(\frac{2}{3} \frac{L}{2} + \frac{L}{2} \right) \right] = \frac{5 PL^3}{48 EI}$$

$$\delta_{DB}^{(N)} = \frac{\sqrt{2} NL^3}{6EI}$$

$$\frac{PL^3}{6EI} - \frac{\sqrt{2}NL^3}{6EI} - \frac{\sqrt{2}NL^3}{6EI} + \frac{5PL^3}{48EI} = \frac{NL}{EA}$$

$$5'037822421 \cdot 10^{-7} P = 8,821595335 \cdot 10^{-7} N$$

$$N = 0'5710783354P$$

Esfuerzo flexor máximo

$$\textcircled{B} M_z = 0,09618660108PL \Rightarrow \sigma_{\text{máx}} = \frac{M_z \cdot 4\text{cm}}{I} = \sigma_f \Rightarrow P = 186,304 \text{ kN}$$

$$\textcircled{C} M_z = \frac{\sqrt{2}}{4} NL \Rightarrow \sigma_{\text{máx}} = \frac{M_z \cdot 4\text{cm}}{I} = \sigma_f \Rightarrow P = 82'414 \text{ kN}$$

Pandeo $L_k = \sqrt{2}L$

$$P_{cr} = \frac{\pi^2 E I}{L_k^2} = 69950,82119 \text{ N}$$

$$\bar{\lambda} = \sqrt{\frac{\sigma_f A}{P_{cr}}} = 1'898033449 \text{ (B)}$$

$$\phi = 0'5(1 + 0'49(\bar{\lambda} - 0'2) + \bar{\lambda}^2) = 2,717283682$$

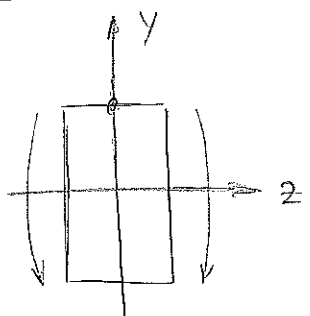
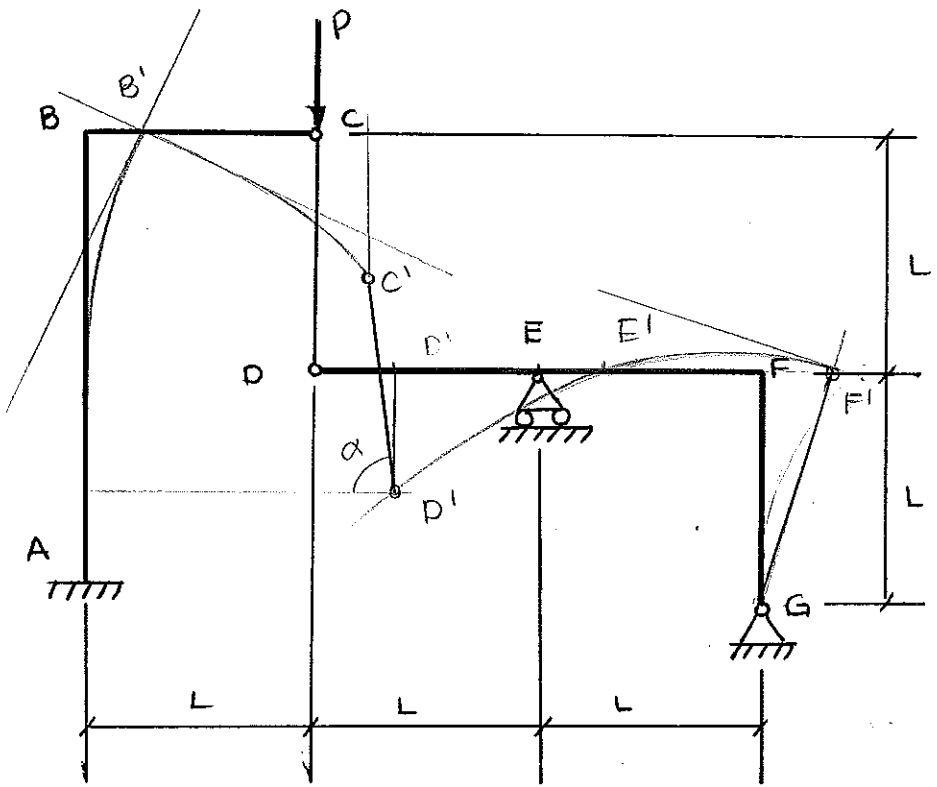
$$\chi = 0'2145099385$$

$$P_b = 54056,50449 \text{ N} = N \Rightarrow P = 94'657 \text{ kN}$$

Está hecho con $\sigma_f = 280 \text{ MPa}$

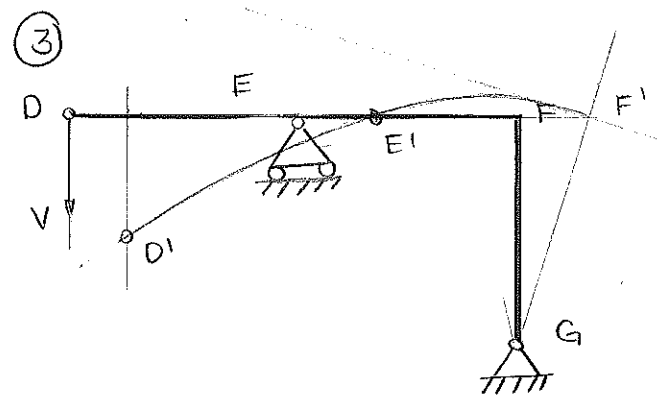
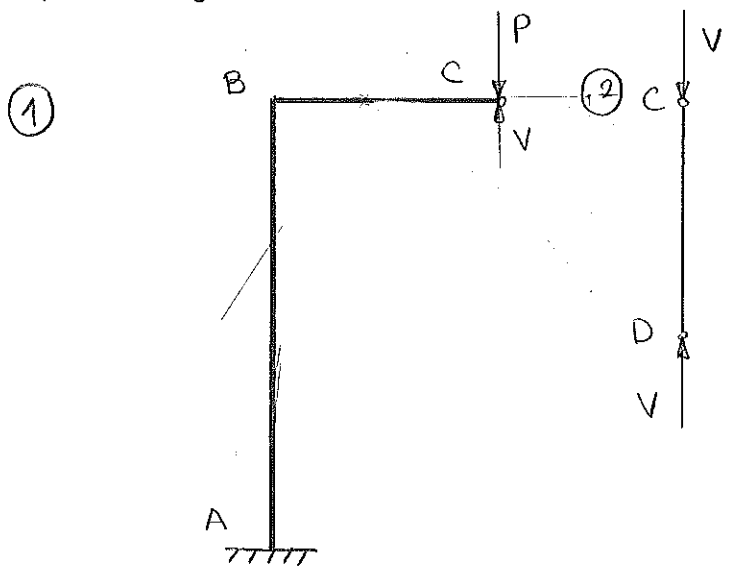
Falla por flexión en C con $P = 82'414 \text{ kN}$

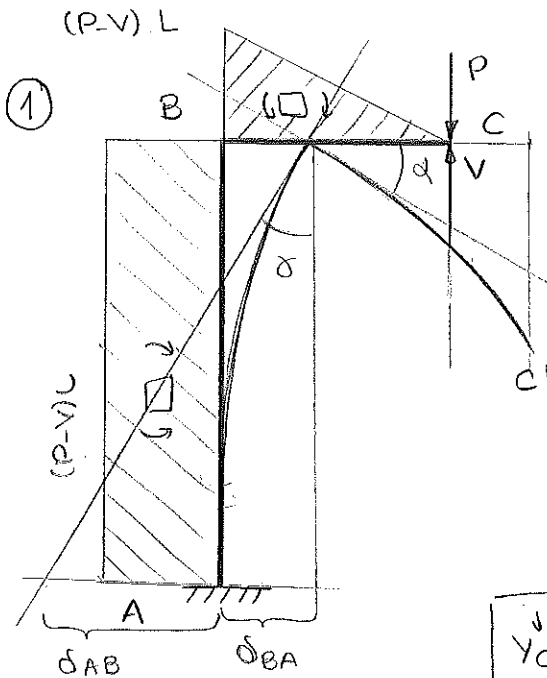
C-4.9



- Barra CD: $A_{co} = 2 \times 2 \text{ cm}^2$
- Resto Barras: $A = 6 \times 8 \text{ cm}^2$; $I = \frac{1}{12} 6 \text{ cm} (8 \text{ cm})^3 = 2,56 \cdot 10^{-6} \text{ m}^4$
- $L = 1 \text{ m}$
- Acero S-275 ; $E = 200 \text{ GPa}$; $\sigma_f = 280 \text{ MPa}$.

(Se desprecian los efectos de los esfuerzos axial y cortante en los elementos que trabajan a flexión)





$$Y_{C1} = \alpha \cdot L + \delta_{CB}$$

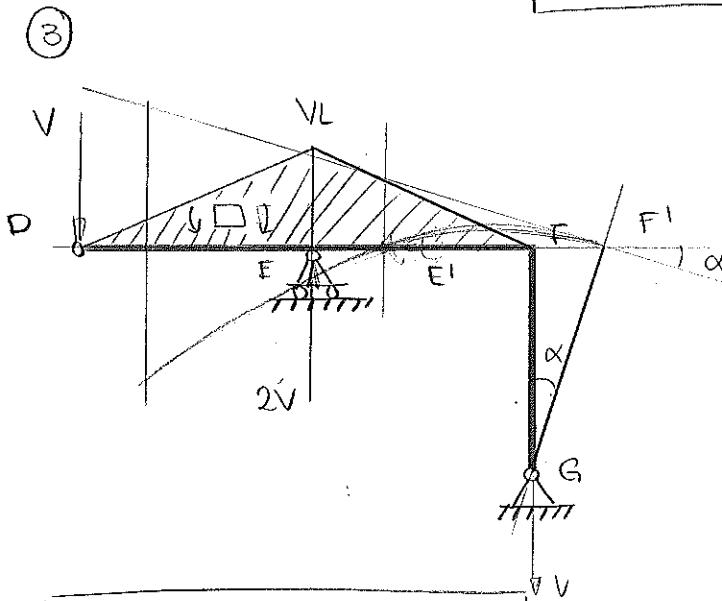
$$\alpha = \frac{\delta_{AB} + \delta_{BA}}{2L} = \frac{2(P-V)L^2}{EI}$$

$$\delta_{AB} = \frac{1}{EI} \left[\frac{1}{2} (P-V)L \cdot 2L \cdot L \right] = \frac{2(P-V)L^3}{EI} = \delta_{BA}$$

$$\delta_{CB} = \frac{1}{EI} \left[\frac{1}{2} (P-V)L \cdot L \cdot \frac{4}{3}L \right] = \frac{(P-V)L^3}{6EI}$$

$$Y_{C1} = \frac{2(P-V)L^3}{EI} + \frac{(P-V)L^3}{6EI} = \frac{13(P-V)L^3}{6EI}$$

$$BB' = \delta_{BA} = \frac{2(P-V)L^3}{EI}$$



$$Y_{D3} = \delta_{DF} - \alpha \cdot 2L$$

$$\alpha = \frac{\delta_{EF}}{L} = \frac{VL^2}{6EI}$$

$$\delta_{EF} = \frac{1}{EI} \left[\frac{1}{2} VL \cdot L \cdot \frac{1}{3}L \right] = \frac{VL^3}{6EI}$$

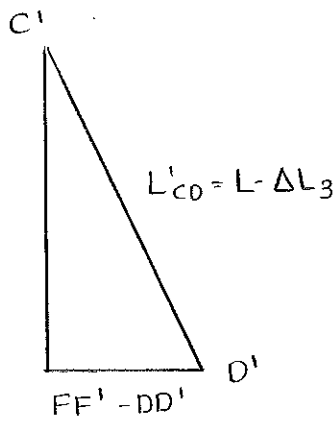
$$\delta_{DF} = \frac{1}{EI} \left[\frac{1}{2} VL \cdot 2L \cdot L \right] = \frac{VL^3}{EI}$$

$$Y_{D3} = \frac{VL^3}{EI} - \frac{VL^3}{3EI} = \frac{2VL^3}{3EI}$$

$$FF' = \alpha \cdot L = \frac{VL^3}{6EI}$$

②

$$\Delta L_3 = \frac{P \cdot L}{A_{co} \cdot E}$$



⊗ Ec. Compatibilidad:

$$\left(L + \frac{2VL^3}{3EI} - \frac{13(P-V)L^3}{6EI} \right)^2 + \left(\frac{VL^3}{6EI} \right)^2 = \left(L - \frac{PL}{Ac_0E} \right)^2$$

$V = f(P)$

⊗ Flexión máxima:

$$\textcircled{E} \quad \sigma_{xx} = \frac{VL \cdot 4\text{cm}}{2,56 \cdot 10^{-6} \text{m}^4} = 15625 \cdot V = 280 \text{MPa}$$

$$V = 17920 \text{N} \Rightarrow P = 23507,30 \text{N}$$

(A)

Para $P = 23507,30 \text{N}$

$$\textcircled{A} \quad \sigma_{xx} = \frac{(P-V) \cdot L \cdot 4\text{cm}}{2,56 \cdot 10^{-6} \text{m}^4} = 87,302 \text{MPa} < \sigma_f: \text{OK!}$$

PANDEO Si $V = 17920 \text{N}$ ¿falla? $L_k = 1\text{m}$: curva c $\alpha = 0,49$

$$P_{cr} = \frac{\pi^2 E I}{L_k^2} = 26318,94507 \text{N}$$

$$\bar{\lambda} = \sqrt{\frac{\sigma_f A}{P_{cr}}} = 2,062883884 \quad \lambda \rightarrow A \}$$

$$\phi = 0,5 [1 + \alpha (\bar{\lambda} - 0,2) + \bar{\lambda}^2] = 3,084151396 \quad \lambda \rightarrow B \}$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \bar{\lambda}^2}} = 0,1859822484$$

$$P_b = \chi \cdot \sigma_f \cdot A = 20830,01182 \text{N} > V \Rightarrow \text{No falla}$$

La estructura falla para $P = 23507,30 \text{N}$ y el fallo se produce en E por flexión

