

$$y_b = -\frac{L}{2} ; C_x = \frac{2}{3} \pi L^2$$

$$M = \frac{\pi}{2} \rightarrow (x_m, y_m, z_m)$$

Equilibrado estático

$$x_b = 0 ; \text{Intuitivo } x_m = 0$$

$$2\pi \cdot 0 + \frac{\pi}{2} x_m = 0 \Rightarrow x_m = 0$$

$$y_b = 0 \Rightarrow 2\pi \left(-\frac{L}{2}\right) + \frac{\pi}{2} y_m = 0 \Rightarrow y_m = 2L$$

$$C_z = 0 \Rightarrow 0 + \frac{\pi}{2} z_m x_m = 0 \Rightarrow \text{No aporta nada.}$$

$$\begin{matrix} XY & YZ \\ \uparrow & \uparrow \\ z_m & x_m \end{matrix} \quad C_x = 0 \quad \rightarrow \quad \frac{2}{3} \pi L^2 + \frac{\pi}{2} \cdot z_m y_m = 0$$

$$\begin{matrix} XY & XZ \\ \uparrow & \uparrow \\ z_m & y_m \end{matrix} \quad \frac{2}{3} \pi L^2 + \frac{\pi}{2} z_m \Rightarrow z_m = -\frac{2}{3} L$$

Ejercicio 9.4

A un disco de masa  $4M$ , radio  $R$  y espesor despreciable se le practica un orificio de radio  $R/2$  como muestra la figura. El sistema gira alrededor del eje horizontal  $AB$  con velocidad angular constante  $\omega$ . Determinar:

1. Reacciones en los apoyos.
2. Par que debe aplicarse para conseguir la ley de movimiento indicada.
3. Posición y magnitud de la masa puntual que se ha de colocar en la periferia del disco para equilibrar el sistema.

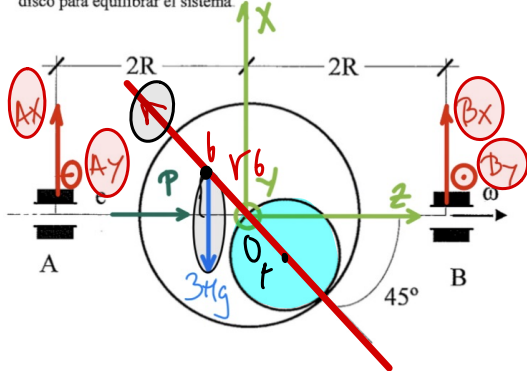


Figura  $\rightarrow$  punto,  $\omega = \text{cte}$

Reacciones  $\rightarrow$  radial  
 $\nabla$  componente axial.

$$\vec{F} = m \vec{a}_b$$

$$M = 4M - m_k$$

$$I_k = \frac{1}{2} \pi \frac{R^2}{4} ; \quad \nabla = \frac{4M}{\pi R^2}$$

$$m_k = \frac{4M}{\pi R^2} \frac{\pi R^2}{4} \Rightarrow m_k = M.$$

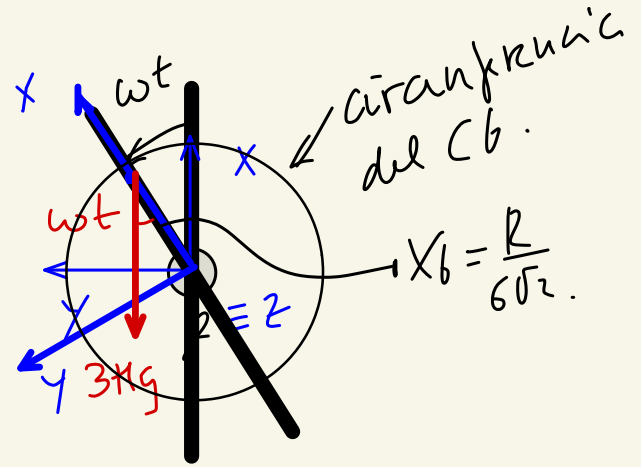
$m = 4M - M = 3M \rightarrow \vec{F} = 3M \vec{a}_b$  circunferencias de radio  $X_b$   
 $\omega = \omega t \Rightarrow b$  sólo tiene  $\alpha$

$$\vec{a}_b = -\omega^2 X_b \hat{e}_1 \rightarrow$$

$$r_b = \frac{4M \cdot 0 + (-M)(-R/2)}{3M} = \frac{R}{6}$$

$$X_b = \frac{R}{6\sqrt{2}}$$

$$\vec{a}_b = -\omega^2 \frac{R}{6\sqrt{2}} \hat{e}_1$$



$$A_x + B_x - 3Mg \cos \omega t = -M \frac{\omega^2 R}{2\sqrt{2}}$$

$$A_y + B_y + 3Mg \sin \omega t = 0$$

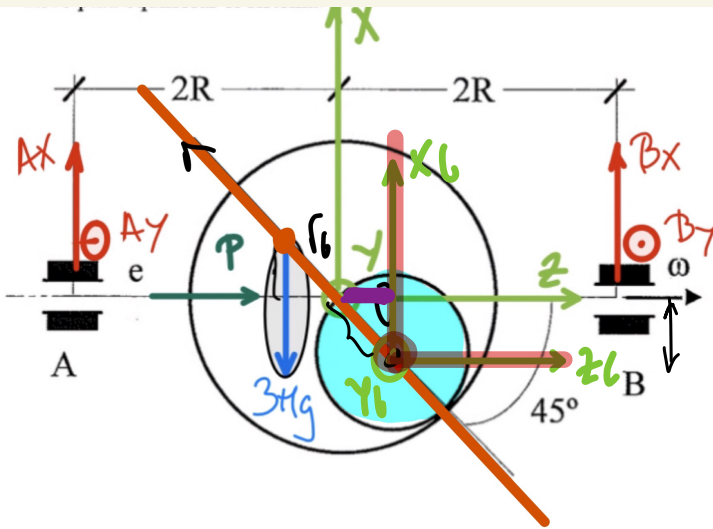
$$\vec{\omega} = \omega \hat{e}_3 \rightarrow$$

$$\vec{H}_b = -C_y \omega \hat{e}_1 - C_x \omega \hat{e}_2 + I_z \omega \hat{e}_3$$

$$\left. \frac{dH_b}{dt} \right|_F = -C_y \omega^2 \hat{e}_2$$

$$\alpha = 0$$

$C_x = 0$  plano  $xz$  contiene a la figura.



$$\left. \frac{dH_b}{dt} \right|_F = (C_x \omega^2 - C_y \alpha) \hat{e}_1 - (C_y \omega^2 + C_x \alpha) \hat{e}_2 + I_z \omega \hat{e}_3$$

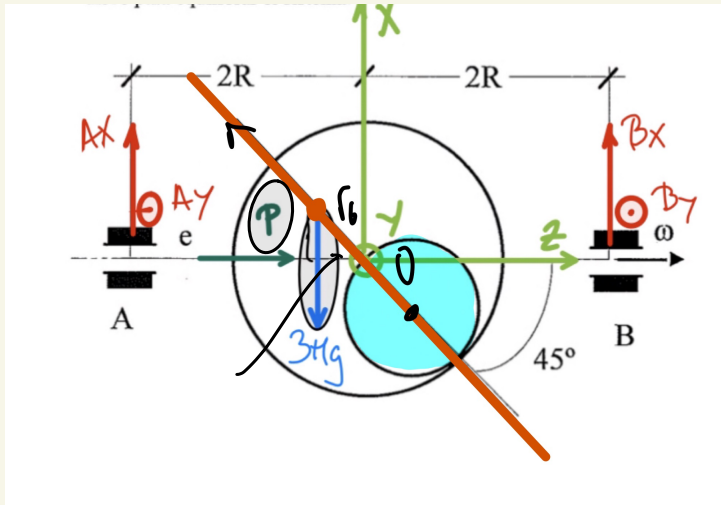
$$C_y = C_x - C_y^0 \Rightarrow C_y = -C_y^0 \Rightarrow C_y = \frac{1}{8} \pi R^2$$

$$G^h = \cancel{C \frac{h}{16}} + M \left( -\frac{R}{2\sqrt{2}} \right) \left( \frac{R}{2\sqrt{2}} \right) = -\frac{1}{8} \pi R^2$$

↓  
xy z

$$\frac{d\vec{h}_0}{dt} = -\frac{1}{8} \pi R^2 \omega^2 \hat{e}_2$$

$$\frac{d\vec{h}_0}{dt} = \vec{\pi}_0$$



$$\vec{\pi}_0^{pov} = \begin{vmatrix} \hat{e}_1 & \hat{e}_2 & \hat{e}_3 \\ \frac{R}{6\sqrt{2}} & 0 & -\frac{R}{6\sqrt{2}} \\ -3\pi g \cos \omega t & 3\pi g \sin \omega t & 0 \end{vmatrix}$$

$$\vec{\pi}_0^{pov} = \vec{O} \wedge \vec{P} \times \vec{e}_2$$

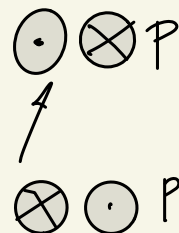
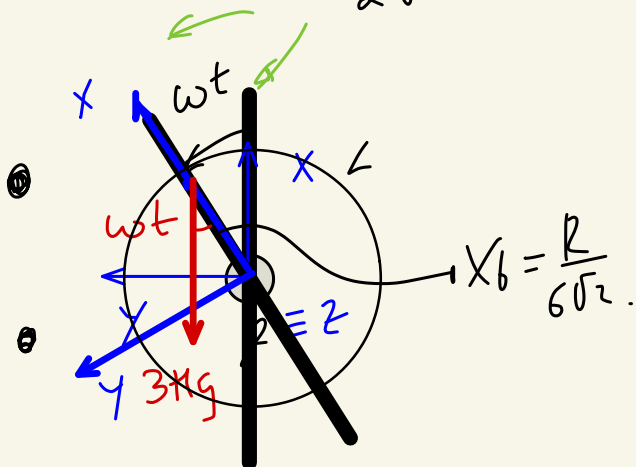
$$\vec{\pi}_0^{pov} = \frac{\pi g R}{2\sqrt{2}} \sin \omega t \hat{e}_1 + \frac{\pi g R}{2\sqrt{2}} \cos \omega t \hat{e}_2 + \frac{\pi g R}{2\sqrt{2}} \sin \omega t \hat{e}_3$$

$$A_y 2R - B_y 2R + \frac{\pi g R}{2\sqrt{2}} \sin \omega t = 0$$

$$B_x 2R - A_x 2R + \frac{\pi g R}{2\sqrt{2}} \cos \omega t = -\frac{1}{8} \pi R^2 \omega^2$$

$$P + \frac{\pi g R}{2\sqrt{2}} \sin \omega t = 0 \Rightarrow$$

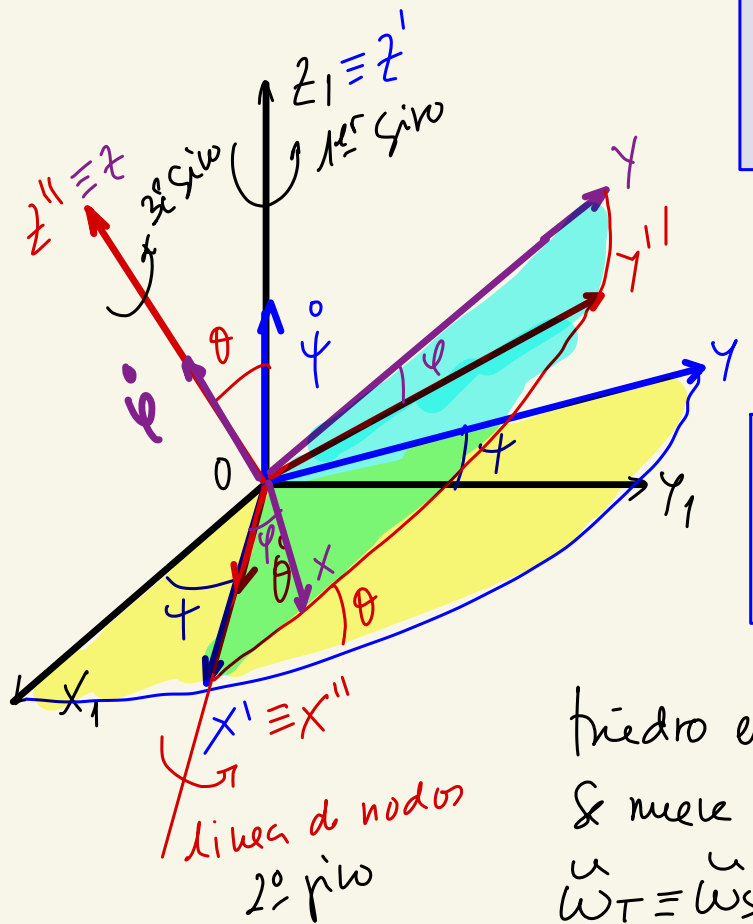
$$P = -\frac{\pi g R}{2\sqrt{2}} \sin \omega t$$



# TENA 8 SÓLIDO CON PUNTO FIJO

3 gdl ; Ángulos de Euler

• Ángulos de Euler



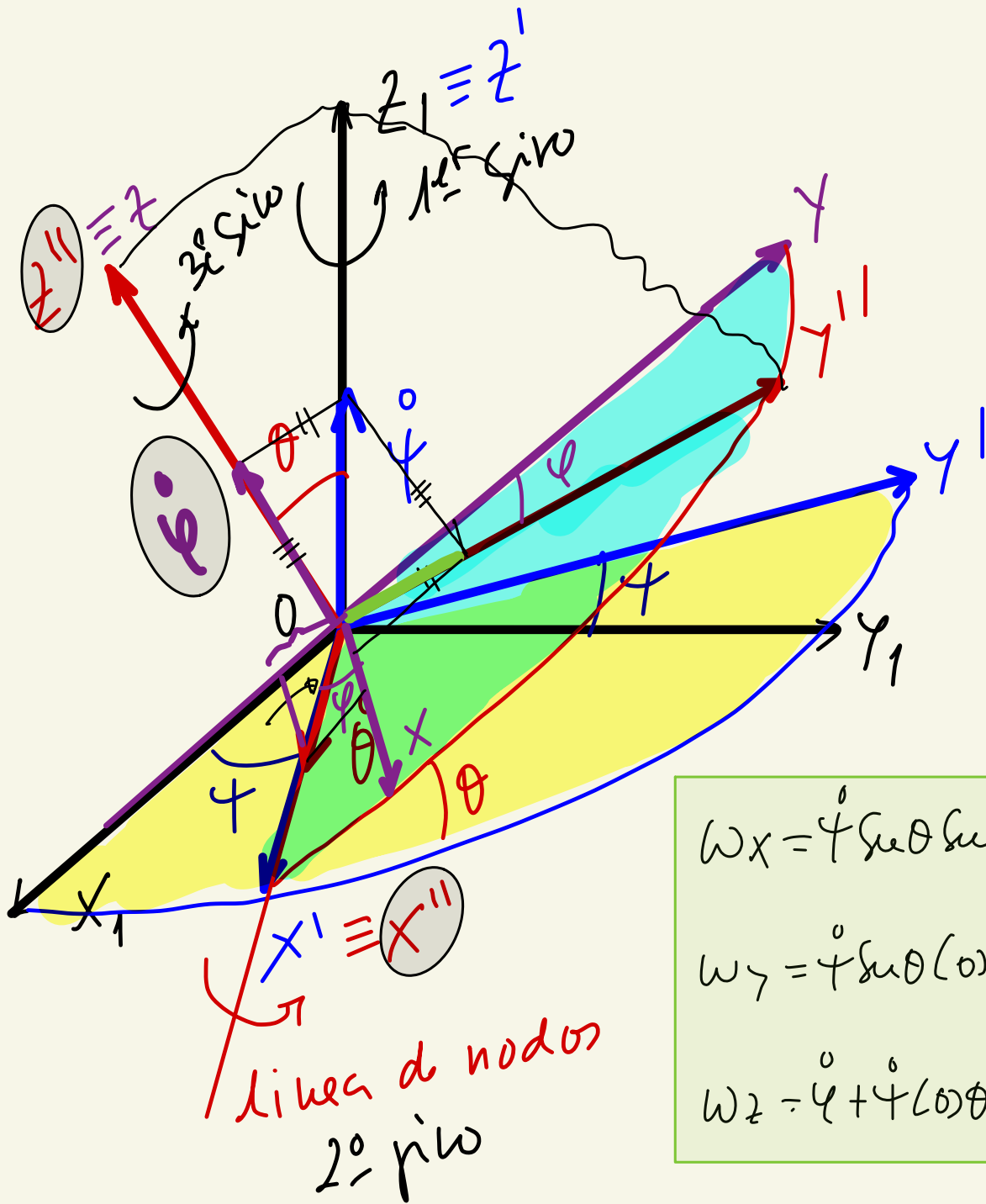
$\Psi \equiv$  ángulo de PRECESIÓN  
 $\dot{\Psi} \equiv$  vel ang " "

$\Theta \equiv$  ángulo de NUTACIÓN  
 $\dot{\Theta} \equiv$  vel. ang " "

$\Phi \equiv$  ángulo de ROTACIÓN PROPIA  
 $\dot{\Phi} \equiv$  vel ang. " " "

triedro es inercial  $\Rightarrow$   
 & mueve como el sólido  
 $\vec{\omega}_T \equiv \vec{\omega}_S$

$\vec{\omega}_S = \vec{\omega}_T = \dot{\Psi} + \dot{\Theta} + \dot{\Phi}$ ,  $\mu$  móvil  $\rightarrow$  las inercias son constantes  
 bajo el último de los sistemas.



$$\omega_x = \dot{\varphi} \sin \theta \sin \varphi + \dot{\theta} \cos \varphi$$

$$\omega_y = \dot{\varphi} \sin \theta \cos \varphi - \dot{\theta} \sin \varphi$$

$$\omega_z = \dot{\varphi} + \dot{\theta} \cos \theta$$

### ECUACIONES DEL MOVIMIENTO

$$\frac{d\vec{H}_0}{dt} = \vec{\tau}_0$$

ejes principales

$$\vec{H}_0 = I_x \omega_x \hat{e}_1 + I_y \omega_y \hat{e}_2 + I_z \omega_z \hat{e}_3$$

$$\frac{d\vec{H}}{dt}\Big|_F = I_x \dot{\omega}_x \hat{e}_1 + I_y \dot{\omega}_y \hat{e}_2 + I_z \dot{\omega}_z \hat{e}_3 + \begin{vmatrix} \hat{e}_1 & \hat{e}_2 & \hat{e}_3 \\ \omega_x & \omega_y & \omega_z \\ I_x \omega_x & I_y \omega_y & I_z \omega_z \end{vmatrix}$$

$$I_x \dot{\omega}_x + \omega_y \omega_z (I_z - I_y) = \tau_{0x}$$

$$I_y \dot{\omega}_y + \omega_x \omega_z (I_x - I_z) = \tau_{0y}$$

$$I_z \dot{\omega}_z + \omega_x \omega_y (I_z - I_x) = \tau_{0z}$$

Ecuaciones de  
Euler

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19 GIROSCOPO DE EULER-POISSON (Plumero por fuerza)

Todas las acciones aplicadas lo están en el pto fijo

$$\vec{\tau}_0 = 0 \Rightarrow \frac{d\vec{H}}{dt}\Big|_F = 0$$

$$\vec{H}_0 = \text{cte}$$